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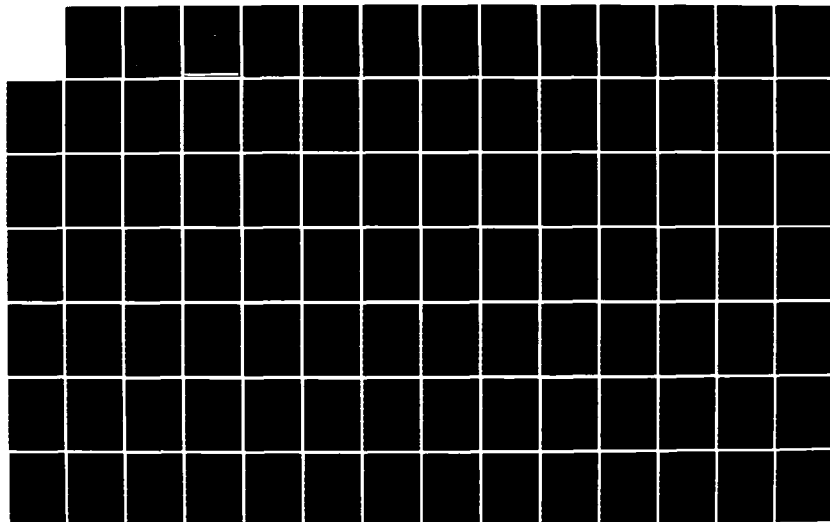
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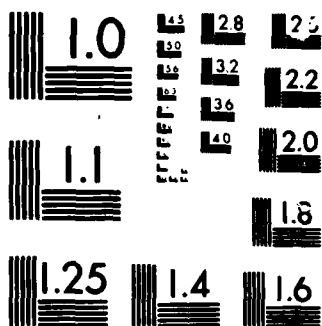
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**FRACTURE IN STABILIZED
SOILS**

VOLUME 2

FINAL TECHNICAL REPORT

DECEMBER 31, 1985

Prepared for
Texas A&M University
and the Air Force Office of Scientific Research

Prepared by
The Texas Transportation Institute



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**The Texas A&M University System
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Conventionally the thickness design of stabilized soil layers has been based upon the tensile strength of the stabilized soil layer and/or the appearance of the first crack. The design literature does not allow one to consider the true development of cracking in the stabilized soil layer. Knowledge of the mode of such cracking could drastically alter the philosophy behind thickness design of layers.

In this research the principles of theoretical fracture mechanics are used to explain the mode and mechanism of fracture in fine grained media stabilized with portland cement. Experimental fracture mechanics is used to validate or verify and in some cases to investigate more fully the hypothesized mechanisms of fracture. The influence of osmotic and matrix soil section, temperature, binder content, thermal and kinetic energy, from sources outside the crack, are considered in the study.

Linear elastic fracture mechanics is proven to be a highly acceptable analytical tool for these materials.

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CHAPTER I: INTRODUCTION

Purpose

The purpose of this research is to establish a preliminary model to investigate the crack propagation history in a given pavement system. This is accomplished through a multi-step process. First, a suitable program is used to calculate the stress distribution in the pavement layers under the load, in this study an elastic layer program. Second, through successive application of a finite element program, the stress intensity factor as a function of crack length is determined. Third, using the stress intensity factor distribution, the number of load cycles required to advance the crack a given increment is calculated. These increments carry the crack from its initial to its final value, which defines failure. The load cycles are calculated using the Paris equation. The remainder of the report is taken up by a detailed description of the problem and an in-depth account of the solution process and results.

Pavement System

The pavement system used in this study consists of a combination of three layers: a subgrade, a base, and a surface. Each layer has variable values for the modulus of elasticity and the thickness. This information is summarized in Table 1.

The subgrade thickness is listed as 50 inches. This is an arbitrary value selected so that the subgrade thickness does not affect the solution.

Table 1. Pavement Data Used in Developing Structural Data.

LAYER	MODULUS VALUES, PSI	THICKNESSES, IN.	POISSON'S RATIO
overlay	350000.0	0 3 6	0.35
base	400000.0 800000.0 1200000.0	8 12 16	0.35
subgrade	4000.0 9000.0	50	0.35

MODULUS CONDITIONS		
MODULUS CONDITION	BASE MODULUS, PSI	SUBGRADE MODULUS, PSI
1	1200000.0	4000.0
2	1200000.0	9000.0
3	800000.0	4000.0
4	800000.0	9000.0
5	400000.0	4000.0
6	400000.0	9000.0

In the analysis of the pavement system, all possible combinations of the values in Table 1 are used to develop the influence of pavement properties on fracture.

Loading

The loading shown in Figure 1 approximates that of an F-4 aircraft. It consists of a total load of 27,000 pounds. This load is distributed over a circle with a radius of 5.7 inches to provide a pressure of 265 psi.

Solution Philosophy

Strictly speaking, this problem is a three dimensional axi-symmetric one. However, because crack propagation itself is generally a plane strain phenomenon, it is considered preferable to rely on a simpler solution scheme. The approach here is to use a two-dimensional plane strain finite element program to obtain approximate stress intensity factors developing under the stresses calculated from an elastic layer analysis. for a single load, as assumed here, the results are satisfactory.

The cornerstone of the solution procedure is the computation of the stress intensity factor as a function of crack length. This is done with the finite element program described in a later chapter and with the aid of superposition techniques. When dealing with boundary values over an infinite region such as a pavement structure, the problem can be analyzed as the superposition of two problems (1) as illustrated in Figure 2. In the first phase, one applies the total loading to the pavement with no crack. For the second phase, the crack is included in the pavement. The stresses where the

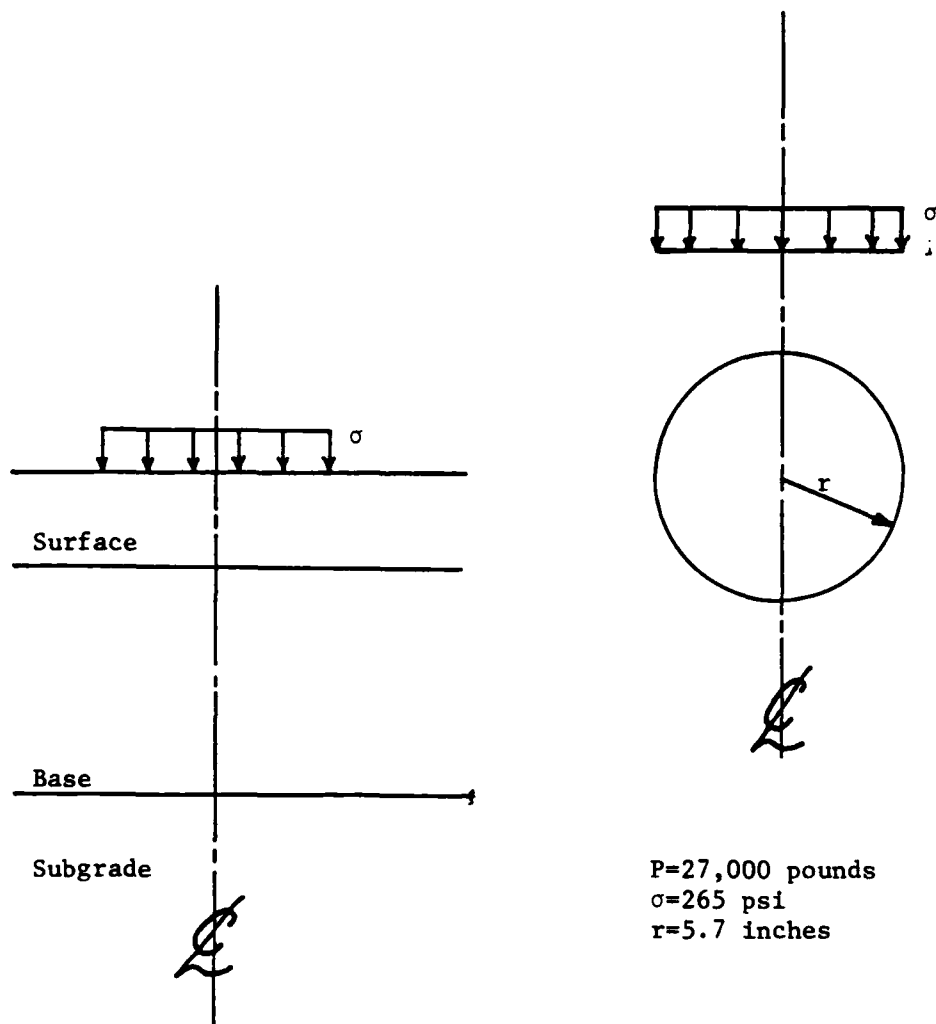


Figure 1. Loading Condition Modelled With Finite Element Fracture Program.

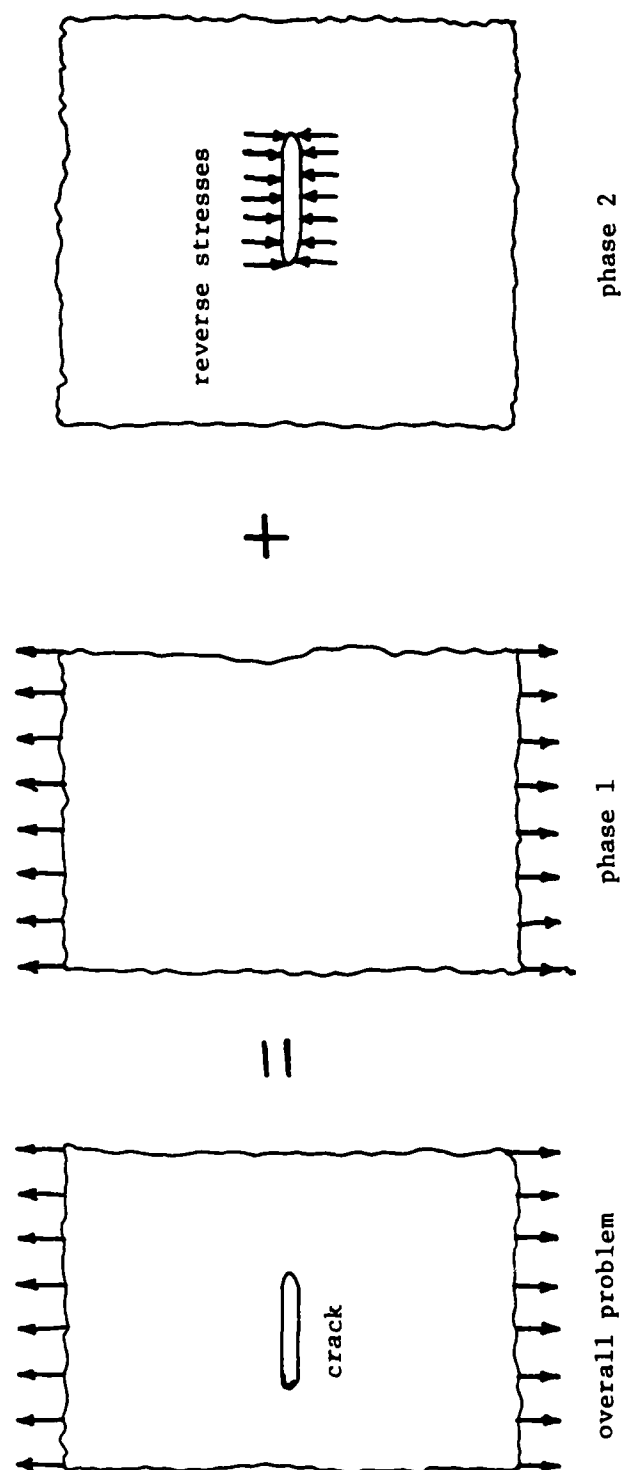


Figure 2. Schematic of Superposition Principle to Solve for Stress Intensity on Crack

crack would be in the first phase are applied in the reverse direction to the crack in the second phase. The stress intensity factor found from the second phase, although of opposite sign, is equal in magnitude to that of the overall problem. Thus the task of finding K_I becomes one of discovering the correct stress distribution due to loading at the crack, applying the reverse stresses to the crack, and analyzing the pavement.

A further difficulty arises when using the above superposition. As will be noted, stresses cannot be applied to the portion of the crack which lies within the crack tip finite element. Thus, a correction factor is required to account for this. The correction factor is given by (1) as:

$$C_k = \frac{\sqrt{2}}{\pi} \int_0^z \frac{\sigma_e(\xi) d\xi}{\sqrt{\xi}} \quad (1)$$

where ξ is the distance away from the crack tip and $\sigma_e(\xi)$ is the stress to be applied to the portion of the crack that lies within the crack element. If z is sufficiently small, $\sigma_e(\xi)$ can be assumed to be a constant, and equation (1) reduces to (2):

$$C_k = \left(\frac{8}{\pi}\right)^{1/2} \sigma_e \sqrt{z} \quad (2)$$

The final, correct stress intensity factor is found from C_K and K_I computed in superposition phase two by

$$K_I \text{ final} = K_I \text{ computed} + C_K \quad (3)$$

Crack propagation is calculated using this stress intensity factor in

numerical integration of the Paris equation. The Paris equation consists of the following:

$$dc/dN = AK_I^n \quad (4)$$

where

dc = differential crack extension

dN = differential increase in the number of load cycles

K_I = stress intensity factor

A, n = dimensionless material constants determined experimentally

Manipulation of the Paris equation gives:

$$dN = dc/AK_I^n \quad (5)$$

from which ΔN_f , the number of load cycles required to advance the crack an increment can be calculated as:

$$\Delta N_f = \int_{c_o}^{c_f} \frac{dc}{A K_I^n} \quad (6)$$

Writing the integral in terms of c/b leaves:

$$\Delta N_f = b \int_{(c/b)_o}^{(c/b)_f} \frac{d(c/b)}{A K_I^n} \quad (7)$$

When the current crack length equals the crack length at failure, the final value of N_f is reached.

The total number of load cycles required to advance the crack from the initial crack length to the current crack length, N_{fn} , is found by summing the ΔN_f values for the n crack increments which total the current crack length. Thus,

$$N_{fn} = \sum_{1}^n \Delta N_{fn} = N_{f(n-1)} + \Delta N_{fn} \quad (8)$$

Note that K_I is considered a function of (c/b) now, not crack length. This allows regression equations to be developed for the range of materials and loading conditions which lessens the reliance on the finite element computer code.

In this analysis the crack is assumed to form directly beneath the center line of the load, on the line of symmetry. It is assumed to originate at the interface of the base and the subgrade and to propagate upward through the base. The greatest crack length at which the pavement is assumed to have failed is the full thickness of the base. If c = crack length in the base and b = thickness of the base, then $c/b = 1.0$ is the maximum value at which failure can occur. This would lead one to believe that the life of the pavement ranges between c/b values of 0.0 and 1.0. However, this is true only if two conditions exist. First, K_I , the stress intensity factor, must be positive for all c/b . Second, the critical value of the stress intensity factor, K_{IC} , must not be exceeded. In the first case, if K_I is negative, then the crack cannot propagate in that region and that region cannot figure in the fracture life of the pavement. The areas defining the fracture life of the pavement when K is negative are illustrated in Figure 3, where typical K_I distributions for this problem are depicted. Case I and II are self-explanatory. In case III, it is assumed that some perturbation advances

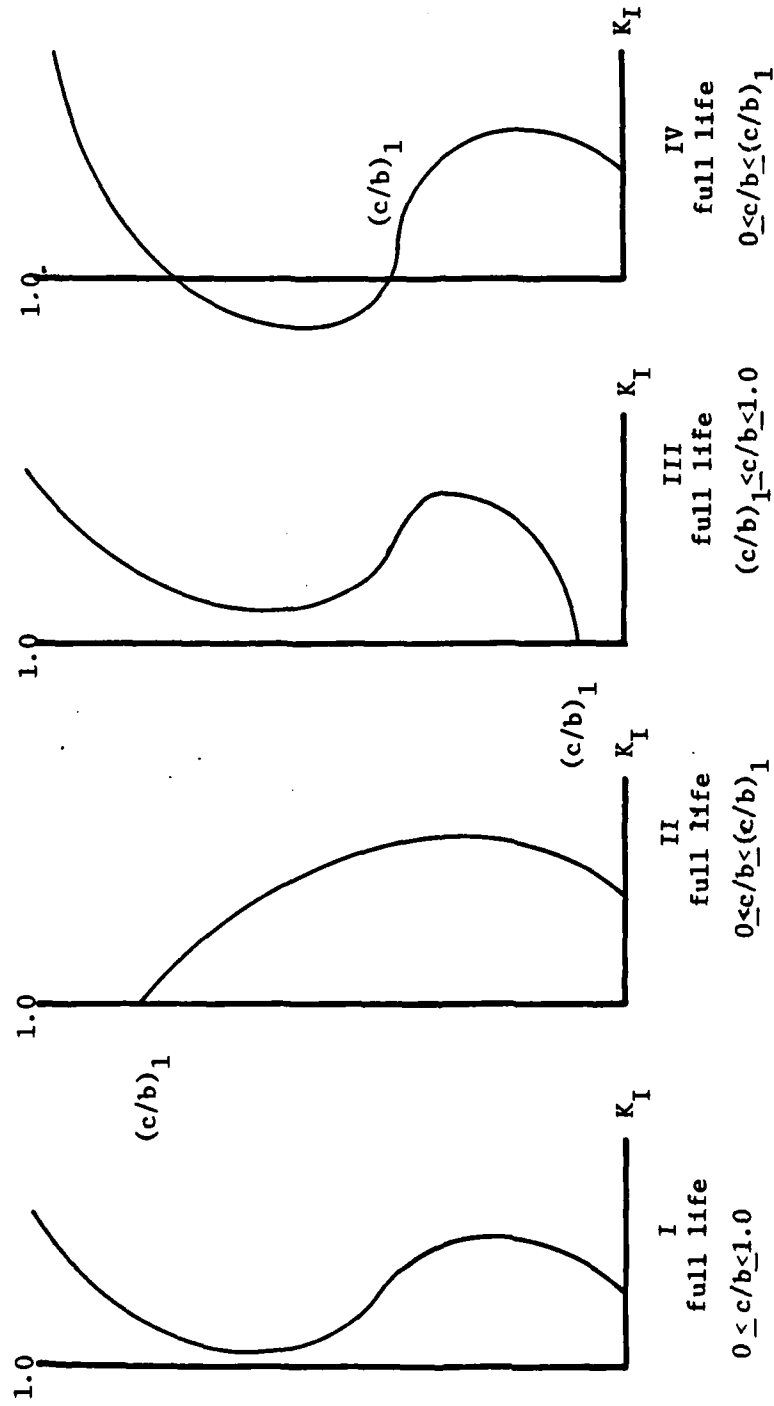


Figure 3. Schematics of Possible Stress Intensity Factor Distributions

the crack to $(c/b)_1$, so that the crack can propagate. Here $(c/b)_1$ is considered small so that this is readily possible and handled with the concept of starter flaws. If $(c/b)_1$ is not small, then the crack may never propagate. In case IV, the crack propagates to $(c/b)_1$ in the normal fashion and once there encounters negative K_I values and stops. It is unlikely that the crack will somehow be advanced suddenly through the relatively extensive negative K_I region, and if it is, the pavement will be nearly destroyed and can be said to have failed. Thus $(c/b)_1$ is the upper limit of the fatigue life in this case. One should note that if $(c/b)_1$ is significantly less than 1.0, then exhausting the fatigue life may not constitute failure of the pavement. The pavement may still maintain its structural integrity.

If K_{IC} is exceeded, then the crack instantly propagates to a point where $K_I = K_{IC}$. The fatigue life would then range between c/b values where K_I is less than or equal to K_{IC} . If K_I never falls below K_{IC} again, then the crack instantly propagates to failure. Unfortunately, K_{IC} values for the pavements in this problem are unknown, so there is no way to incorporate them into the solution algorithm. Therefore, for purposes of analysis, it is assumed that K_{IC} is greater than K_I for all c/b .

Solution Sequence

the solution sequence briefly described here will be described in detail in the following chapters. The stress distribution calculations will not be described, as this program can be replaced with any program which calculates stresses and strains under a loading. Other elastic theory programs can be used which will allow multiple wheel loadings to be used to produce the

stress distribution which is then used in the finite element fracture program. The Hybrid crack tip element of Pian and Tong used in a plane strain finite element program is used to determine the stress intensity factors resulting when a cracked base course is subjected to the calculated stress distribution. Regression equations have been developed in this study to reduce the use of the finite element program for simple pavement structures. Finally, the Paris equation is used in a program to calculate the number of loads to failure under the given stress intensity distribution.

CHAPTER II: THE FINITE ELEMENT PROGRAM

The Singularity Problem

For linearly elastic plane strain and plane stress problems, in the vicinity of a crack tip the stress varies as $1/\sqrt{r}$, where r is the radial coordinate of any point in the plane measured from the crack tip. Thus, at the crack tip where $r = 0$ the stress is singular. In the equations for stress, the coefficient of the singular $1/\sqrt{r}$ term is called the stress intensity factor and is representative of the "strength" of the singularity. It has units of (force/area)* $\sqrt{\text{length}}$. There are two main types of crack which govern the behavior of linearly elastic plane stress/strain problems. These are:

- 1) mode I or opening mode
- 2) mode II or in-plane shearing

These modes are illustrated in Figure 4.

For isotropic materials, the singular terms of the stress distribution in the vicinity of a crack tip are (3):

mode I:

$$\{\sigma\} = \begin{Bmatrix} \sigma_x \\ \sigma_y \\ \sigma_{xy} \end{Bmatrix} = \left(K_I \sqrt{2\pi r} \right) \begin{Bmatrix} \cos(\theta/2)[1 - \sin(\theta/2)\sin(3\theta/2)] \\ \cos(\theta/2)[1 + \sin(\theta/2)\sin(3\theta/2)] \\ \sin(\theta/2)\cos(\theta/2)\cos(3\theta/2) \end{Bmatrix} \quad (8)$$

mode II:

$$\{\sigma\} = \begin{Bmatrix} \sigma_x \\ \sigma_y \\ \sigma_{xy} \end{Bmatrix} = \left(K_{II} \sqrt{2\pi r} \right) \begin{Bmatrix} -\sin(\theta/2)[2 + \cos(\theta/2)\cos(3\theta/2)] \\ \sin(\theta/2)\cos(\theta/2)\cos(3\theta/2) \\ \cos(\theta/2)[1 - \sin(\theta/2)\sin(3\theta/2)] \end{Bmatrix} \quad (9)$$

In many formulations, the coefficient involving K in front of the vectors containing trigonometric terms in equations (8) and (9) is given as $(K/\sqrt{2r})$. The finite element program used in this research uses equations (8) and (9) as is. With the use of these equations it is necessary to multiply the results of the finite element program by $\sqrt{\pi}$ in order to match any results obtained from stress distributions based on the alternative coefficient. It is a minor point but mix-up in definitions can lead to confusion and incomprehensible results.

The stress intensity factors K_I and K_{II} are directly proportional to the magnitude of applied loading and are dependent on the geometry of the structure, the size and shape of the crack, and the nature of the applied loading.

Displacement fields in the vicinity of a crack tip are also functions of the stress intensity factors and can be written as :

mode I:

$$\begin{bmatrix} u \\ v \end{bmatrix} = (K_I \sqrt{\frac{2r}{\pi}}) / 8G \begin{bmatrix} (2K - 1)\cos(\theta/2) - \cos(3\theta/2) \\ (2K + 1)\sin(\theta/2) - \sin(3\theta/2) \end{bmatrix} \quad (10)$$

mode II:

$$\begin{bmatrix} u \\ v \end{bmatrix} = (K_{II} \sqrt{\frac{2r}{\pi}}) / 8G \begin{bmatrix} (2K + 3)\sin(\theta/2) + \sin(3\theta/2) \\ -(2k - 3)\cos(\theta/2) - \cos(3\theta/2) \end{bmatrix} \quad (11)$$

In equations (10) and (11), u and v are the displacements along the x and y axes in Figure I-1, G is the shear modulus, $k = (3 - 4\nu)$ for plane strain and $(3 + \nu)/(1 + \nu)$ for plane stress, and ν is Poisson's ratio.

The stress intensity factor is also related to the strain energy release rate, that is the change in strain energy in the structure per unit distance of crack extension. For linearly elastic plane strain/stress problems the strain energy release rate is given by (3):

mode I:

$$\mathcal{L}_I = K_I^2(k+1)/8G \quad (12)$$

mode II:

$$\mathcal{L}_{II} = K_{II}^2(k+1)/8G \quad (13)$$

where k and G are as defined previously.

The computer program provides for the calculation of the stress intensity factor in two ways. The first and far more emphasized way is to utilize eqs. (8)-(11) and solve for the stress intensity factor directly. The second method is to run the program twice, the second time with a slightly longer crack, note the difference in strain energy, and apply equation (12) and (13). This option is there if the analyst wants to take advantage of it, but it is not the main thrust of the program and is limited in that if both mode I and mode II are present equations (12) and (13) are not applicable.

The justification for using elements with assumed stress and displacement distributions that have the $1/\sqrt{r}$ singularity built in is based on the convergence rate of problems with singularities. The convergence of such problems has been shown to be of order h , where h is the maximum size of the elements used in the solution. The convergence rate is independent of p , the order of the complete polynomial used in the interpolation functions for stress and displacement. The quantities for which the convergence rate was established are the strain energy U and the stress intensity factors K_I and K_{II} . Given the above, in order to achieve good results for U or K using elements without singularities built in, the number of elements needed

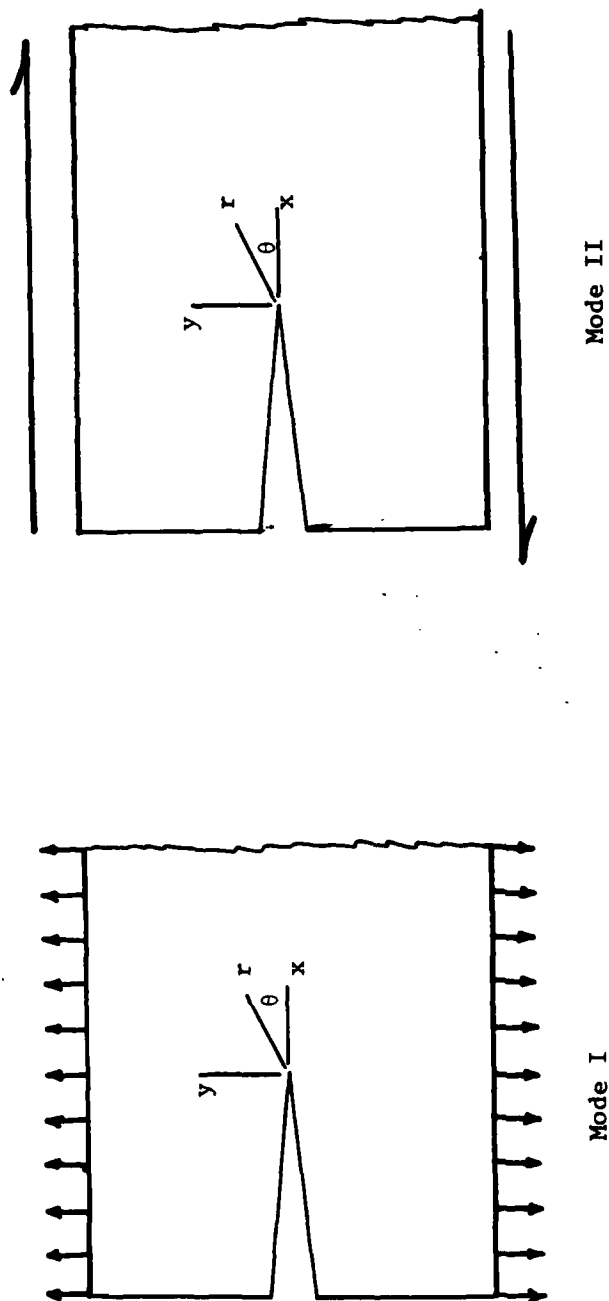


Figure 4. Two Major Modes of Fracture Analyzed in Finite Element Fracture Program.

becomes impractical because results for relatively large h are highly inaccurate. This is the value of solutions using elements with singularities in the region around the crack tip. Although they converge no faster, for relatively large element sizes they give acceptable results. Indeed, making the elements that include singularities too small can have an adverse effect on the results. This peculiarity occurs for this reason. If the elements containing singularities are small enough, the region in the structure significantly affected by the crack will extend beyond the scope of the singular elements and cause errors in adjacent non-singular elements. In the limit of mesh refinement, as all element sizes go to zero, the beneficial effects of elements with singularities disappear, leaving only a solution based on conventional elements.

The Crack Element

The crack element first developed by Pian and Tong (3), and which is shown in Figure 5 is the element used in the finite element program. It must be of sufficient size to insure good results, as mentioned above. The crack element is developed using a functional and methods which are a combination of both assumed stress and displacement models. As such, in this development, internal stresses and displacements as well as boundary displacements and tractions can be assumed independently. The necessary equilibrium and compatibility requirements are incorporated into the Euler equations, which result from the stationary condition of the functional. The Euler equations can be satisfied either exactly or in an average sense, the latter condition being the source of approximation in the finite element method.

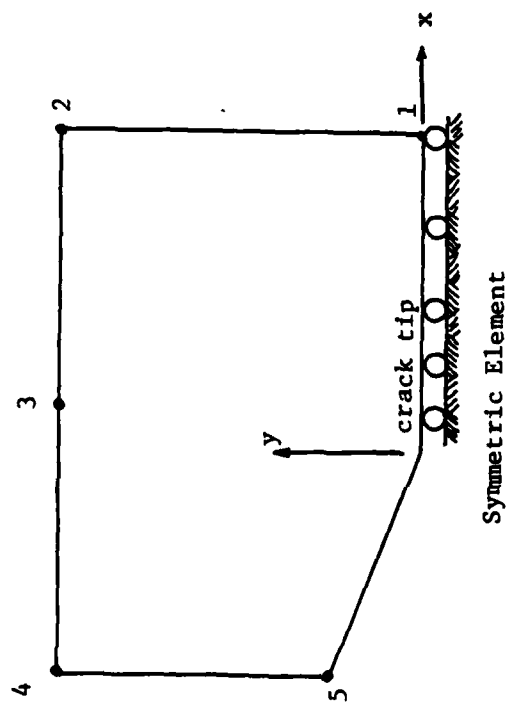
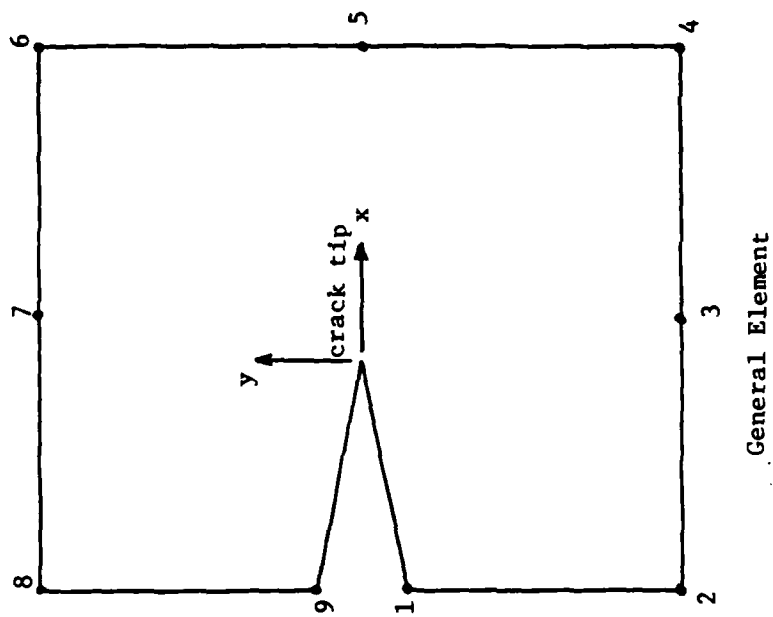


Figure 5. Crack Tip Elements

In the previous section a \sqrt{r} term is included in the equations for displacements. It is not desirable to introduce this term into the assumed boundary displacements, for two reasons. First, it can be incorporated into the assumed internal displacements. Second, one of the virtues of this crack element is that it can be used in conjunction with conventional finite elements, and if a \sqrt{r} term were included in its boundary displacement, boundary compatibility between it and a conventional element could not be insured. Of course, insuring this compatibility causes internal and boundary displacements for the crack element to be incompatible, a source of error.

The functional used in this development is, for plane problems,

$$\pi_m = \int_{\partial A_m} (\tilde{u}_i - u_i) T_i ds - \int_{S_m} u_i T_i ds + \frac{1}{2} \int_{A_m} [\sigma_{ij} (u_{i,j} + u_{j,i}) - C_{ijkl} \sigma_{ij} \sigma_{kl}] dA \quad (14)$$

where

A_m = area of the m th element

∂A_m = the boundary of A_m

S_m = the portion of ∂A_m over which tractions are prescribed

S_u = the portion of ∂A_m over which displacements are prescribed

C_{ijkl} = the elastic compliance tensor

u_i = a smooth internal displacement field

\tilde{u}_i = assumed boundary displacements

\bar{u}_i = prescribed boundary displacements

T_i = assumed boundary tractions

\bar{T}_i = prescribed boundary tractions

\tilde{u}_i is assumed such that \tilde{u}_i equals \bar{u}_i on S_u . Note that the thickness t is absent from equation (14) as unity is assumed, but can be multiplied in if the thickness is variable.

The Euler equations for the functional in equation (7) are

$$\frac{1}{2} (U_{i,j} + U_{j,i}) = C_{ijkl} \sigma_{kl} \quad \text{in } A_m \quad (15a)$$

$$\sigma_{ij,j} = 0 \quad \text{in } A_m \quad (15b)$$

$$J_i = \sigma_{ij} v_j \quad \text{on } A_m \quad (15c)$$

$$U_i = \tilde{U}_i \quad \text{on } A_m \quad (15d)$$

$$T_i = \bar{T}_i \quad \text{on } m \quad (15e)$$

Note that the body force is excluded for simplicity, and that v_j is a direction cosine.

Equation (15a) represents compliance with stress-strain laws by assumed stresses and displacements, equation (15b) represents the satisfaction of internal equilibrium, equation (15c) represents compatibility of assumed stresses and boundary tractions, equation (15d) represents compatibility of internal and boundary displacements, and equation (15e) represents satisfaction of boundary conditions by boundary tractions. The crack element provides for internal equilibrium and compatibility, interelement compatibility, and boundary conditions on displacements identically. The exact solution satisfies all the Euler equations. Also, if the solution is exact, then the stationary condition on equation (14) will provide for interelement equilibrium and compatibility. Otherwise, interelement equilibrium is satisfied only in a work equivalent sense, while interelement compatibility will be satisfied by assuming the same boundary displacement functions for all elements, crack or conventional.

The element stiffness matrix, whose development is discussed in the following pages, is derived using complex variable techniques.

To begin with, let $z = x + iy$, where x and y are as depicted in Figure 4. As shown previously, $(\sigma)^{-1}/\sqrt{r}$ & $(u/v)^{-1}/\sqrt{r}$, so that stresses and displacements can be expressed as (3):

$$\left. \begin{aligned} \sigma_y + \sigma_x &= 2[\phi'(z) + \overline{\phi'(z)}] \\ \sigma_y - \sigma_x + 2i\sigma_{xy} &= 2[\overline{Z}\phi''(Z) + \chi'(Z)] \end{aligned} \right\} \quad (16)$$

$$Z_u(U+iV) = \eta\phi(Z) - Z\overline{\phi'(Z)} - \overline{\psi(Z)} \quad (17)$$

where $\eta = E/2(1 + \nu)$, $\eta = 3-4\nu$ for plane strain and $(3 - \nu)/(1 + \nu)$ for plane stress. E and ν are Young's modulus and Poisson's ratio, respectively. It can be seen that $\mu = G$ and $\eta = k$ in Section I. $()'$ denotes differentiation and $()$ denotes the complex conjugate. ϕ and ψ are analytic functions. The above definitions of stress and displacement satisfy Euler equations (15a) and (15b). In addition, boundary tractions are chosen in this development so that they satisfy equation (15c).

In order to choose proper stresses and displacements for the crack element that account for singularities of all order, the following mapping function is introduced (4).

$$Z = W(\mathcal{L}) = \zeta^2 \quad (18)$$

or

$$\mathcal{L} = z^{1/2} \quad (19)$$

with $-\pi/2 \leq \arg \zeta \leq \pi/2$ and $-\pi \leq \arg z \leq \pi$. On the \mathcal{L} plane, the element lies on the region where the real part of \mathcal{L} is positive and the crack lies on the imaginary axis. ϕ and ψ are analytic functions of \mathcal{L} , enabling simple polynomials in terms of \mathcal{L} to be used in the finite element solution.

Using the mapping given above, equations (16) and (17) become

$$\begin{aligned}\sigma_x + \sigma_y &= 4R_e [\phi'(\mathcal{L})/w'(\mathcal{L})] \\ \sigma_x + \sigma_y - 2i\sigma_{xy} &= 2 \overline{w(\mathcal{L})} [\phi'(\mathcal{L})/w'(\mathcal{L})]' + \psi'(\mathcal{L})/w'(\mathcal{L}) \\ 2u(+iv) &= \eta\phi(\mathcal{L}) - w(\mathcal{L})\sigma'(\mathcal{L})/w'(\mathcal{L}) - \psi(\mathcal{L})\end{aligned}\quad (20)$$

In order to satisfy the stress free condition on the crack tip given by (3).

$$\sigma(\mathcal{L}) + w(\mathcal{L})\overline{\phi'(\mathcal{L})/w'(\mathcal{L})} + \overline{\psi(\mathcal{L})} = 0 \quad (21)$$

the following form of ψ is chosen

$$\psi(\mathcal{L}) - \overline{\phi(-\mathcal{L})} - \overline{w(-\mathcal{L})}\phi'(\mathcal{L})/w'(\mathcal{L}) \quad (22)$$

By using equations (15c), (19), (20), and (22) all of the Euler equations except for equation (15d) are satisfied by this crack element model. Substituting equations (21) into (14) gives in matrix form (4):

$$\pi_m = \int_{\partial A_m} (T)^T (\tilde{U}) ds - \int_{\partial A_M} (T)^T (U) ds \quad (23)$$

in which

$$\begin{aligned}(T) &= \begin{bmatrix} T_1 \\ T_2 \end{bmatrix} = \begin{bmatrix} \sigma_{xx}^v + \sigma_{xy}^v y \\ \sigma_{xy}^v + \sigma_{yy}^v y \end{bmatrix} = \text{boundary tractions} \\ (u) &= \begin{bmatrix} u \\ v \end{bmatrix} = \text{internal displacements} \\ (\tilde{u}) &= \begin{bmatrix} \tilde{u} \\ \tilde{v} \end{bmatrix} = \text{boundary displacements}\end{aligned}\quad (24)$$

For the derivation of the element stiffness matrix, the following forms of $\phi(L)$ and $\psi(L)$ may be assumed: (4)

$$\begin{aligned}\phi(\xi) &= \sum_{j=1}^N b_j \xi^j \\ \psi(\xi) &= - \sum_{j=1}^N [\bar{b}_j (-1)^{j+\frac{j}{2}} b_j] \xi^j\end{aligned}\quad (25)$$

where N is a finite integer and $b_j = \beta_j + i \beta_{N+j}$ with the β 's being real constants and $\beta_{N+2} = 0$. This is because $\phi = i^2$ and $\psi = 0$ give no contribution to the stresses in equations (20).

Using equations (20), (24), and (25) one can express the following: (3)

$$(T) = [R](\beta) \quad (26)$$

$$(u) = [U](\beta)$$

where (β) includes components $\beta_1, \beta_2, \dots, \beta_{2N}$ excluding β_{N+2} . The boundary displacements (\tilde{u}) are expressed as

$$(\tilde{u}) = [L](q) \quad (27)$$

(q) is the vector of nodal displacements and $[L]$ is an interpolation matrix defined on ∂A_m . $[L]$ is such that boundary displacements for the crack element and adjacent conventional elements are the same. For instance, if, as in the finite element program, linear boundary displacements are employed, then between any two nodes on the crack element $[L]$ will be such that (\tilde{u}) has a linear variation.

The number of nodes in the crack element used in the finite element program varies from 5 to 9 depending on symmetry, as shown in Figure 5. Boundary displacements are not assumed along the crack edge of the element because tractions are zero and there is no adjacent element. Because of this, in the program, there is no way to load the portion of the crack within the element and there is no way to control the crack's displacement within the element.

Substitution of equations (26) and (27) into equation (23) gives

$$\pi_m = (\beta)^T [G](q) - 1/2 (\beta)^T [H](\beta) \quad (28)$$

in which

$$[G] = \int_{\partial A_m} [R]^T [L] ds \quad (29)$$

$$[H] = 1/2 \int_{\partial A_m} ([R]^T [U] + [U]^T [R]) ds \quad (30)$$

As the β 's can be assumed independently from surrounding elements, the stationary condition of equation (28) with respect to (β) gives

$$[H](\beta) = [G](q) \quad (31)$$

or

$$(\beta) = [H]^{-1} [G](q) \quad (32)$$

With equation (31), one can substitute back into equation (28) and eliminate (β) . One does not have to. Instead, one could solve for (β) 's simultaneously with the (q) 's. However, in the finite element program the (β) 's are eliminated.

Substituting equation (32) into equation (28) gives (3)

$$\pi_m = 1/2 (q)^T [k] (q) \quad (33)$$

where the element stiffness matrix for the crack element is

$$[k] = [G]^T [H]^{-1} [G] \quad (34)$$

After global assembly and solution for the global displacement vector, the stress intensity factors can be found from the now known (q) . The stress intensity factors can be shown to be related to (β) in the following manner: (4)

$$K_I = (\beta_I) \sqrt{2} \quad (35)$$

$$K_{II} = (\beta_{N+1}) \sqrt{2}$$

Because (β) is related to (q) by equation (32), one can write

$$K_I = (B_I)^T (q) \quad (36)$$

$$K_{II} = (B_{II})^T (q)$$

It appears that $(B_I)^T$ is the first row of $[H]^{-1}[G]$ multiplied by $\sqrt{2}$ and that $(B_{II})^T$ is the $(N+1)^{th}$ row of $[H]^{-1}[G]$ multiplied by $\sqrt{2}$ as is shown in the program.

The reference followed in the development of this chapter gives details as to how the integrations implied by equations (29) and (30) are to be accomplished in terms of complex variables, and it gives certain properties of $[H]$ (namely that its upper right and lower left quadrants are blocks of zeros) that make it more efficient to invert (4).

The finite element program uses the crack element of the previous section in conjunction with CST triangular elements and 4-CST quadrilateral elements obtained by condensing the middle node. In fact, the program is essentially the program in Desai and Able's finite element text (2) modified to incorporate the crack element.

A complete discussion of the entire program and its input requirements is given in Appendix B.

Operational Parameters

It is convenient to define certain parameters which are used to describe and discuss the crack element. The first parameter is ϵ .

$$\epsilon = (\text{Length of crack within crack element}) / (\text{Total Crack Length}) = a/c$$

The second parameter is the ratio a/l .

$$a/l = (\text{Length of crack within crack element}) / (\text{Length of Crack Element})$$

The last parameter is the ratio c/b .

$$c/b = (\text{Total crack length}) / (\text{Depth of Base})$$

The element shown in Figure 6 is used for all the finite element meshes in the solution procedure yet to be described. Because of the symmetry of the problem, the five node element is used here. It is desirable to use one element for all the meshes so that one can create and build meshes quickly and easily. This particular element is chosen for various reasons. First, it gives a reasonable value of a/l for all c/b ratios. A plot of a/l versus percent error in K_I is shown in Figure 7 for the crack element applied to the Bowie crack problem (5). The above element has a constant a/l ratio of 0.5. It is seen that this a/l ratio yields about a 3 percent error, which is relatively small.

A second reason that the element of Figure 6 is used is that as the crack propagates, ϵ remains within reasonable limits for accuracy. This can be seen from Figure 8, which is a plot of ϵ versus K_I for the Bowie crack problem (5). The element used in this research is the 2 element. From this plot, a value of $\epsilon > 0.2$ is needed to insure reasonable accuracy in K_I . In the solution process for the problem of this report, a maximum value of c/b of $1/2$ is used, and a minimum c/b value of $1/12$ is used. Also, the greatest b value is 16 inches, and the least is 8 inches. Thus, c_{max} is 8 inches, and c_{min} is $2/3$ inches. ϵ is therefore such that $0.03125 < \epsilon < 0.375$, and within the range of acceptable accuracy shown in Figure 8.

The above two reasons for using the element shown in Figure 6 are important, but many elements which satisfy the a/l and ϵ criteria could be created. The main reason to use the Figure 6 element is that it is small enough so that reverse stresses can be applied to enough of the crack length to make the solution process viable. For instance, c_{min} is $2/3$ inches.

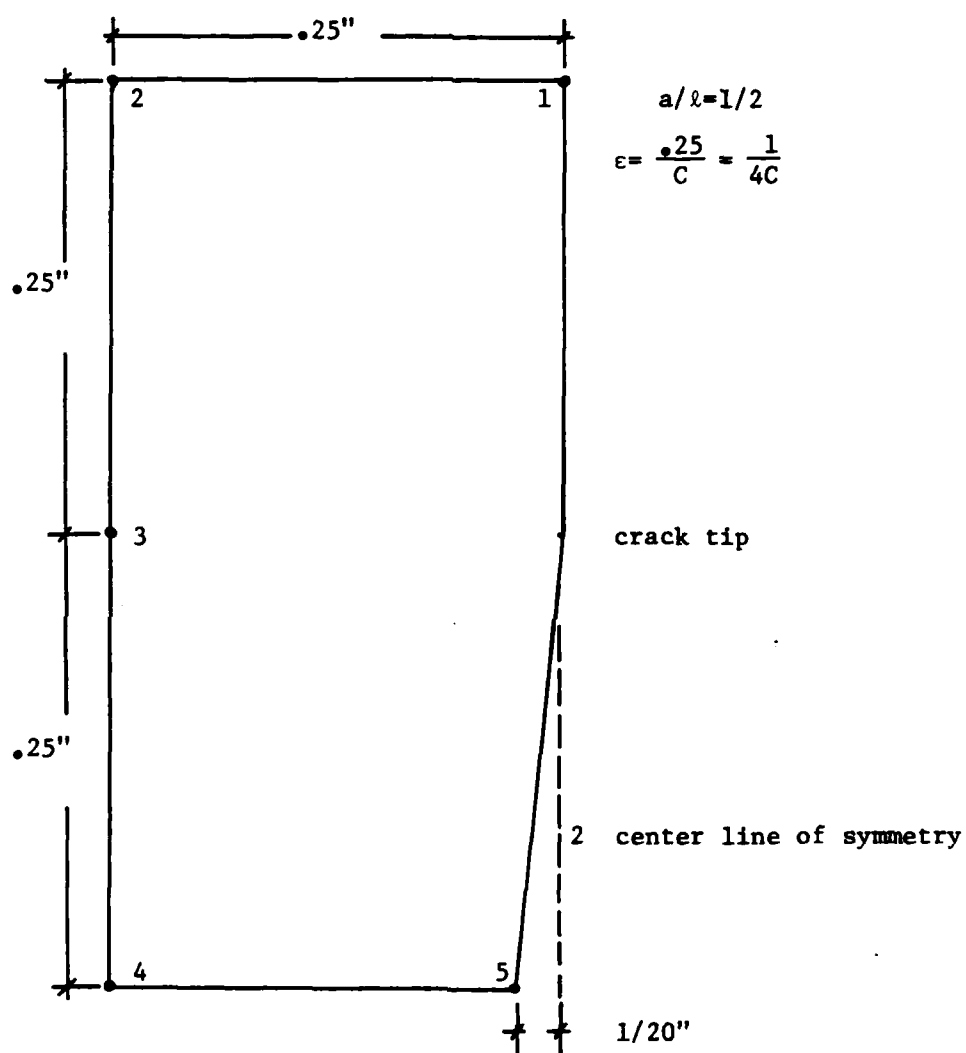


Figure 6. Crack Tip Element Used in Analyses

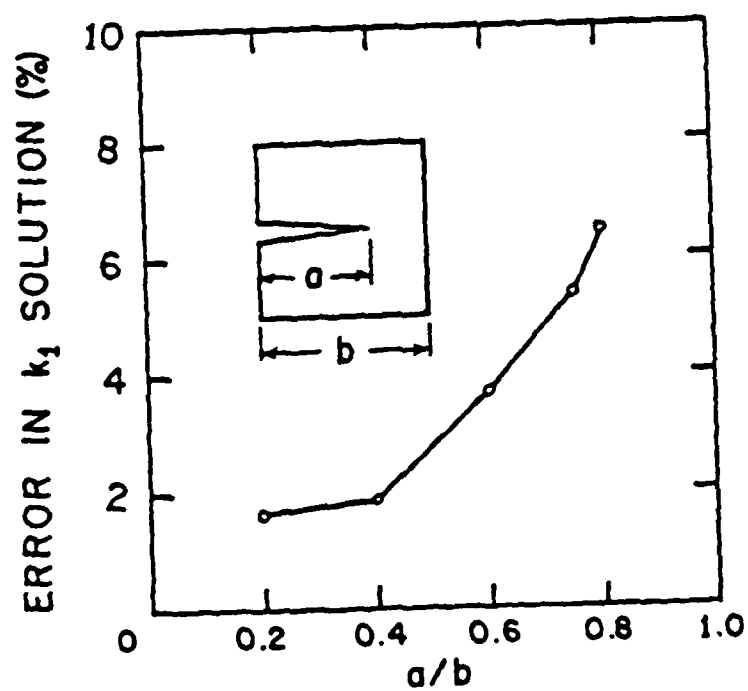


Figure 7. Illustration of Errors Produced By Variation in Crack Tip Length Within The Element (4)

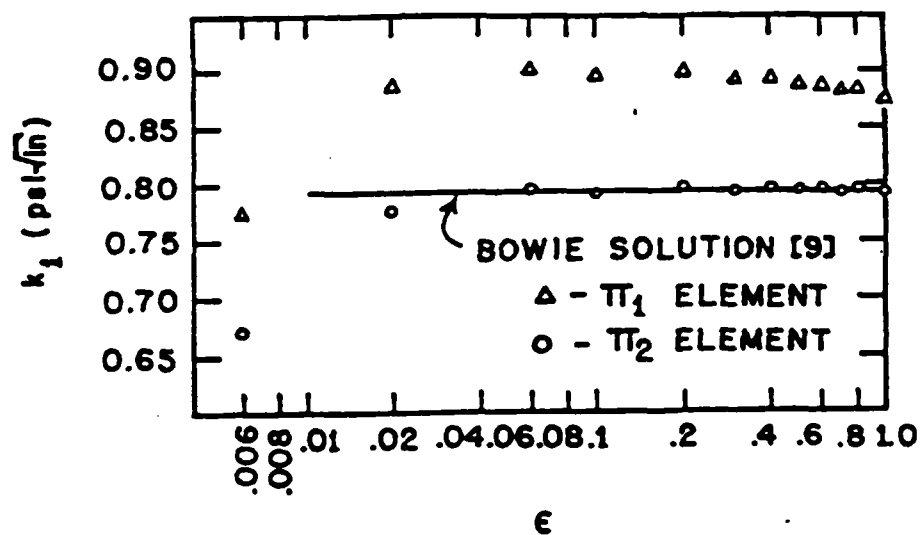


Figure 8. Illustration of Accuracy Related to Crack Tip Length (4)

With $a = 1/4$ inches, as in Figure 6, one can apply tractions to $5/12$ inches, or 62.5%, of the crack length. Remember, one cannot apply tractions to the portion of the crack within the crack element. The 62.5% value can be considered to be a significant amount and meaningful results can be obtained from it. However, as a increases, this percent value decreases and quickly becomes insignificant. If $a = c$, or $\epsilon = 1$, then there is no crack length available to apply reverse stresses to, and the solution philosophy outlined earlier is impossible.

There will be five values of c/b used in the finite element analysis of each combination of base, overlay, and modulus. These are shown in Table 2. These values were chosen for accuracy, uniformity, and from an initial study that $c/b < 1/2$ was the most critical range. Concerning accuracy, it was seen above that $c/b = 1/2$ produced a maximum c for the 16 inch base of 8 inches and a minimum ϵ of 0.03125, which is very near the limit for accuracy of 0.02. Actually, the absolute greatest value of c possible is 16 inches in the 16 inch base, and for the element of Figure 6 this corresponds to an ϵ of 0.015625, which is quite close to 0.02. Therefore, accuracy is not a compelling reason to keep c/b less than $1/2$, although for c/b greater than $1/2$ one approaches the ragged edge of accuracy.

The uniformity criteria is mainly for convenience. For purposes of comparing trends in K_I distributions from one base-overlay combination to another, it is very helpful to have results computed for the same or similar c/b values. The differences seen in Table 2 for the 12 inch base are because of ease in mesh creation.

The main reason why $c/b < 1/2$ values are used is that it was thought that they were the most critical values, especially those $< 1/6$. This thought

Table 2. C/B Values Used in Analysis

BASE, IN	C/B VALUES
8, 16	$1/12, 1/8, 1/6, 7/16, 1/2$
12	$1/12, 5/36, 1/6, 5/12, 1/2$

arose from intuition and previous experience, and is not necessarily borne out by the results in Chapter III.

Other details concerning the element in Figure 6 should be addressed. In Figure 6, the side opposite the crack is straight. For other problems, like the Bowie crack problem, better results are obtained if the side is humped, as in Figure 9. For this problem, however, results using the humped element were poor. Thus the straight side is strongly recommended for this problem. Also, note that in Figure 6 an offset appears at the bottom of the element. This is to simulate the physical opening of the crack. For convenience, this offset is present all the way through the subgrade.

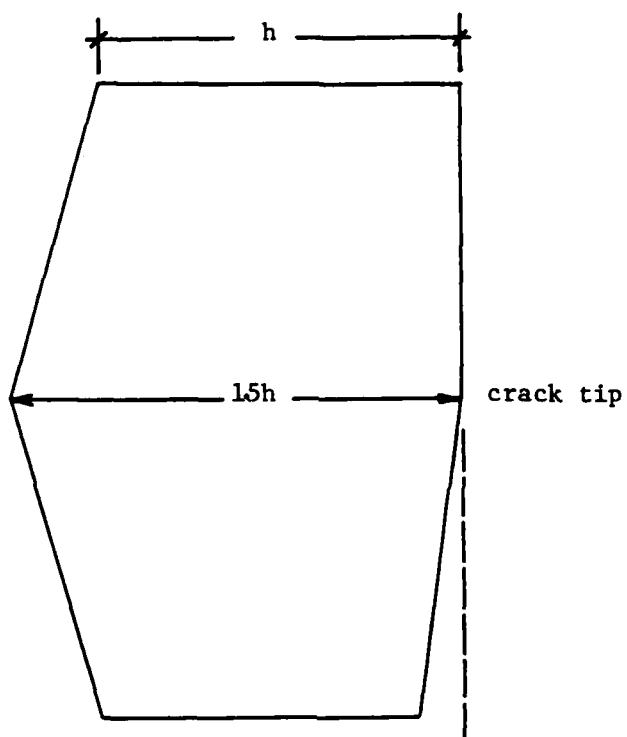


Figure 9. Example of a "Humped" Crack Tip Element

CHAPTER III: SOLUTION PROCEDURE

Development of Stress Distributions

The first step in the solution process is the calculation of the stress distributions to be used in superposition phases discussed in Chapter I. In this problem there is no crack, and thus the finite element program with no crack element can be used to calculate the stress distribution. A distribution must be found for each combination of base thickness, overlay thickness, and modulus to be analyzed. The following combinations are analyzed in this study: all modulus conditions for all base thicknesses with no surface, and all modulus conditions for a base thickness of 8 inches and surfacing of 3 and 6 inches. This information is summarized in Table 3. These specific combinations are analyzed so that all base thicknesses for a given surface (0 inches) can be compared, and all surfaces for a given base thickness (8 inches) can be compared. A sample mesh is shown in Figure 10 for reference. In this mesh, the base thickness is equal to 8 inches and the overlay thickness is 3 inches. The input corresponding to this mesh is listed in Appendix B, along with the corresponding output file. Note that the different modulus conditions can be achieved by simply changing the input values. Also note that the thickness of the meshes in this section is one inch, allowing the 270 psi load to be applied as a traction of the same magnitude.

The results are shown in Table 4. The numbers in the headings are element numbers, and correspond to the elements in the base, along the line of symmetry, where the cracks would run. Lower element numbers refer to elements at the bottom of the base, while higher numbers refer to elements at

Table 3. Pavement Parameters Used in the Factorial Analysis

<u>BASE, IN</u>	<u>SURFACE</u>	<u>MODULUS CONDITIONS</u>
8	0 3 6	1-6
<hr/>		
12	0	1-6
<hr/>		
16	0	1-6
<hr/>		

Table 4. Stress Distributions for Superposition Phase one, psi

8 inch base, no overlay							
MC	266	285	304	323			
1	1705.1	546.4	-584.5	-1772.1			
2	1229.7	383.75	-438.09	-1314.2			
3	1455.1	461.01	-507.06	-1530.6			
4	1032.3	315.74	-378.84	-1126.6			
5	1086.8	334.56	-395.06	-1178.2			
6	751.6	218.24	-297.81	-864.87			

12 inch base, no overlay							
MC	266	285	304	323	342	361	
1	1205.5	704.01	234.94	-230.70	-725.58	-1279.1	
2	942.39	545.64	178.77	-185.68	-578.79	-1028.9	
3	1078.3	627.40	207.79	-208.84	-654.43	-1157.7	
4	807.06	464.26	149.79	-162.89	-503.95	-900.41	
5	845.81	487.56	158.10	-169.37	-525.31	-936.99	
6	597.49	338.45	104.64	-128.65	-389.89	-704.47	

16 inch base, no overlay								
MC	266	285	304	323	342	361	380	399
1	825.67	572.11	340.62	120.46	-98.98	-331.43	-597.65	-915.71
2	699.45	482.34	285.93	99.95	-85.69	-284.17	-515.73	-798.05
3	786.64	531.53	315.90	111.20	-92.96	-310.04	-560.58	-862.48
4	620.79	426.45	251.89	87.61	-77.50	-254.89	-464.93	-725.03
5	644.37	443.19	262.09	91.00	-79.94	-263.64	-480.13	-746.89
6	478.20	325.34	190.36	63.95	-62.95	-202.42	-373.73	-593.71

TABLE 1-2(D): 8 inch base, 3 inch overlay

MC	266	285	304	323
1	1466.0	629.89	-172.65	-998.51
2	1075.8	457.60	-130.19	-741.01
3	1232.9	566.90	-66.59	-717.41
4	887.08	402.36	-53.30	-526.53
5	903.29	469.46	69.95	-345.38
6	632.18	321.91	37.94	-255.06

8 inch base, 6 inch overlay				
MC	266	285	304	323
1	1190.6	615.34	69.38	-382.55
2	911.27	467.40	50.40	-308.17
3	1003.4	561.06	146.45	-376.58
4	751.0	415.71	105.63	-296.51
5	745.18	468.70	216.88	-312.75
6	535.85	332.48	150.93	-236.52

the top of the base. The numbers below the heading MC refer to the modulus condition, all of which are depicted previously in Table 1. The numbers below the element headings are the stresses at the element centroids, in psi. A positive value denotes tension. The stresses for modulus condition one in each table are graphed in Figure 11.

Linearize Stress Distributions

As can be seen from Figure 11, the stress distributions are very nearly linear, especially in the range of Y coordinate values which encompass the crack lengths to be investigated, up to 1/2 the base thicknesses. As a result, for the purpose of applying the reverse stresses to the crack lengths, it is expedient to linearize the stress distributions by assuming a linear distribution between each element centroid. Note that element centroids are two inches apart in Figures 10 and 11. This is because each horizontal line of elements in the base is two inches thick, for all meshes. Figure 10 shows an 8 inch base. Additional elements for 12 and 16 inch bases shown in Table 4 reflect these supplemental lines of elements.

The linearized distributions are given in Table 5. Listed are the slopes, m , and the initial stress value for each interval between element centroids. Values are listed for each modulus condition. The intervals are delineated by the Y coordinate values of the appropriate element centroids. Only intervals up to 1/2 the base thickness are listed. Note that the interval between the lowest element centroids, that is between 51 and 53 inches, is extended to the bottom of the base, which is located at a Y value of 50 inches. The stress at the bottom of the base is unknown, and the same slope that acts between 51 and 53 inches is assumed to hold between 50 and 51. The stress at the bottom of the base, or the value for the initial

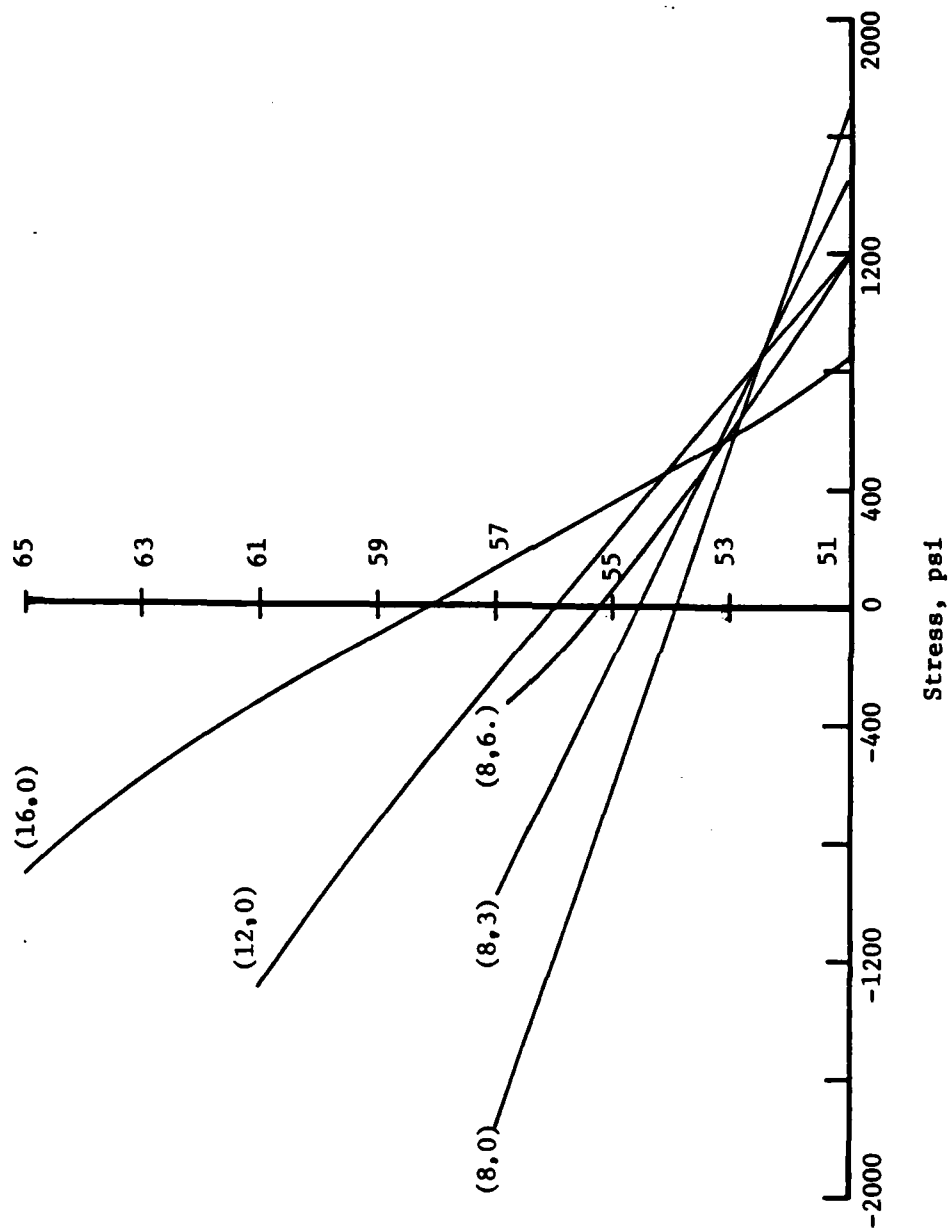


Figure 11. Stress Distributions for Modulus Condition 1, Parenthesis Contain Base Thickness and Surface Thickness Respectively

Table 5. Linearized Stress Distributions, psi

8 inch base, no overlay

MC	50-53 inches		53-55 inches	
	m		m	
1	579.35	2284.5	565.45	546.40
2	422.98	1672.7	410.92	383.75
3	497.05	1952.1	484.04	461.01
4	358.28	1390.6	347.29	315.74
5	376.12	1462.9	364.81	334.56
6	266.68	1018.3	258.03	218.24

12 inch base, no overlay

MC	50-53 inches		53-55 inches		55-57 inches	
	m		m		m	
1	250.75	1456.3	234.54	704.01	232.82	234.94
2	198.38	1140.8	183.44	545.64	182.23	178.77
3	225.45	1303.8	209.81	627.40	208.32	207.79
4	171.40	978.50	157.24	464.26	156.34	149.79
5	179.13	1024.9	164.23	487.56	163.74	158.10
6	129.52	727.01	116.91	338.45	116.65	104.64

16 inch base, no overlay

MC	50-53 inches		53-55 inches		55-57 inches		57-59 inches	
	m		m		m		m	
1	126.28	951.95	115.75	572.11	110.08	340.62	109.72	120.46
2	108.56	808.01	98.21	482.34	92.99	285.93	92.82	99.95
3	127.56	914.19	107.82	531.53	102.35	315.90	102.08	111.20
4	97.17	717.95	87.28	426.45	82.14	251.89	82.55	87.61
5	100.59	744.96	90.55	443.19	85.55	262.09	85.47	91.00
6	76.43	554.63	67.49	325.34	63.21	190.36	63.45	63.95

8 inch base, 3 inch overlay

MC	50-53 inches		53-55 inches	
	m		m	
1	418.06	1884.1	401.27	629.89
2	309.10	1384.9	293.90	457.60
3	333.0	1565.9	316.75	566.90
4	242.36	1129.4	227.83	402.36
5	216.92	1120.2	201.76	469.46
6	155.14	787.30	141.99	321.91

8 inch base, 6 inch overlay

MC	50-53 inches		53-55 inches	
	m		m	
1	287.63	1478.2	272.98	615.34
2	221.94	1133.2	208.50	467.40
3	221.17	1224.6	207.31	561.06
4	167.65	918.65	155.04	415.71
5	138.24	883.42	125.91	468.70
6	101.69	637.54	90.78	332.48

interval of 50-53 inches, is obtained by adding this slope to the stress at the first element centroid, located a unit distance away at 51 inches. Since there is a material (and thus stress) discontinuity between the base and the subgrade it is impossible to obtain the stress at the bottom of the base from the nearest subgrade element. It is thus necessary to assume something, and the above procedure is consistent with the concept of linearized stress distributions.

Mesh Considerations for Finite Element Runs

The sample mesh shown in Figure 12 is for a base of 8 inches, a surface of 3 inches, and a crack length of 3.5 inches (c/b of 0.4375). Also shown is an enlargement of a section of mesh in the base which is too fine to represent with the rest and an enlargement of the area immediately surrounding the crack element. This crack element and the area immediately surrounding it is common to all meshes. There is a mesh for each crack length in each base-overlay combination. The common crack element enables one to build one mesh from another quickly and easily and to be able to efficiently execute the analysis. As discussed in Chapter II, there are five crack lengths to be investigated for each base-overlay combination, and there are five base-overlay combinations targeted for analysis. Thus, there are 25 separate meshes to be considered and the efficiency afforded by a common crack element is essential.

The maximum aspect ratio in these meshes is 40:1. This occurs as far away from the crack element as possible. Near the crack, the aspect ratios are very nearly 1:1. Thus, near the crack where greater accuracy is required, the mesh is in its most accurate configuration, and far from the

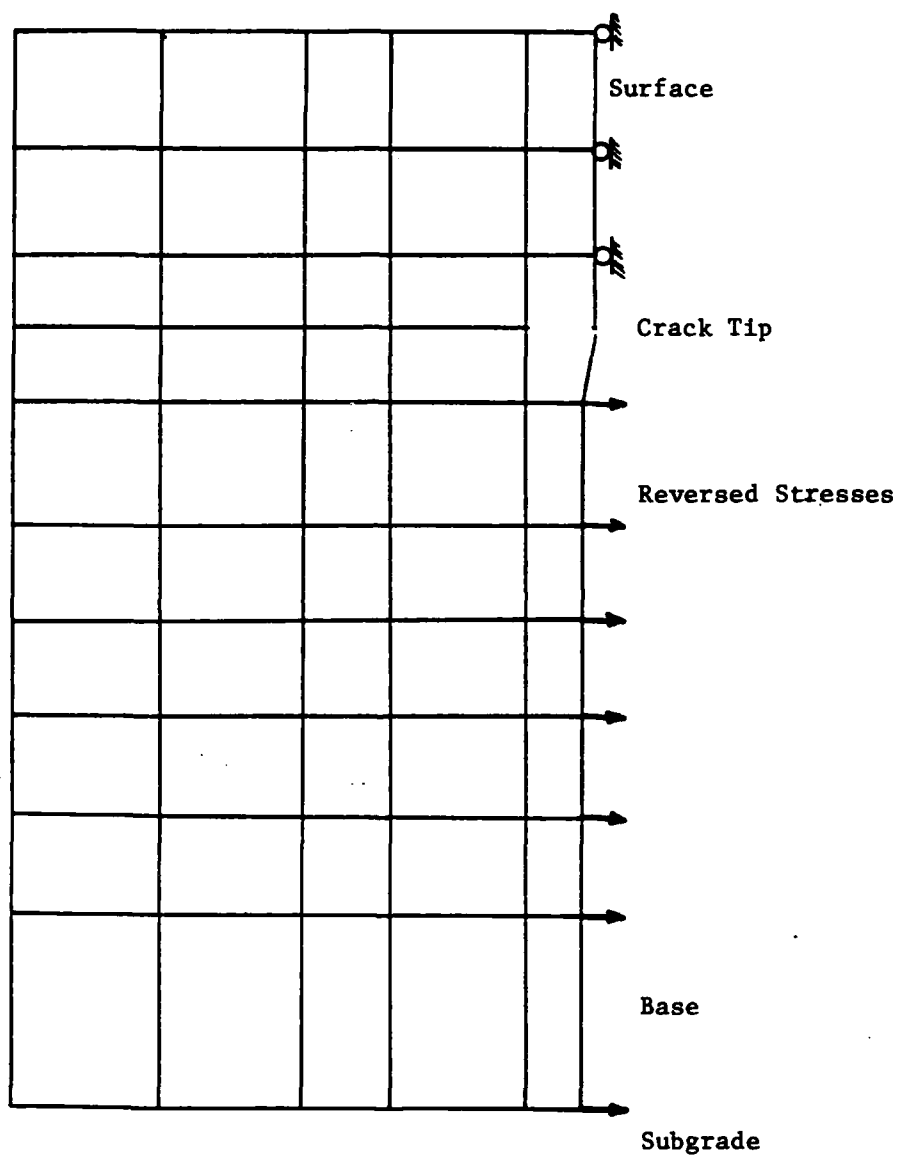


Figure 12. Illustration of Mesh Around Crack

crack where accuracy is not critical, the mesh is in its least accurate configuration.

The thickness of all the meshes of this section is one inch. Therefore, all stresses applied as loads transfer as tractions of the same magnitude.

The input file corresponding to Figure 11 is listed in Appendix B.

Applying Reverse Stresses to Crack Lengths

The linearized stresses shown above are applied to the meshes created above with a reverse sign. Note that there is one mesh for each base-surface combination and one linearized stress distribution for each modulus condition of each base-surface combination. Therefore, there are $25 \times 6 = 150$ different files to be run in the finite element program. Reverse stress values calculated for the mesh shown in Figures 11 with modulus condition one are given in Table 6. These values also appear in the input file for the mesh in Figure 11 that is listed in Appendix B. Note that the values in the input file have units of force/length, not stress, but have the same magnitude as the stresses in Table 6. This is because the thicknesses of all meshes is one inch. A positive sign indicates a stress in the positive X direction.

Uncorrected K_I Values

Each of the 150 runs took between 1.50 and 3.00 minutes on a Harris 800 super mini-computer. The results are listed in Table 7. The values in the tables are uncorrected stress intensity factors in $\text{psi}/\sqrt{\text{inch}}$. The negative signs indicate that the reverse stresses act to close the crack. In actuality, these stress intensity factors are positive, because the crack is

Table 6. An Example of a Reverse Stress Profile for Figure 3, MC=1.

<u>Y COORDINATE, IN</u>	<u>REVERSE STRESS, PSI</u>
50.0	1884.1
50.833	1535.7
51.5	1257.0
51.833	1117.6
52.167	978.27
52.5	838.92
52.833	699.57
53.25	525.38

Table 7. Uncorrected Stress Intensity Factors, psi/ $\sqrt{\text{in}}$

8 inch base, no surface C/B					
MC	1/12	1/8	1/6	7/16	1/2
1	-916.20	-1263.99	-1490.41	-2372.08	-2387.92
2	-653.28	-891.83	-1044.41	-1585.38	-1576.26
3	-778.18	-1068.65	-1256.70	-1959.78	-1962.35
4	-543.48	-735.67	-857.06	-1257.04	-1238.55
5	-573.83	-778.89	-908.91	-1347.53	-1331.49
6	-387.04	-512.59	-589.87	-801.68	-773.87

12 inch base, no surface C/B					
MC	1/12	5/36	1/6	5/12	1/2
1	-798.13	-1147.78	-1216.62	-1490.31	-1388.27
2	-613.84	-878.07	-927.64	-1105.32	-1017.78
3	-709.19	-1017.63	-1077.19	-1304.38	-1209.25
4	-518.45	-738.42	-778.06	-906.79	-827.16
5	-545.82	-778.48	-820.98	-963.63	-881.70
6	-369.66	-520.76	-545.04	-601.43	-535.64

16 inch base, no surface C/B					
MC	1/12	1/8	1/6	7/16	1/2
1	-603.00	-779.24	-913.42	-870.49	-783.82
2	-502.45	-647.50	-757.11	-704.63	-631.38
3	-572.40	-736.91	-860.32	-800.74	-725.99
4	-439.56	-565.24	-659.70	-602.85	-537.92
5	-458.43	-589.91	-688.91	-633.19	-535.64
6	-325.38	-416.05	-483.27	-420.69	-371.11

8 inch base, 3 inch surface C/B					
MC	1/12	1/8	1/6	7/16	1/2
1	-763.07	-1165.89	-2161.30	-1962.77	-2009.58
2	-553.75	-837.89	-1540.76	-1360.62	-1382.10
3	-630.24	-957.27	-1758.64	-1593.91	-1633.61
4	-446.75	-669.68	-1218.07	-1066.60	-1082.68
5	-443.83	-664.67	-1198.61	-1083.01	-1110.89
6	-303.38	-444.61	-791.12	-682.53	-691.41

8 inch base, 6 inch surface C/B					
MC	1/12	1/8	1/6	7/16	1/2
1	-596.74	-906.85	-1655.76	-1532.27	-1579.01
2	-452.21	-681.49	-1237.18	-1122.00	-1150.22
3	-490.88	-741.00	-1338.24	-1243.83	-1284.41
4	-362.56	-541.13	-970.67	-880.61	-903.77
5	-348.11	-517.56	-915.72	-855.82	-884.99
6	-245.00	-357.48	-627.08	-567.97	-582.97

opened by the loads on the pavement. However, as mentioned previously, the magnitudes of these numbers are correct.

A sample output file is given in Appendix B. This output file corresponds to the input file in Appendix B that was discussed previously.

Stress Intensity Correction Factors

The equation used to compute the correction factors is given by equation (2) of Chapter I. In this equation, σ_e is the stress in the vicinity of the crack tip that would be applied to the portion of the crack within the crack element, if that were allowed. Because the reverse stress distribution is assumed to be linear with depth, there is no one value of reverse stress in the vicinity of the crack tip. σ_e is then defined as the average value of the reverse stress that would be applied to the length of the crack within the crack element. The average reverse stress is the stress that would be applied halfway from the crack tip to the edge of the crack element. If Y is the vertical coordinate of the crack tip, then σ_e acts at $Y = Y - z/2$. The value of z , the length of crack within the crack element, is 1/4 inches.

A negative correction factor indicates that the reverse stresses to be applied act in a manner which closes the crack. They add with the negative stress intensity factors computed in the previous section to increase the magnitude of K_I . Positive correction factors add to decrease the magnitude of the stress intensity factor.

The computed correction factors are given in Table 8. The computed stress intensity factors are added to the correction factors in order to find the final values of K_I .

Table 8. Stress Intensity Correction Factors, psi/ in

8 inch base, no surface					
C/B					
MC	1/12	1/8	1/6	7/16	1/2
1	-1572.34	-1418.26	-1264.17	-262.62	-31.492
2	-1135.84	-1023.34	-910.85	-179.63	-10.889
3	-1342.77	-1210.58	-1078.38	-219.11	-20.821
4	-954.68	-859.39	-764.10	-144.72	-1.791
5	-1004.65	-904.65	-804.62	-154.40	-4.352
6	-697.21	-626.29	-555.36	-94.34	12.052

12 inch base, no surface					
C/B					
MC	1/12	5/36	1/6	5/12	1/2
1	-986.86	-853.48	-786.79	-186.60	13.470
2	-771.70	-666.18	-613.42	-138.58	19.698
3	-882.84	-762.92	-702.96	-163.31	16.571
4	-661.04	-569.86	-524.28	-114.01	22.752
5	-692.72	-597.44	-549.08	-121.04	21.881
6	-489.65	-420.75	-386.30	-76.28	27.064

16 inch base, no surface					
C/B					
MC	1/12	1/8	1/6	7/16	1/2
1	-637.80	-570.63	-503.46	-66.84	33.914
2	-540.04	-482.29	-424.59	-49.22	37.393
3	-606.45	-538.60	-470.75	-29.73	72.049
4	-479.16	-427.47	-375.79	-39.82	37.711
5	-497.41	-443.91	-390.40	-42.61	37.649
6	-368.85	-328.19	-287.53	-23.28	37.705

8 inch base, 3 inch surface					
C/B					
MC	1/12	1/8	1/6	7/16	1/2
1	-1322.58	-1211.39	-1100.21	-382.52	-222.43
2	-971.40	-889.19	-806.98	-277.18	-159.93
3	-1105.49	-1016.92	-928.36	-357.55	-231.19
4	-796.42	-731.96	-667.50	-252.87	-161.98
5	-800.05	-742.36	-684.66	-314.21	-233.72
6	-561.14	-519.88	-478.62	-214.36	-157.72

8 inch base, 6 inch surface					
C/B					
MC	1/12	1/8	1/6	7/16	1/2
1	-1055.15	-978.65	-902.15	-409.29	-300.39
2	-808.25	-749.22	-690.20	-310.55	-227.37
3	-881.48	-822.66	-763.83	-385.63	-302.93
4	-660.52	-615.93	-571.34	-285.30	-223.45
5	-645.12	-608.36	-571.59	-336.30	-286.07
6	-464.73	-437.69	-410.64	-238.12	-201.91

The final values of K_I are given in Table 9 and illustrated in Figure 13 through Figure 17. Note that the values of K_I are positive. The actual stress intensity factors are equal in magnitude but opposite in sign to the computed ones; that is, they indicate that the crack will open under load, not close.

From the tables and graphs it can be seen that the K_I values for the 8 inch base, no surface, are greater than those of the other pavements with no surfaces. The 16 inch base has the least K_I values of the above pavements. This is intuitively reasonable. Adding a 3 inch surface to the 8 inch base generally decreases the stress intensity factors, although the K_I values for $c/b = 1/6$ are greater. For the 6 inch surface, the K_I values are less than their counterparts in the no surface case for all the c/b values for which stress intensity factors are calculated.

Table 9. Corrected Stress Intensity Factors

8 inch base, no surface					
C/B					
MC	1/12	1/8	1/6	7/16	1/2
1	2488.5	2682.2	2754.6	2634.7	2419.4
2	1789.1	1915.2	1955.2	1765.0	1587.1
3	2120.9	2279.2	2335.1	2178.9	1983.2
4	1498.2	1595.1	1621.2	1401.8	1240.3
5	1587.5	1683.5	1713.5	1501.9	1335.8
6	1084.2	1138.9	1145.2	896.02	761.82

12 inch base, no surface					
C/B					
MC	1/12	5/36	1/6	5/12	1/2
1	1785.0	2001.3	2003.4	1676.9	1374.8
2	1385.5	1544.2	1541.1	1243.9	998.08
3	1592.0	1780.6	1780.2	1467.7	1192.7
4	1179.5	1308.3	1302.3	1020.8	804.41
5	1238.5	1375.9	1370.8	1084.7	859.81
6	859.31	941.51	931.34	677.71	508.57

16 inch base, no surface					
C/B					
MC	1/12	1/8	1/6	7/16	1/2
1	1240.8	1349.9	1416.9	937.34	749.90
2	1042.5	1129.8	1181.7	753.85	593.99
3	1178.8	1275.5	1331.1	830.47	653.94
4	918.72	992.71	1035.5	642.67	500.21
5	955.84	1033.8	1079.3	675.80	528.11
6	694.22	744.24	770.81	443.97	333.40

8 inch base, 3 inch surface					
C/B					
MC	1/12	1/8	1/6	7/16	1/2
1	2085.7	2377.3	3261.5	2354.3	2232.0
2	1525.2	1727.1	2347.7	1637.8	1542.0
3	1735.7	1974.2	2687.0	1951.5	1864.8
4	1243.2	1401.6	1885.6	1319.5	1244.7
5	1243.9	1407.0	1883.3	1397.2	1344.6
6	864.52	964.5	1269.7	896.89	849.13

8 inch base, 6 inch surface					
C/B					
MC	1/12	1/8	1/6	7/16	1/2
1	1651.9	1885.5	2557.9	1941.6	1879.4
2	1260.5	1430.7	1927.4	1432.5	1377.6
3	1372.4	1563.7	2102.1	1629.5	1587.3
4	1023.1	1157.1	1542.0	1165.9	1127.2
5	993.24	1125.9	1487.3	1192.1	1171.1
6	709.73	795.17	1037.7	806.09	784.90

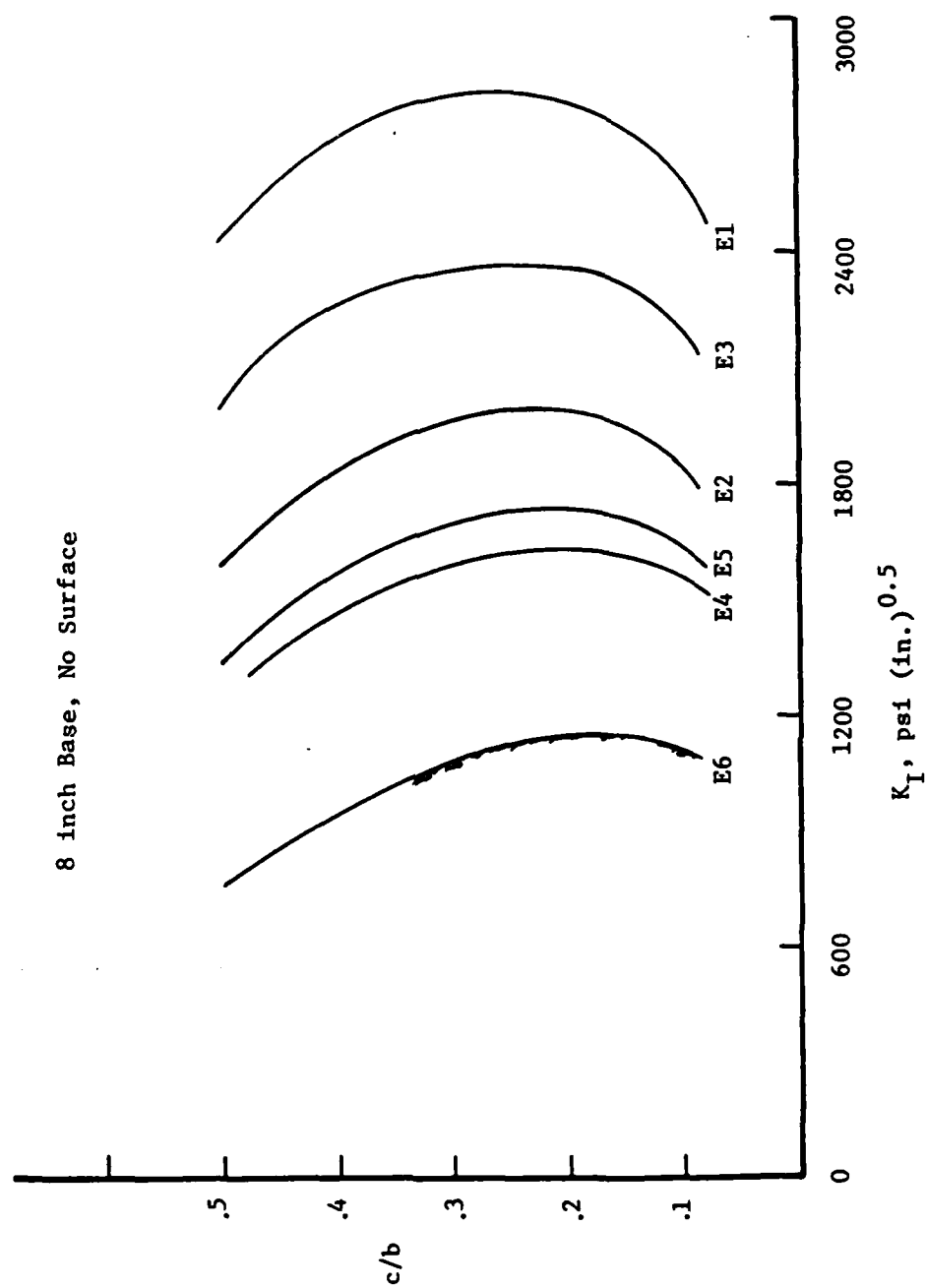


Figure 13. Distribution of Stress Intensity Factor

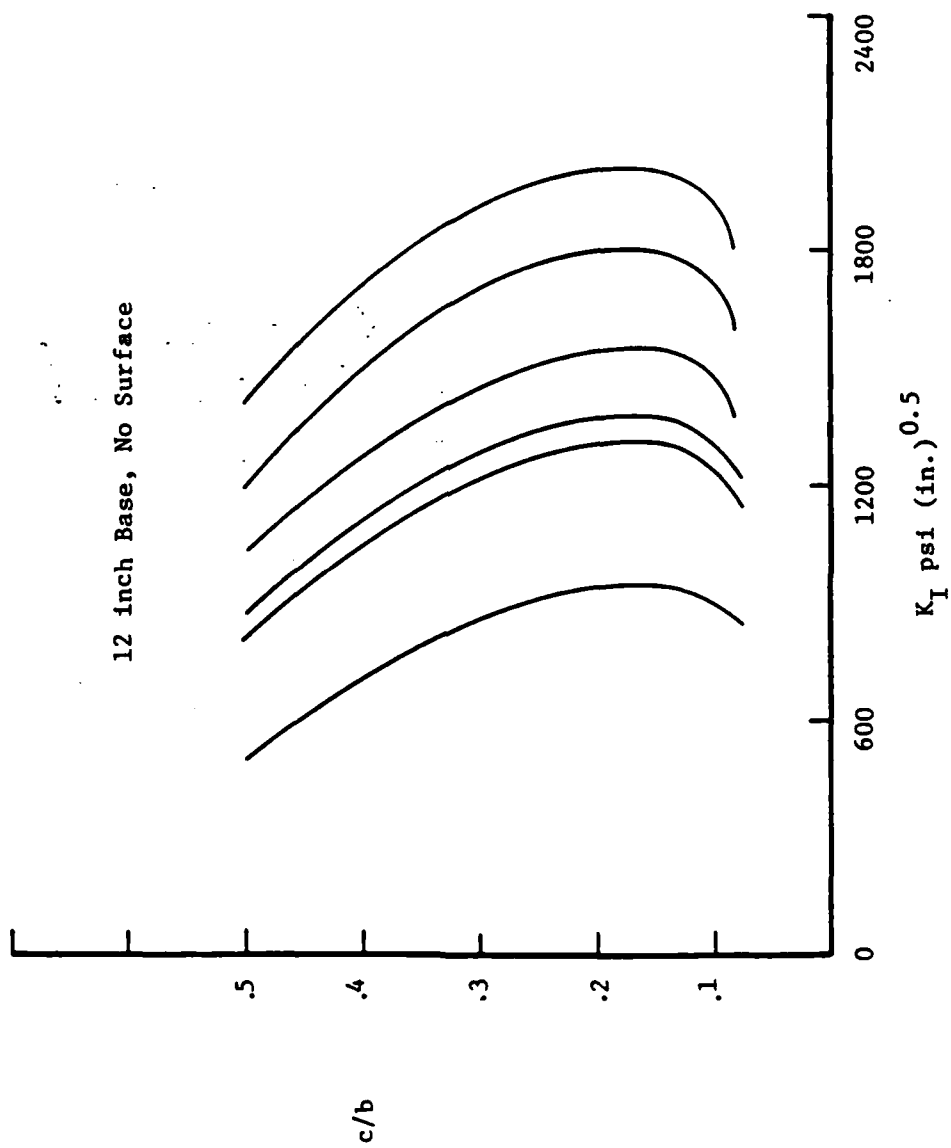


Figure 14. Distribution of Stress Intensity Factor

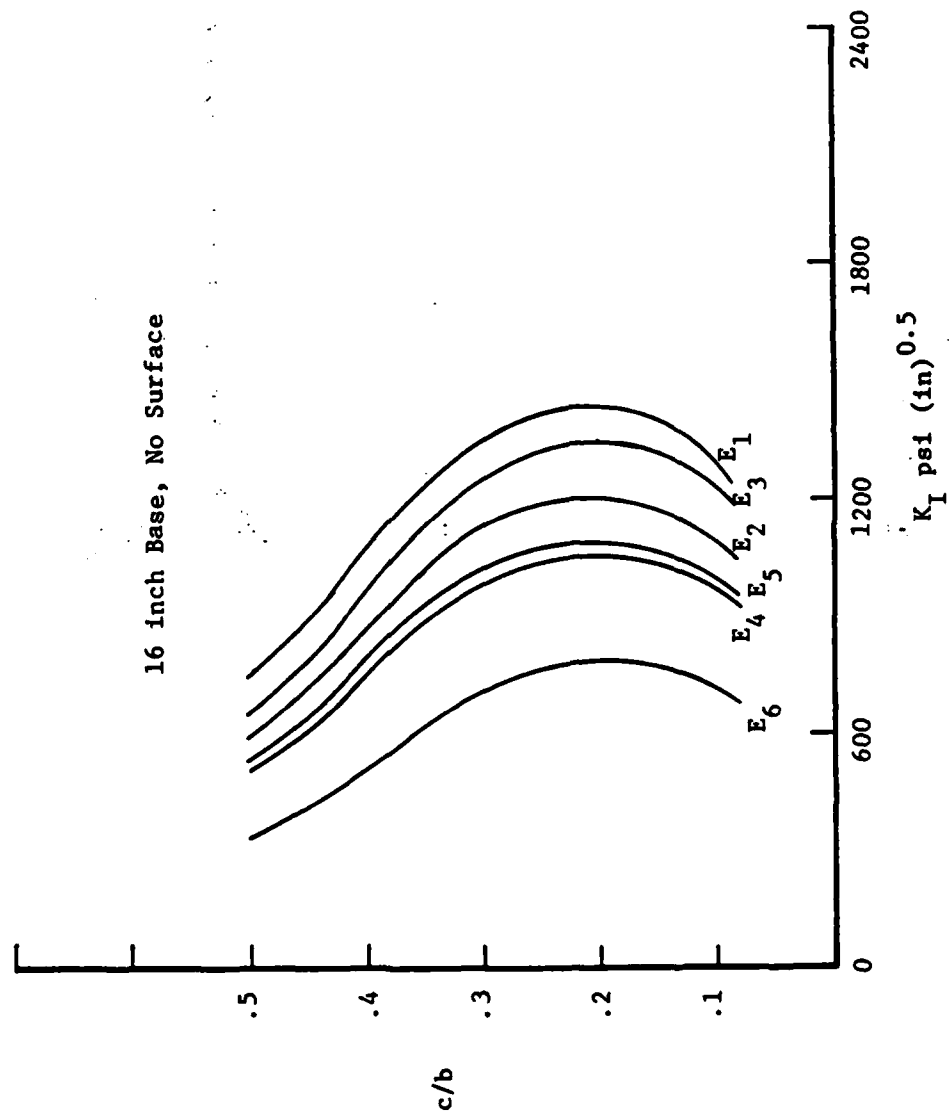


Figure 15. Stress Intensity Factor Distribution

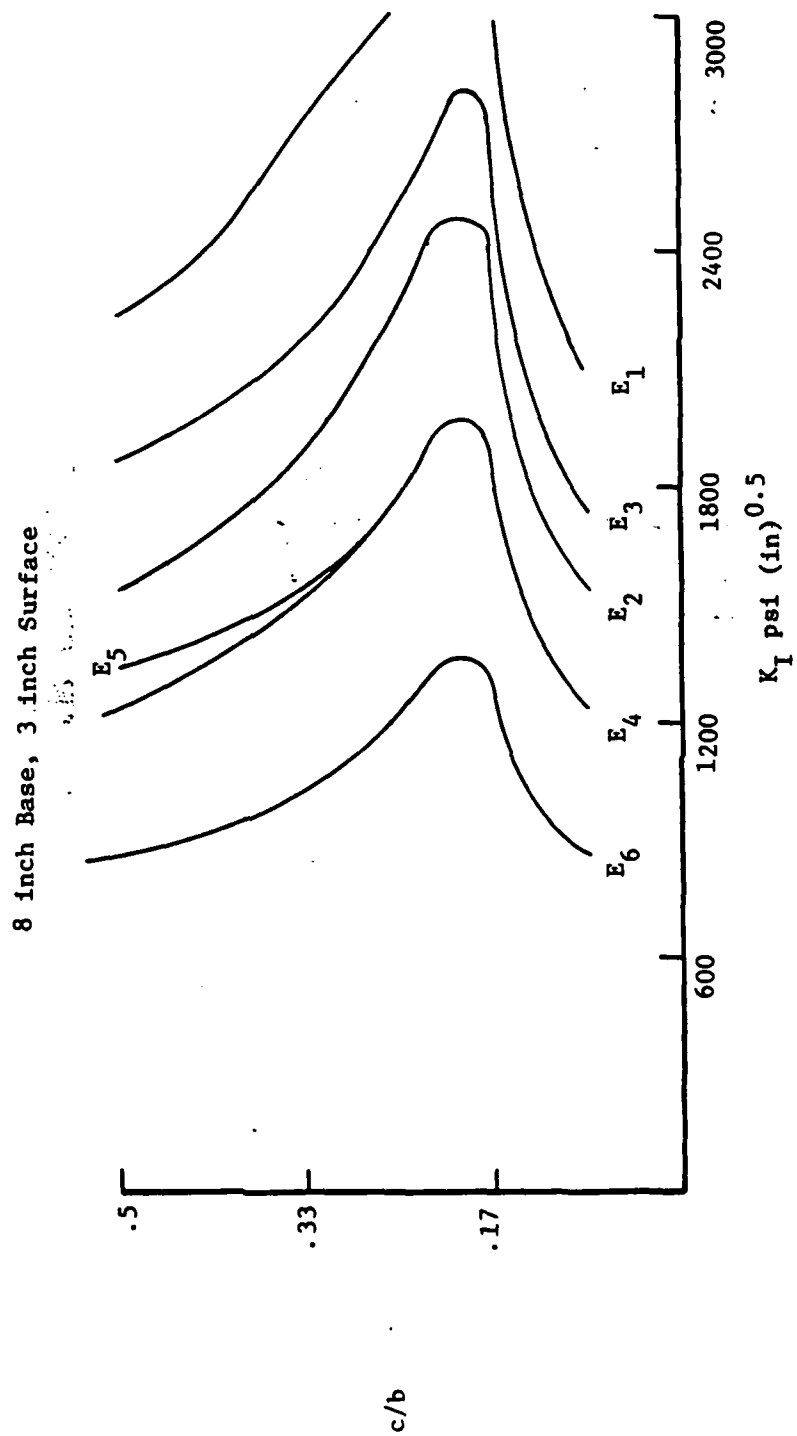


Figure 16. Stress Intensity Factor Distribution

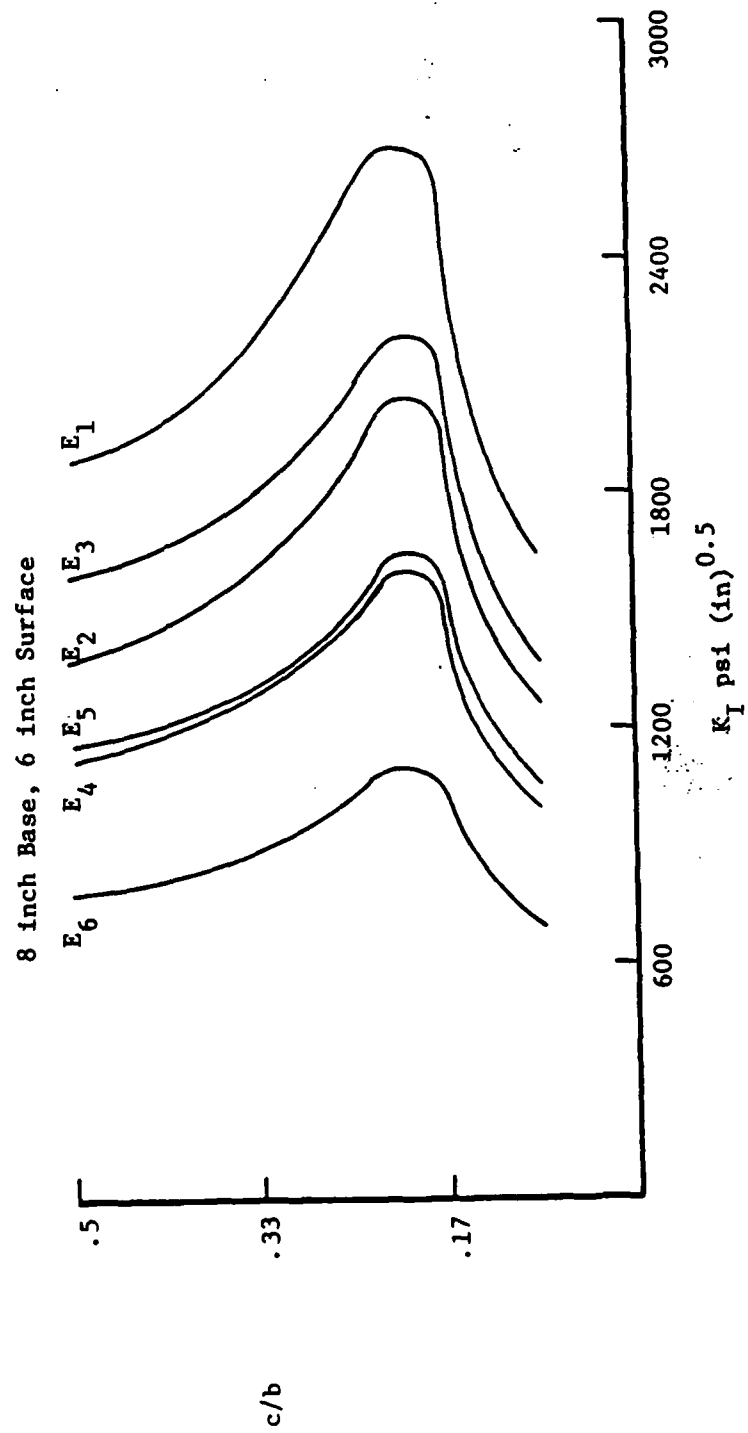


Figure 17. Stress Intensity Factor Distribution

CHAPTER IV: CALCULATING FRACTURE LIFE

Regression Equations for Stress Intensity Factors

In order to compute a fracture life for the pavement system the stress intensity factor distribution must be calculated for every combination of pavement and loading, and then this distribution must be used in the Paris equation. A simpler approach is to develop a series of regression equations to describe the distribution of the stress intensity factor. This eliminates the need to rerun the finite element program a large number of times to develop a series of stress intensity factors differing only slightly from each other. The equation must relate K_I to c/b .

Examining the graphs of the stress intensity factor distribution shown in the previous chapter, it is reasonable to assume a cubic distribution of K_I versus c/b , especially for the cases with no overlay. In order to define this assumed cubic relationship, regression techniques are used. A statistical package on the IBM personal computer was utilized to find the best fit coefficients to a cubic distribution. The results of the polynomial regression analysis are given in Table 10. The R^2 parameters of the regression are also listed. A value of $R^2 = 1.0$ denotes a perfect fit. Note that the fit is very good for the cases with no surface, and satisfactory for those with a surface. Plots of K_I versus c/b for both the cubic distribution and the distributions of the previous section are shown for representative cases in Figure 18 through Figure 22.

For reasons discussed previously, the range of c/b values for which stress intensity factors were computed extend from $1/12$ to $1/2$. For no c/b

Table 10. Regression Coefficients for Cubic Polynomial
Fit to Stress Intensity Distribution.

$$K = B_0 + B_1 (C/B) + B_2 (C/B)^2 + B_3 (C/B)^3$$

MC	B ₀	8 inch base, no surface			R ²
		B ₁	B ₂	B ₃	
1	2047.043	6567.108	-14172.16	5067.807	0.9861128
2	1490.105	4544.518	-10815.35	4237.885	0.9940159
3	1753.576	5513.253	-12415.28	4625.061	0.9903571
4	1260.895	3666.674	-9340.74	3857.532	0.9966238
5	1351.920	3600.485	-8792.111	3060.146	0.9971226
6	938.9210	2345.091	-6942.935	3093.236	0.9988023

MC	B ₀	12 inch base, no surface			R ²
		B ₁	B ₂	B ₃	
1	1297.536	7693.483	-22383.32	14619.22	0.9951294
2	1019.034	5847.388	-17613.80	11676.70	0.9962779
3	1162.857	6804.254	-20087.00	13202.72	0.9957248
4	875.8471	4887.160	-15129.88	10145.02	0.9968587
5	916.8071	5163.552	-15845.97	10587.44	0.9966850
6	654.5362	3373.223	-11169.04	7681.451	0.9978152

MC	B ₀	16 inch base, no surface			R ²
		B ₁	B ₂	B ₃	
1	808.9672	6928.043	-22642.82	17098.41	0.9999410
2	687.1326	5742.700	-19189.89	14661.88	0.9998762
3	776.4540	6547.792	-22345.70	17517.90	0.9998906
4	611.02611	5000.017	-16980.85	13073.25	0.9999341
5	633.80810	5223.816	-17649.63	13556.44	0.9999398
6	472.87540	3650.309	-12930.04	10141.79	0.9998798

MC	B ₀	8 inch base, 3 inch surface			R ²
		B ₁	B ₂	B ₃	
1	-394.5381	37465.05	-117437.1	105884.90	0.8500299
2	-235.5056	26687.00	-84582.00	76535.600	0.8609347
3	-280.6712	30515.79	-96126.81	87240.120	0.8503666
4	-137.2445	20948.11	-66598.86	60381.990	0.8641640
5	-112.6183	20547.46	-64805.32	59002.900	0.8512328
6	-12.82105	13339.98	-42658.10	38802.090	0.8696607

MC	B ₀	8 inch base, 6 inch surface			R ²
		B ₁	B ₂	B ₃	
1	-247.4966	28694.99	-89707.74	81544.99	0.8456994
2	-148.9915	21328.18	-67057.85	60936.81	0.8496895
3	-153.3865	23050.09	-71954.05	65549.56	0.8465933
4	-72.99951	16591.39	-52166.28	47507.98	0.8494838
5	-35.03687	15331.10	-48342.45	44153.58	0.8497349
6	17.19202	10486.71	-32985.49	30127.45	0.8487314

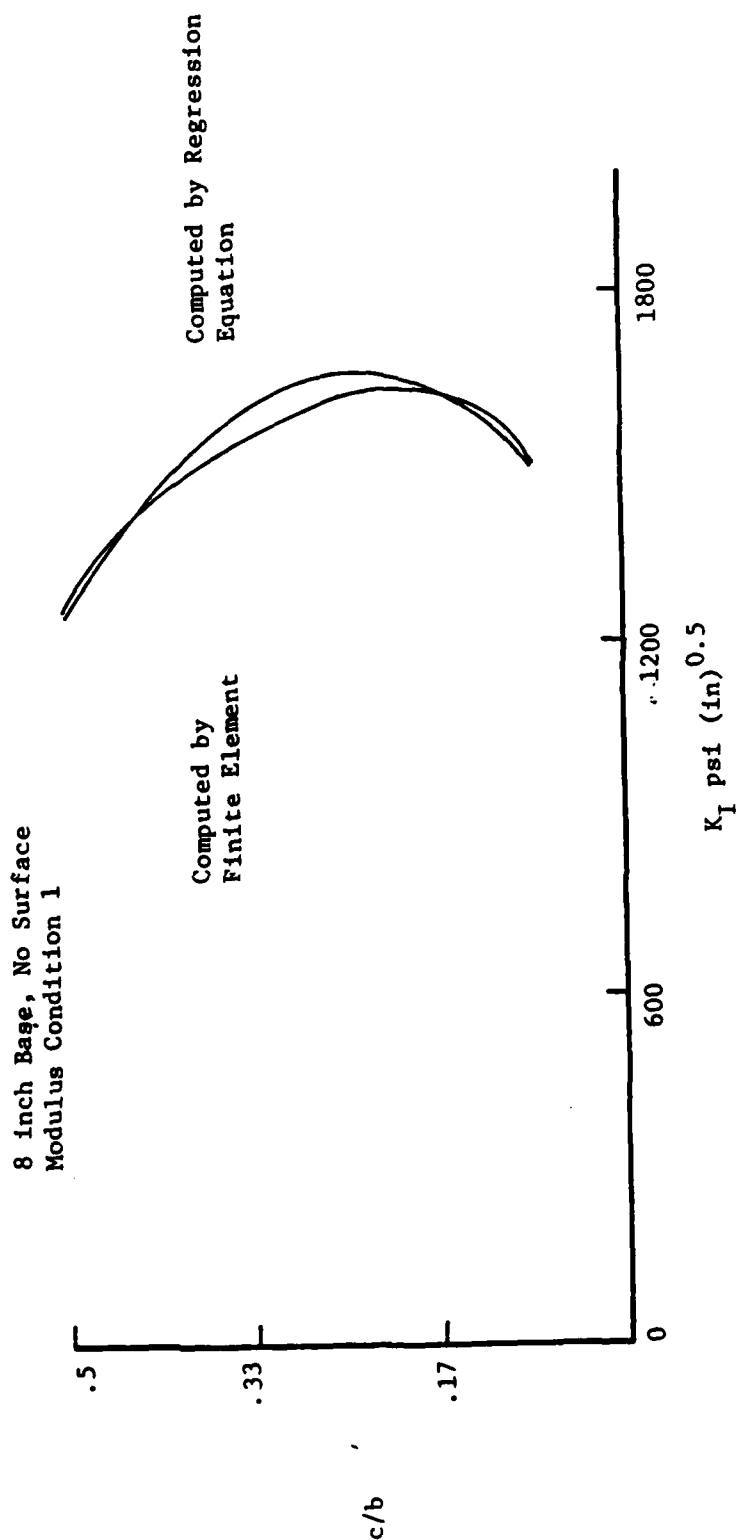


Figure 18. Comparison of Computed and Regression Stress Intensity Factors

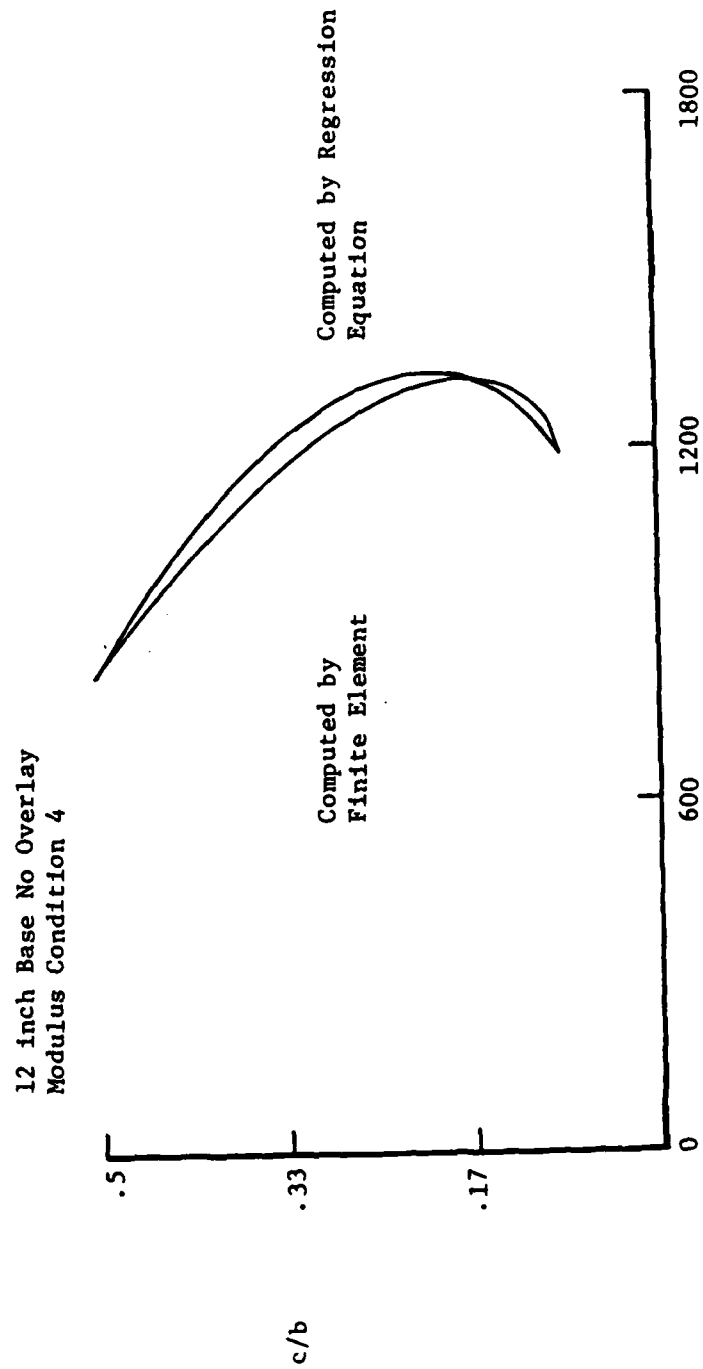


Figure 19. Comparison of Computed and Regression Stress Intensity Factors

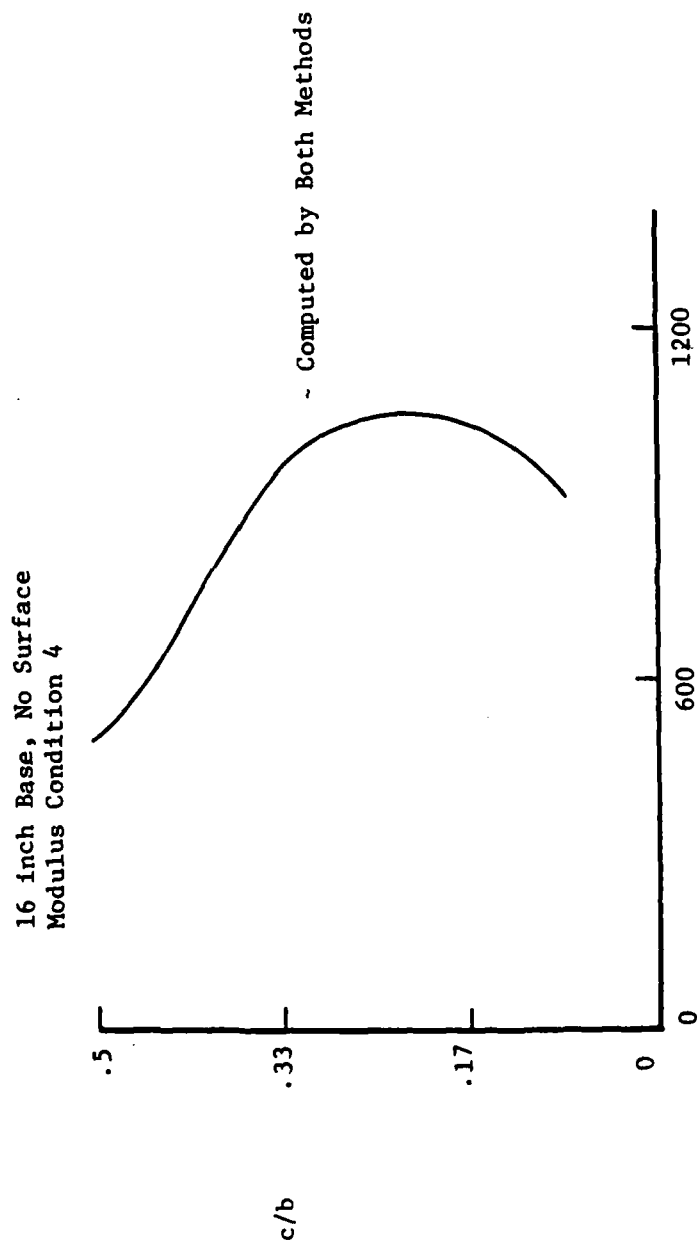


Figure 20. Stress Intensity Factor Distribution

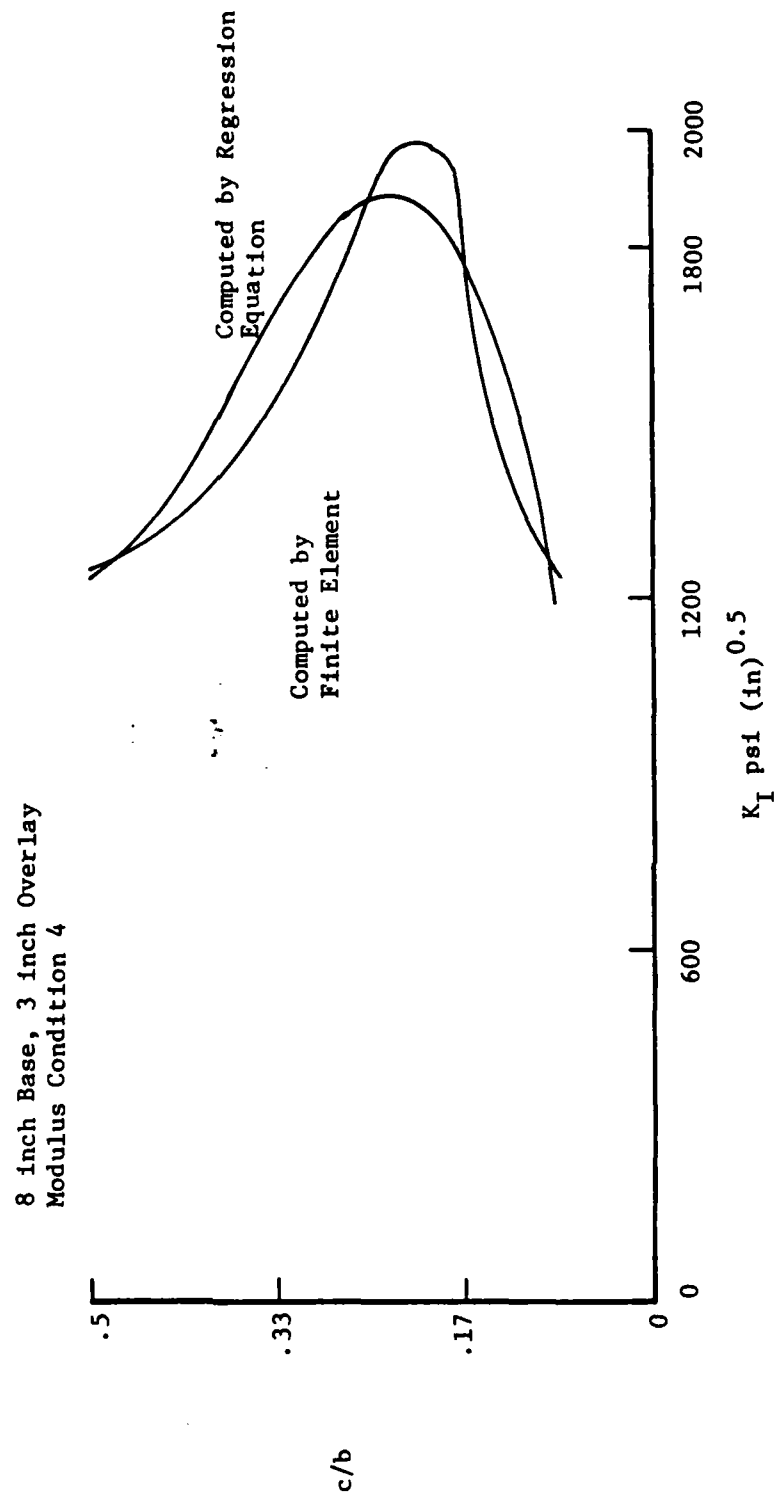


Figure 21. Distribution of Stress Intensity Factor

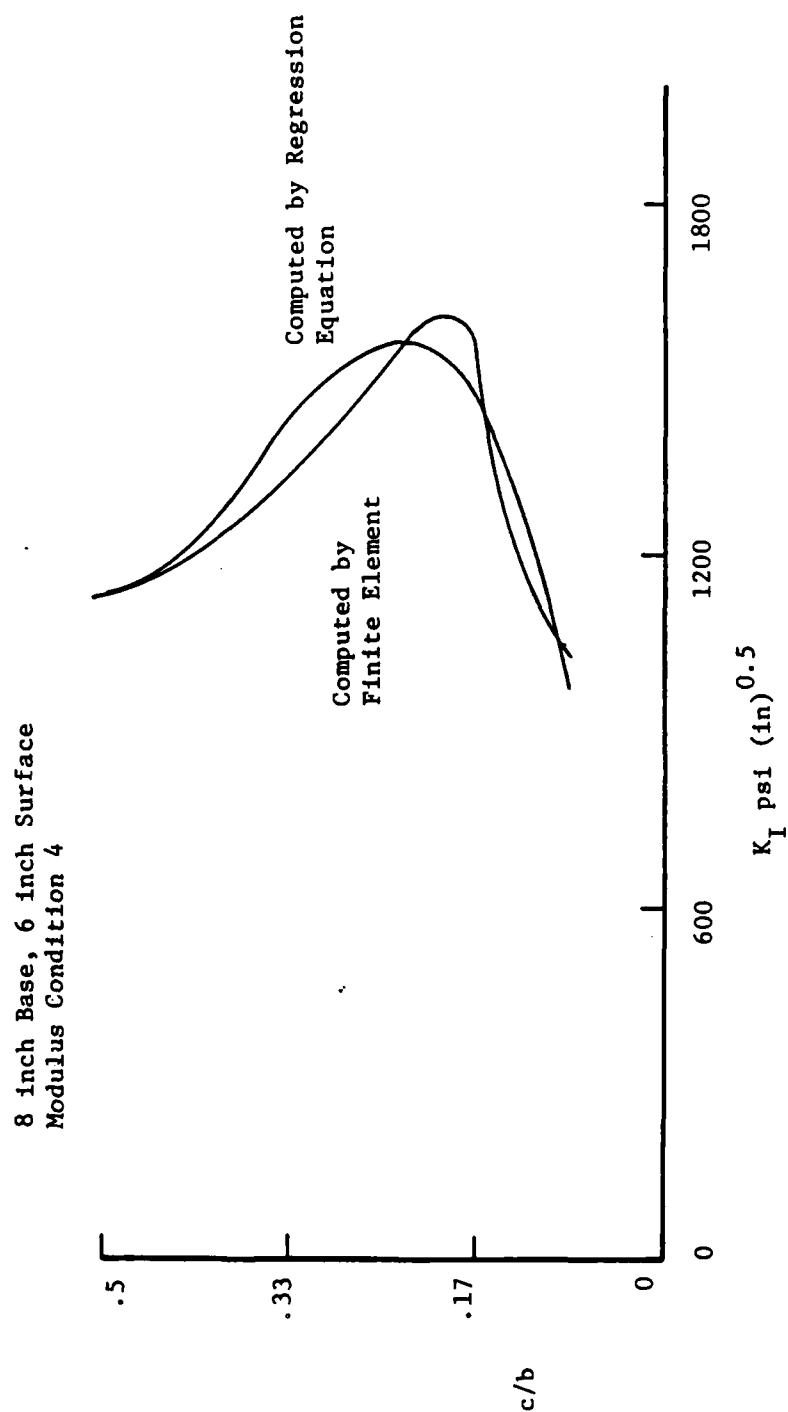


Figure 22. Distribution of Stress Intensity Factors

values greater than $1/2$ were stress intensity factors calculated. Thus, for c/b greater than $1/2$, the assumed cubic distributions give the only K_I values; they must be relied on exclusively. The graphs of the predicted distributions for c/b greater than $1/2$ are shown in Figure 23 through 27. As can be seen, the plots for all cases get smaller directly after $c/b = 1/2$, and then double back and increase. For the 8 inch base, no surface, the K_I values go negative and never become positive again. For the rest of the cases, K_I is always positive past $c/b = 1/2$, and for the 8 inch base, 3 and 6 inch surface cases K_I gets very large at $c/b = 1$. In addition, noting that B_0 equals K_I at $c/b = 0$, it can be seen that for the 8 inch base, 3 and 6 inch surface cases K_I is negative at $c/b = 0$. The exception is modulus condition 6 of the 6 inch surface, where K_I manages to remain positive at $c/b = 0$.

The predicted K_I values at $c/b = 0$ cannot be checked (there is no crack length on which to apply reverse stresses), but for c/b greater than $1/2$ the cubic values should be checked with calculated K_I values. It may be that the cubic distributions are misleading in this range, but these values have not been checked.

Computation of Crack Propagation Histories

To calculate the rate of crack propagation, the Paris equation described in Chapter II was coded into a Fortran program to allow easy application of stress intensity factors, pavement geometry and structural properties into a program that predicted the number of load cycles to failure. Up to this point, all the necessary input has been calculated except for the initial and final crack lengths for each modulus condition, base, and surface

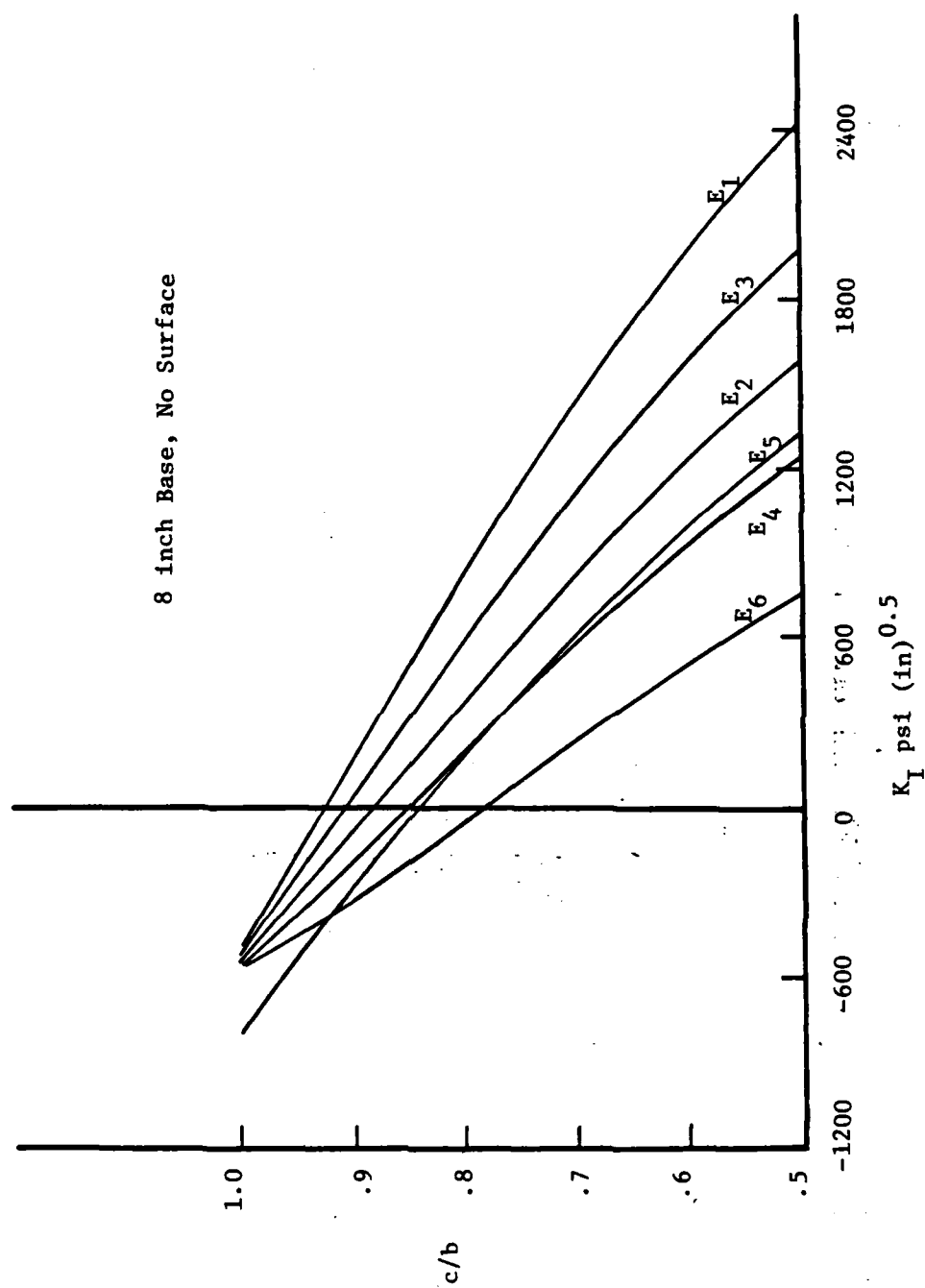


Figure 23. Distribution of Stress Intensity Factors Calculated by Regression Equation for Various Modulus Conditions

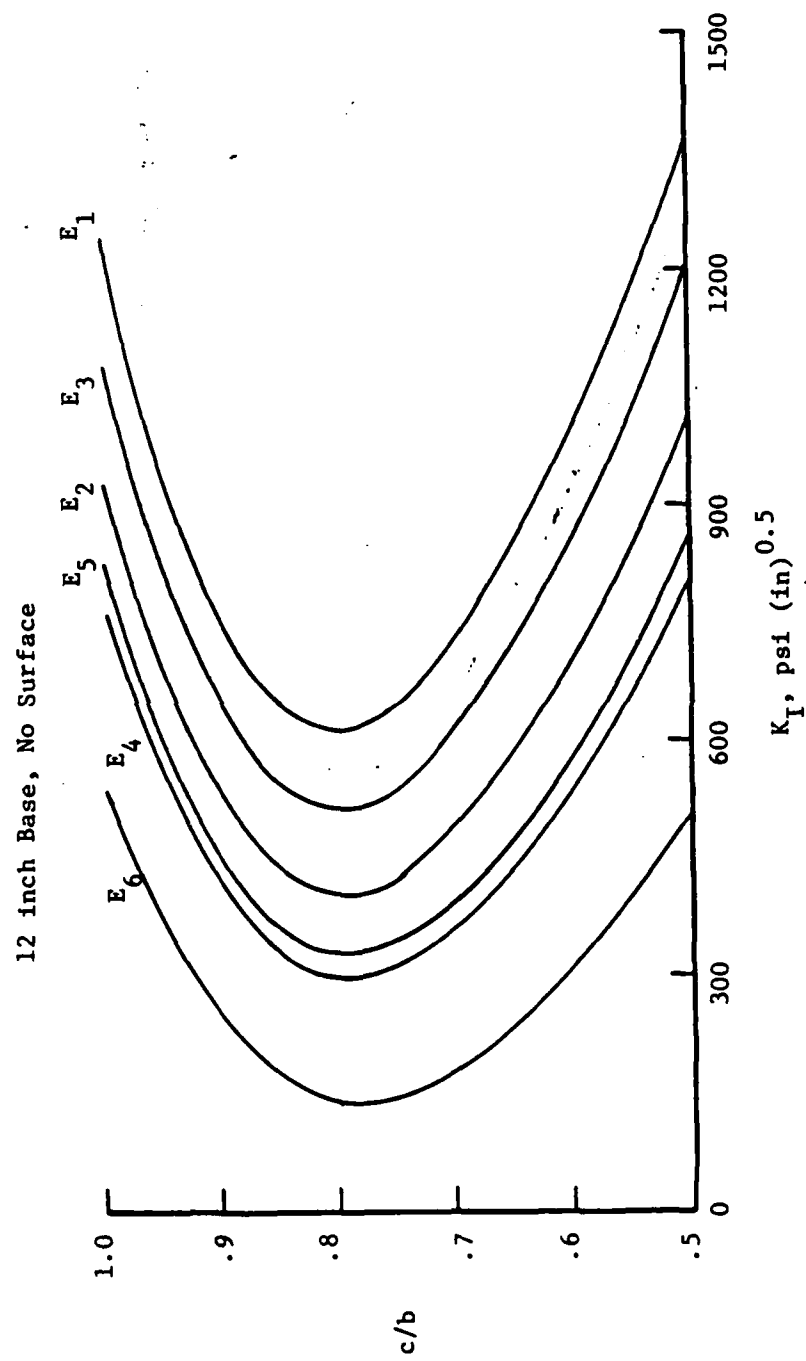


Figure 24. Distribution of Stress Intensity Factors Computed by Regression Equation for Various Modulus Conditions

16 inch Base, No Surface

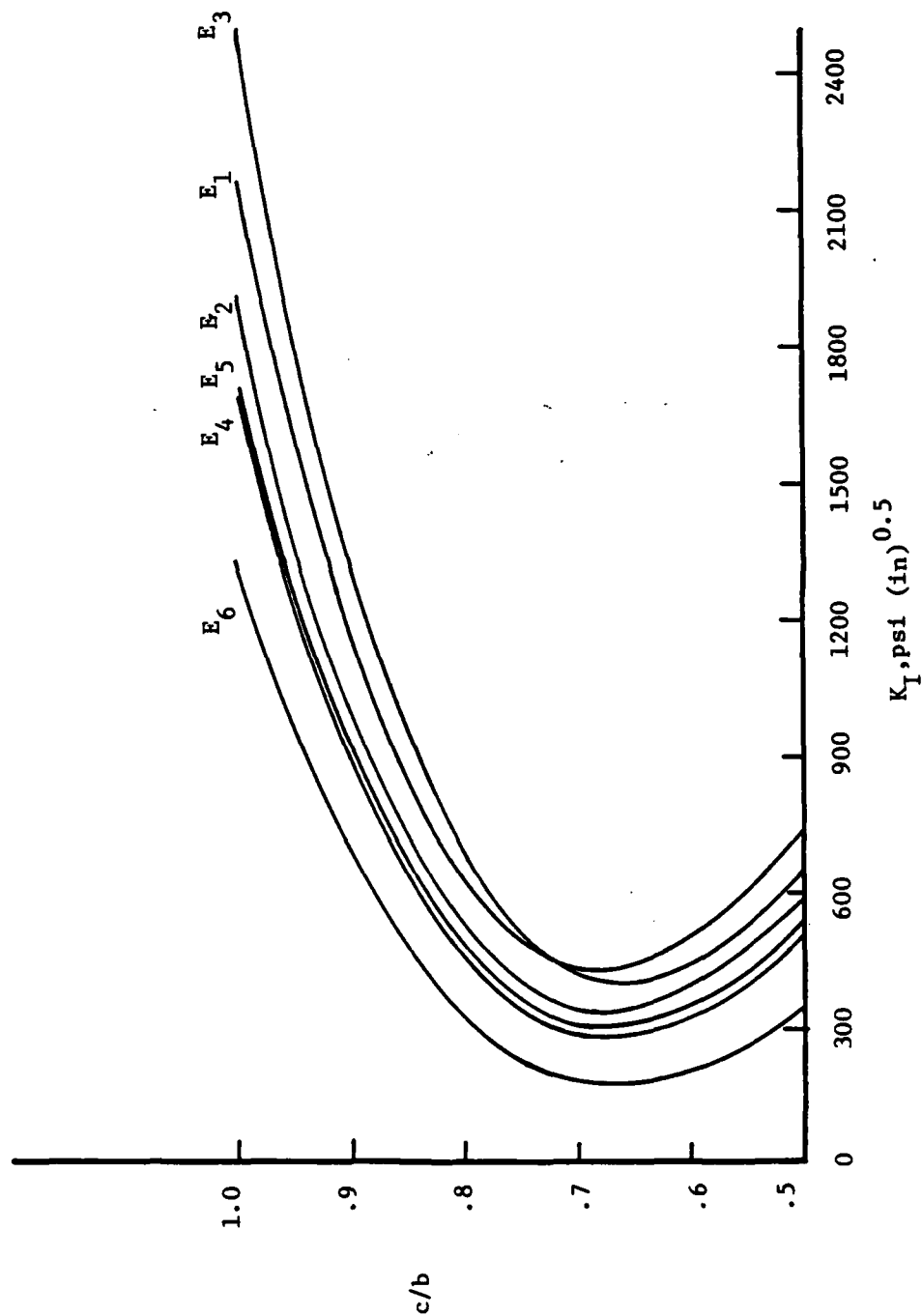


Figure 25. Distribution of Stress Intensity Factors Computed by Regression Equation for Various Modulus Conditions

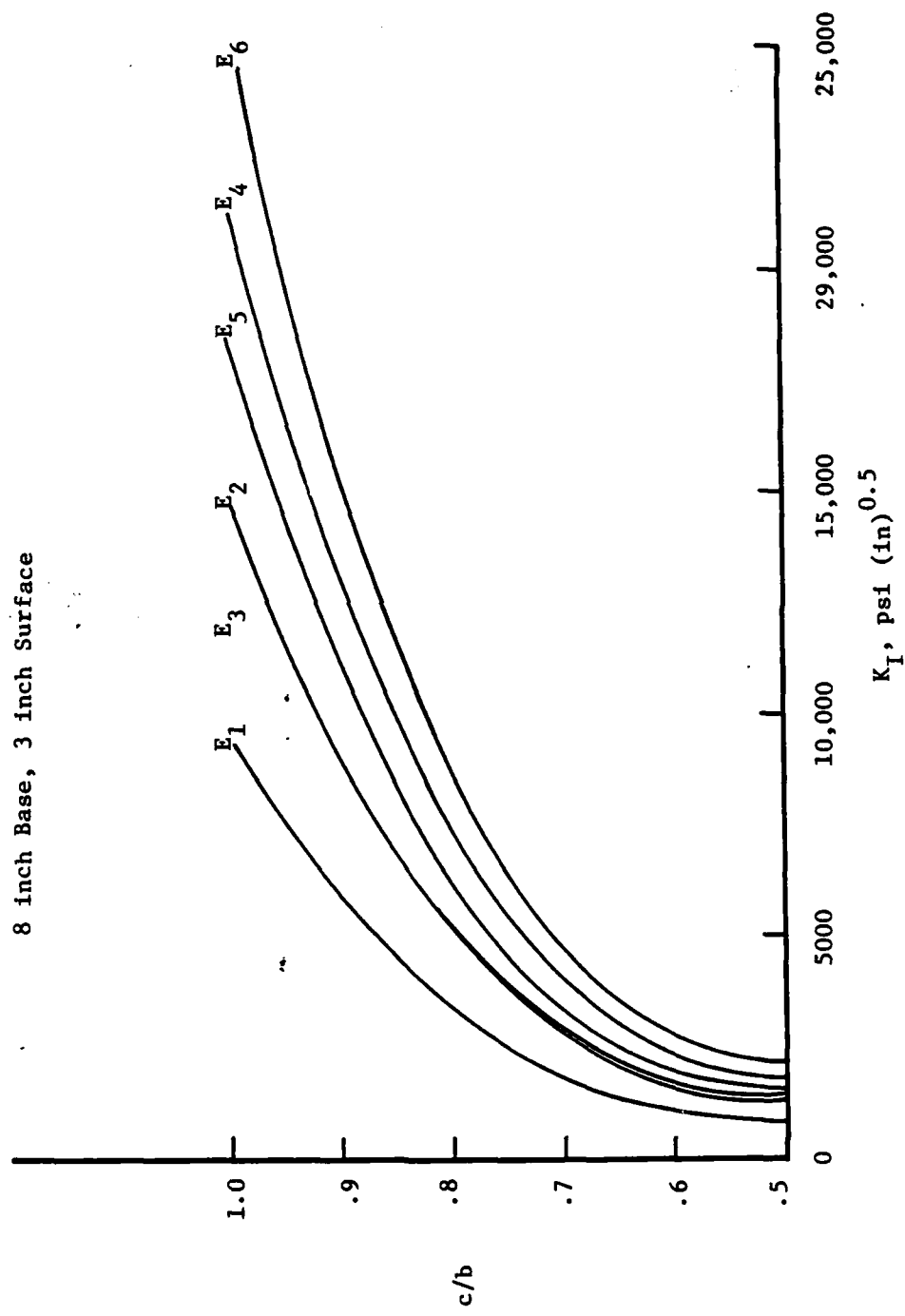


Figure 26. Distribution of Stress Intensity Factors Computed by Regression Equation for Various Modulus Conditions

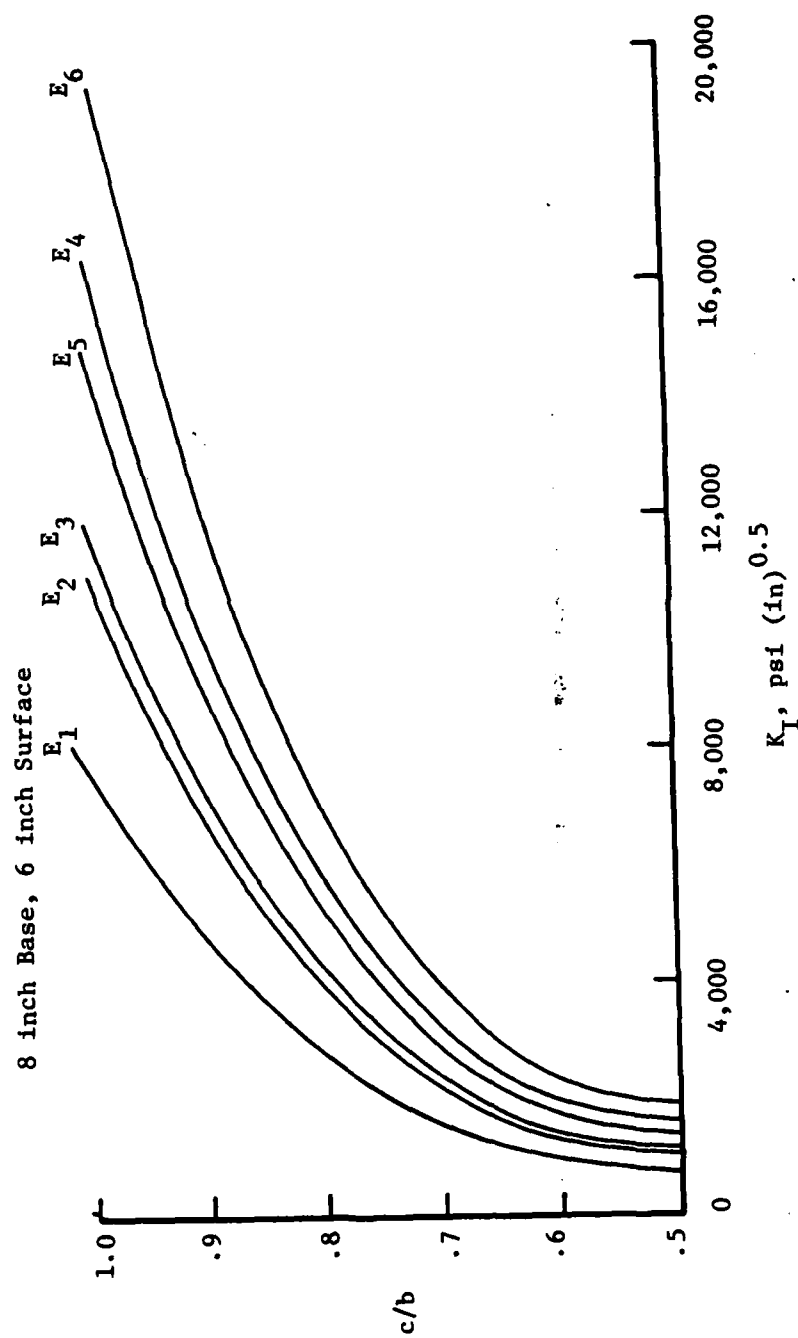


Figure 27. Distribution of Stress Intensity Factors Computed by Regression Equation for Various Moduli Conditions

combination. These are found by calculating the roots of the K_I cubic distributions. The first two roots in the range $0 \leq c/b \leq 1$ which define a positive region of K_I are the endpoints, as explained in Chapter 1. If K_I is positive in the range $0 \leq c/b \leq 1$, then the endpoints are 0 and 1. The endpoints found by using this procedure are shown in Table 11. When input, if the endpoint in question is a root, then it is modified slightly so that it does not assign a zero (or negative due to round-off errors) value to K_I and thus cause computational errors. If it is a right endpoint, it is made smaller, and if it is a left endpoint it is made larger so that K_I is always positive.

The crack propagation histories for modulus condition 4, all base and surface combinations, are given in Figure 28 through Figure 32. These histories correspond to arbitrarily selected values of A and n equal to 10^{-10} and 2.0 respectively. The numbers defining these graphs can be found in the output in Appendix B, as mentioned before. The histories graphed are typical in their relationship to each other for each modulus condition and A and n combination.

It appears from the graphs that of the pavements with no surface, the 8 inch base is the most critical, followed by the 12 and then 16 inch bases. Although the number of cycles required to traverse the last crack increment in the 8 inch base is five orders of magnitude greater than the total to failure of the 12 and 16 inch bases, the load cycles needed to reach near failure in the 8 inch base is significantly smaller than the aforementioned totals. It is reasonable to believe that in such a state of near failure some perturbation could advance the crack to failure. Besides, the 8 inch base is practically destroyed at near failure anyway.

Table 11. Endpoints For Crack Propagation Histories

8 inch base, no surface		
MC	LEFT	RIGHT
1	0.0	0.925978
2	0.0	0.880403
3	0.0	0.905504
4	0.0	0.849452
5	0.0	0.837094
6	0.0	0.784196

12 inch base, no surface		
MC	LEFT	RIGHT
1	0.0	1.0
2	0.0	1.0
3	0.0	1.0
4	0.0	1.0
5	0.0	1.0
6	0.0	1.0

16 inch base, no surface		
MC	LEFT	RIGHT
1	0.0	1.0
2	0.0	1.0
3	0.0	1.0
4	0.0	1.0
5	0.0	1.0
6	0.0	1.0

8 inch base, 3 inch surface		
MC	LEFT	RIGHT
1	0.010900	1.0
2	0.009084	1.0
3	0.009478	1.0
4	0.006693	1.0
5	0.005579	1.0
6	0.000964	1.0

8 inch base, 6 inch surface		
MC	LEFT	RIGHT
1	0.008869	1.0
2	0.007145	1.0
3	0.006798	1.0
4	0.004462	1.0
5	0.002302	1.0
6	0.0	1.0

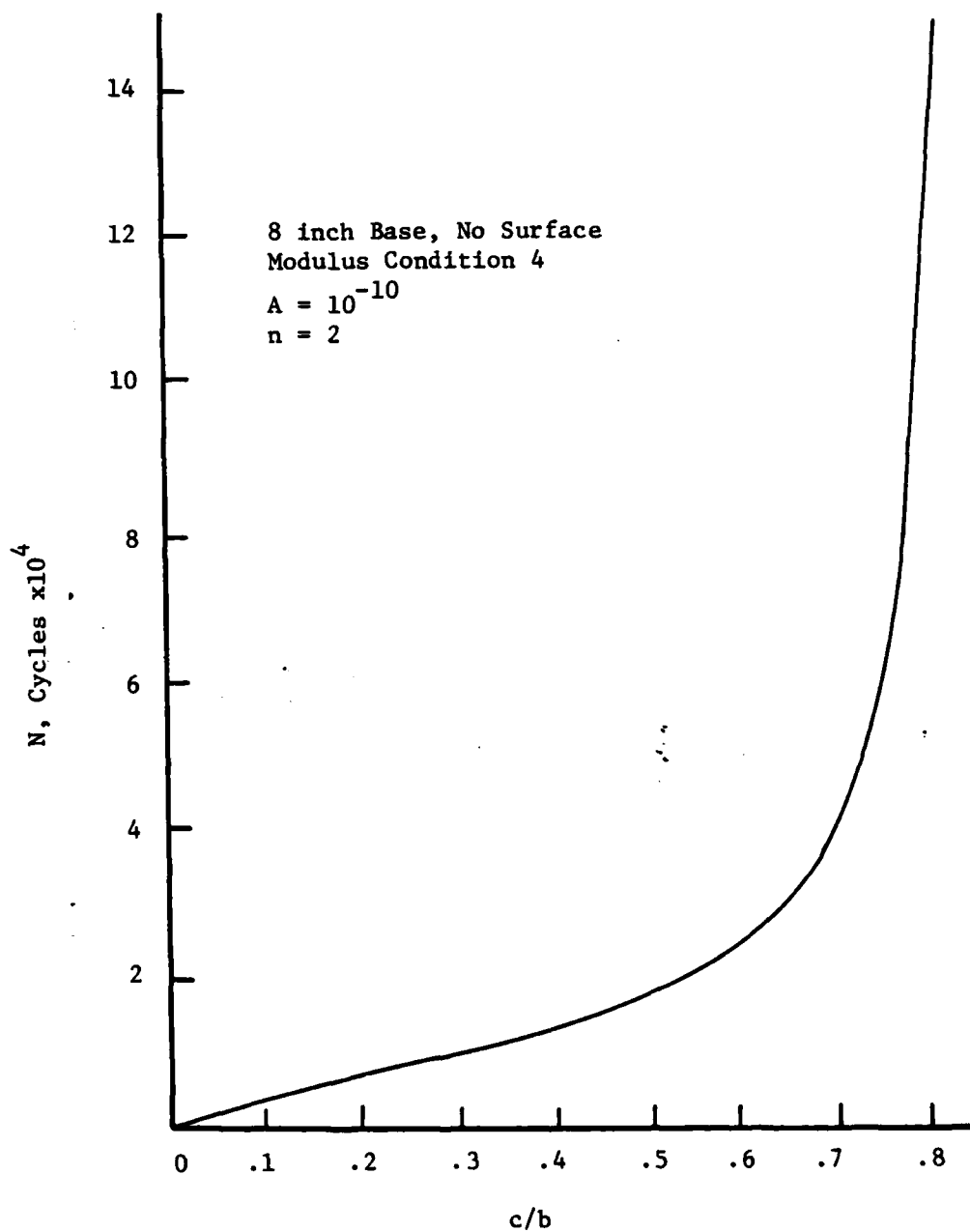


Figure 28. Progression of Crack (c) Through Base (b) as a Function of Load Cycles, N

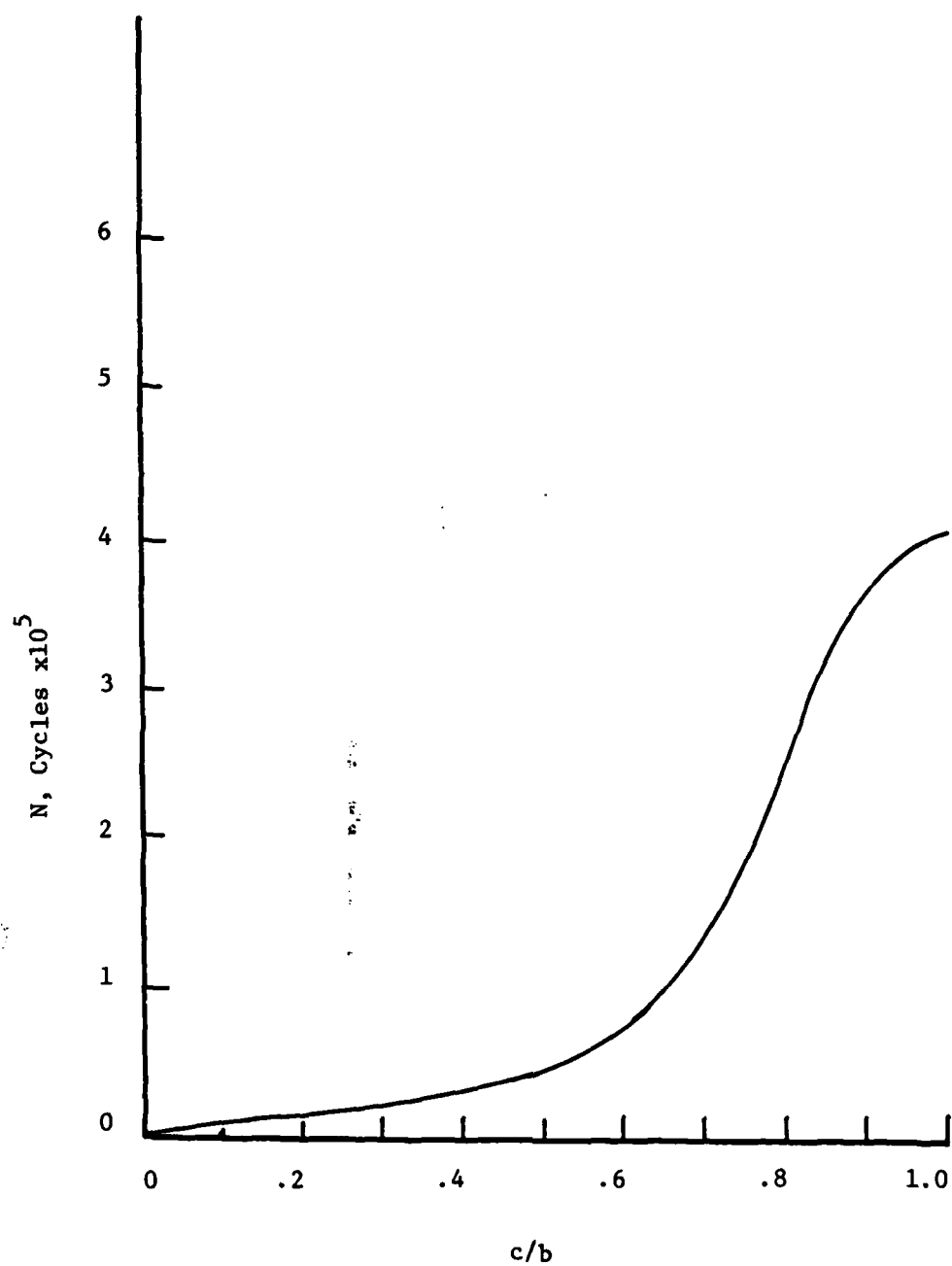


Figure 29. Progression of Crack (c) Through Base (b) as a Function of Load Cycles, N

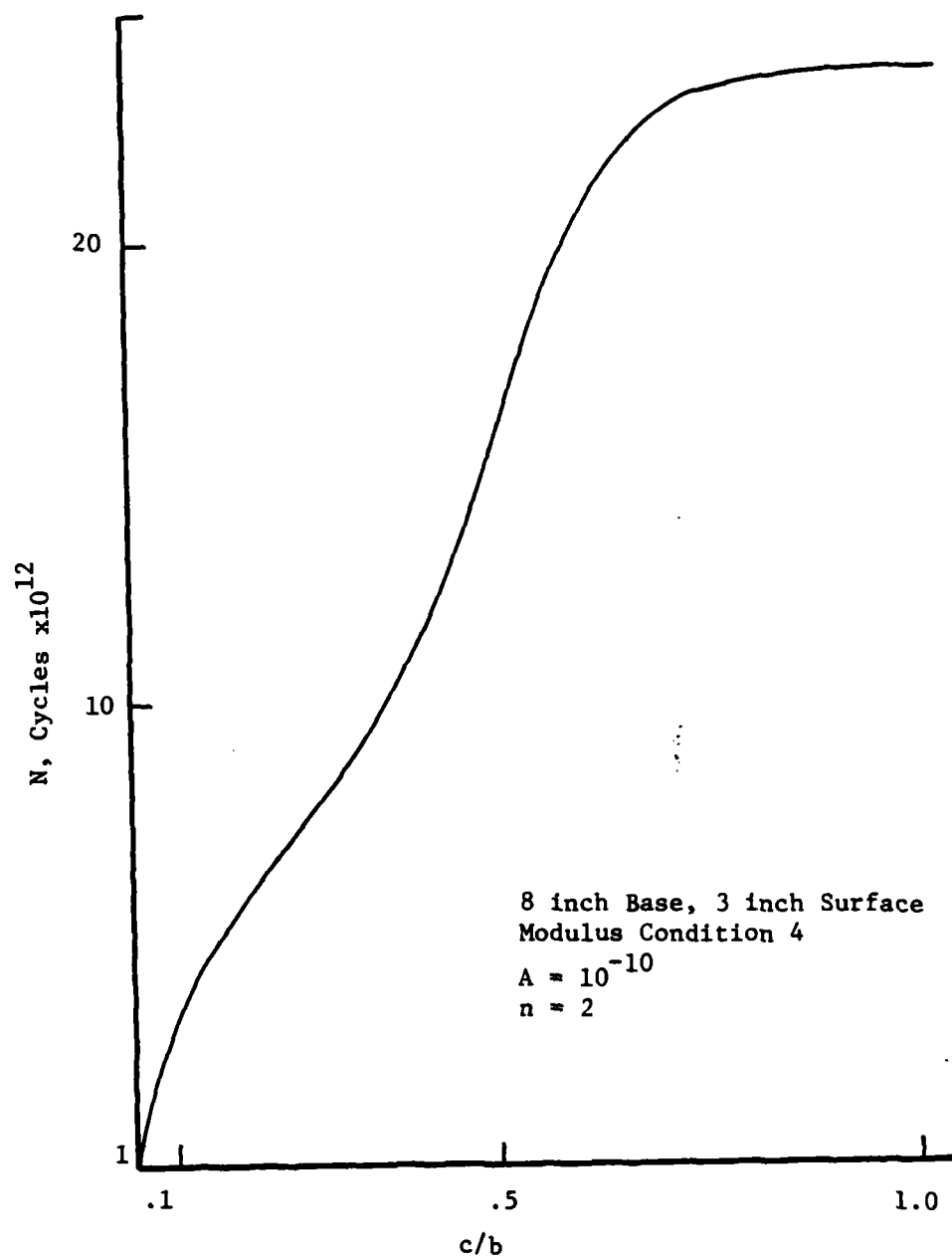


Figure 30. Progression of Crack (c) Through Base (b) as a Function of Load Cycles, N

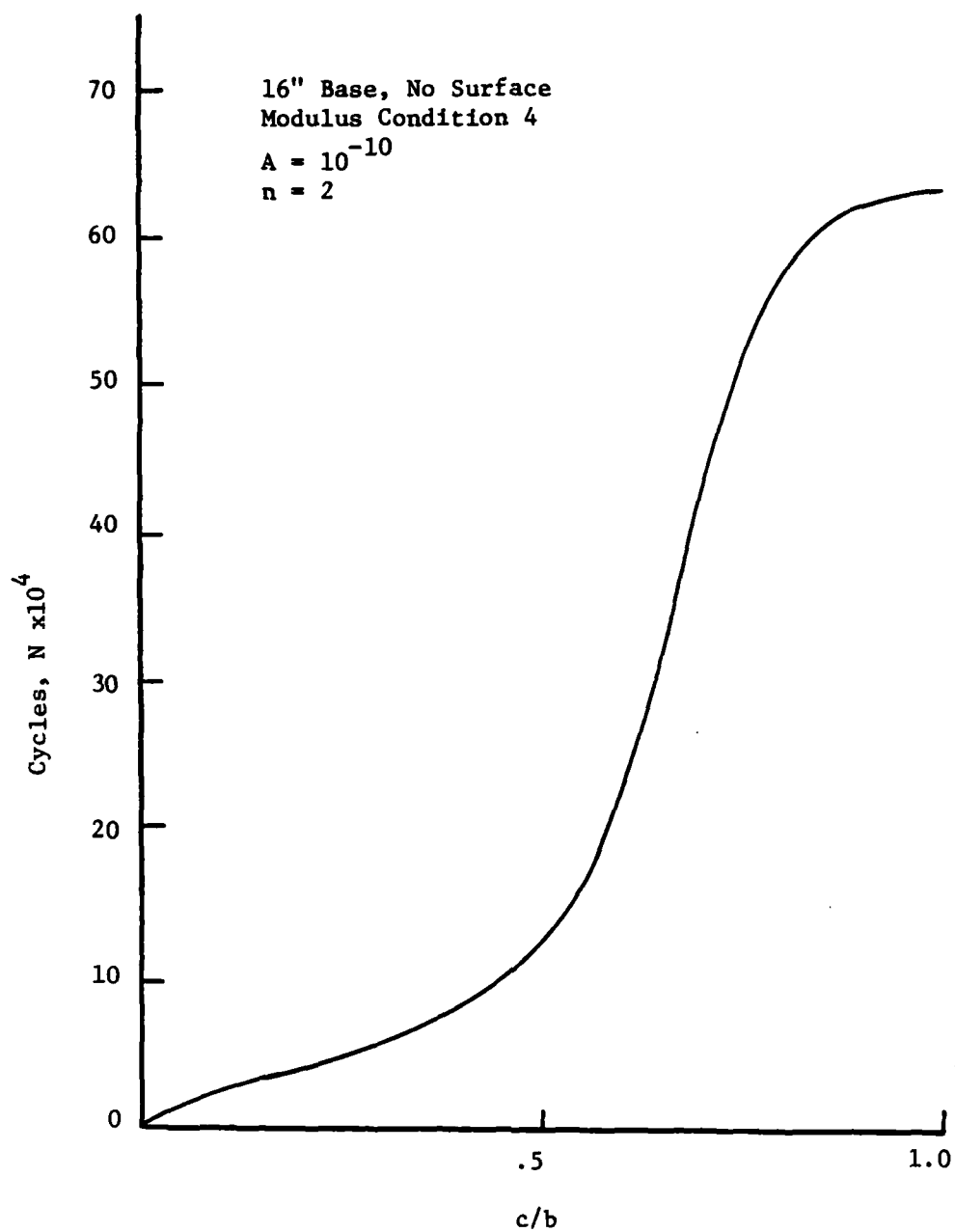


Figure 31. Progression of Crack (c) Through Base (b)
as a Function of Load Cycles, N

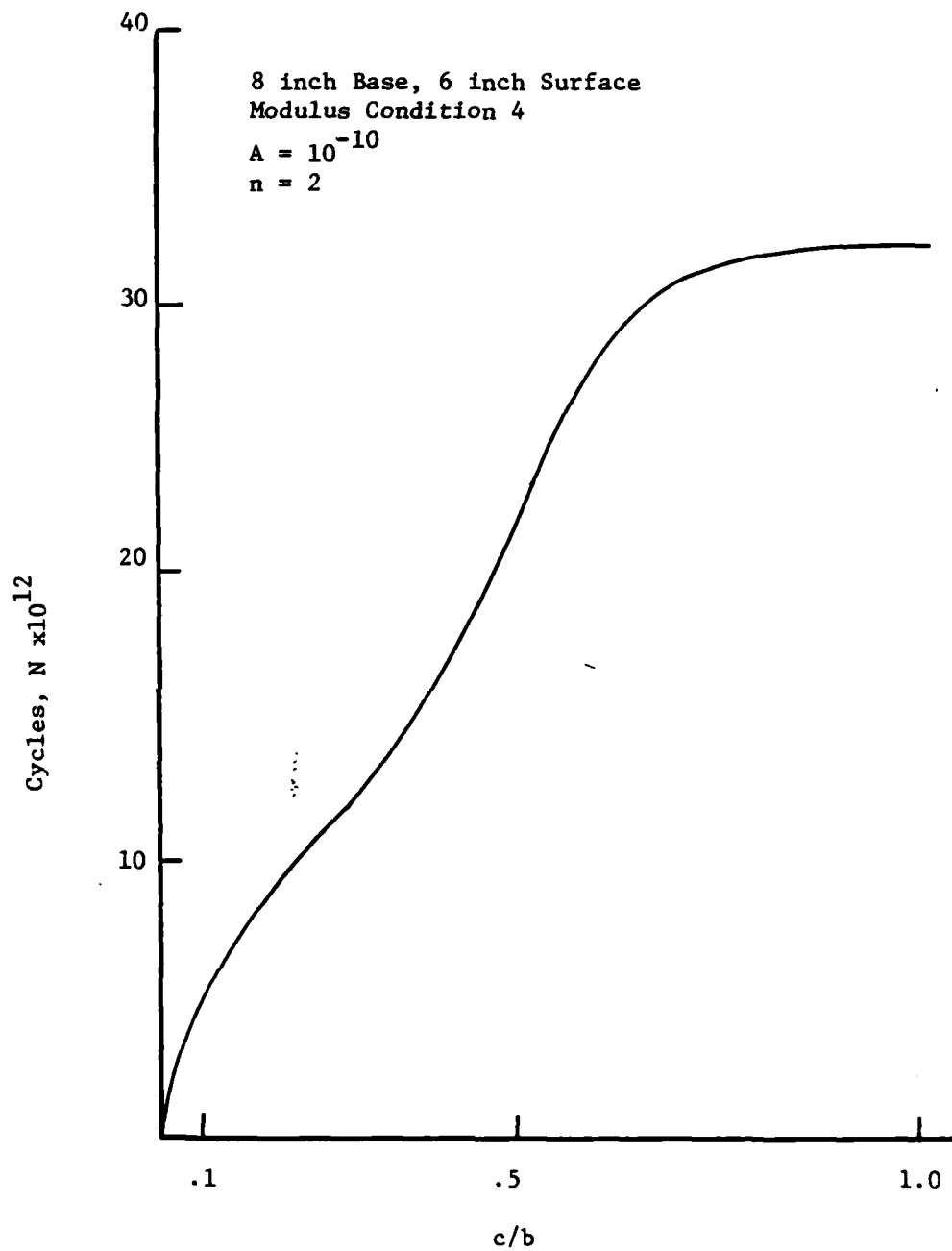


Figure 32. Progression of Crack (c) Through Base (b) as a Function of Load Cycles, N

The 3 and 6 inch surfaces are added to the 8 inch base with the intent of increasing the fatigue life. From the graphs it is seen that theoretically they do, but perhaps realistically they won't. In both the 3 and 6 inch surface cases, it takes a tremendous amount of load cycles to advance the crack through the first increment of approximately 0.05 inches. After that, though, it takes less cycles to propagate the crack to failure than it does for the crack to reach near failure in the no surface case. Therefore, if the crack were somehow jumped past the initial increment in the 3 and 6 inch surface cases, due to say a starter flaw, which may typically have dimensions capable of doing this, those pavements would be more critical than the no surface case. It is certainly reasonable to think that the first increment could be jumped, as it is assumed that a zone of negative K_I is jumped to get to the first increment in the first place. In this case the negative zone is only about 0.01 inches at most, so that it is no great achievement to bypass it. So, in one sense, adding a surface will greatly postpone failure as long as the negative K_I zone and the first crack increment are not vaulted. In another sense, if the above were circumvented, adding a surface could hasten failure.

As would be expected, the 3 inch surface case requires more cycles to failure than the 6 inch surface case for the same pavement structure.

Calculations With Actual Laboratory Fracture Properties

Extensive laboratory studies conducted at Texas A&M University have indicated that the fracture parameters, A and n can be predicted from tests along with K_{Ic} for stabilized materials. These parameters are necessary to predict the fracture life of a material insitu. Typical values determined are as follows:

CEMENT LEVEL	E=355,600 psi	v = 0.15
10 %	CREEP VALUES	A=8.12x10 ⁻⁵⁹
K _{IC} =138.6		n=25.28
	FATIGUE VALUES	A=15.4x10 ⁻²⁷
		n=10.0

	E=597,600 psi	v = 0.15
15 %	CREEP VALUES	A=1.45x10 ⁻⁵³
K _{IC} =209.3		n=21.24
	FATIGUE VALUES	A=10x10 ⁻⁵¹
		n=22.0

Comparison of these critical stress intensity factors indicates that the stress intensity factors calculated using the procedures outlined here are all larger than the critical stress intensity factors which means that fracture will proceed instantaneously. This indicates the inaccuracy of the pavement structure selected for modelling the stress intensity factors calculated.

CHAPTER V: CONCLUSIONS AND RECOMMENDATIONS

The analysis procedure presented in this report illustrates the complexities involved in calculating stress intensity factors, and applying them to a structure to predict cycles to failure. The stress intensity factors used in this analysis are not as accurate as may be desired, due to the use of plane strain, plane stress solutions used in the finite element procedure to calculate the initial stresses. The modification of more sophisticated programs to accept the fracture elements in an axi-symmetric analysis would increase the accuracy of the predictions of fracture in the cement stabilized base materials, or any material susceptible to fracture.

The methodology of applying the principles of fracture, as demonstrated in the development of this report are applicable to a design methodology, and indicate the modularized approach which must be taken to adequately characterize the pavements, their geometry, and the loading conditions which may be expected to occur in an actual airfield pavement.

The principles of fracture mechanics and crack propagation can be applied to pavement design life, to account for the gradual failure due to repeated loading over fracture susceptible materials. The parameters required to predict crack propagation are the material properties of A and n , as described in this report. A significant amount of testing has gone into evaluating these parameters for cement stabilized materials, and the influence of these material properties on fracture life was shown.

The predicted stress intensity factors were high, due directly to the level of structure modeled and the magnitude of loading investigated.

A broader range of loading conditions, and a thicker pavement structure would have produced more moderate stress intensity factors which would have shown a more pronounced effect on fracture life than was seen in this report.

1. The methodology developed in this study indicates the potential for considering fracture in pavement design methodology.
2. The stress calculation scheme can be refined and replaced by a general program to analyze an axisymmetric problem with multiple wheel loadings. This can be a simple elastic layer program, a stress dependent elastic layer program, or even a sophisticated stress dependent finite element program such as Illi-Pave.
3. The hybrid crack tip element provides a simple means of accurately evaluating the influence of loading on fracture life in fracture susceptible layers of a pavement system given the stresses present in the layers.
4. The modular approach developed to provide stress intensity factors using regression techniques to replace repetitive calculations with the finite element program is an accurate procedure to obtain the stress intensity factors in a manner applicable to design without extensive use of computer execution time.

REFERENCES

1. Chang, Hang-Sun, "Prediction of Thermal Cracking in West Texas," Masters Thesis, Texas A&M University, Department of Civil Engineering (1975).
2. Desai, C. S., and Abel, J. F., Introduction to the Finite Element Method, Van Nostrand Reinhold Company, 1972.
3. Pian, T.H.H., Tong, P., Luk, C. H., "Elastic Crack Analysis by a Finite Element Hybrid Method," (paper) 3rd Conf. Matrix Meth. Struct: Mech., Wright-Patterson AFB, Ohio (1971).
4. Pian, T.H.H., Tong, P., Lasry, S. J., "A Hybrid Element Approach to Crack Problems in Plane Elasticity," *IJNME*, Vol. 7, 1973, pp. 297-308.
5. Tong, P., Lin, K. Y., Orringer, O., "Effect of Shape and Size on Hybrid Crack-Containing Finite Elements," *Computational Fracture Mechanics*, ASME, 1975.

APPENDIX A: NOTATION AND CONVERSION FACTORS

SYMBOL	MEANINGS
A	regression constant in fracture equation; area in other cases
A_m	Element areas in finite element development
b	base course thickness in fracture analysis
c	crack length within a pavement layer
C_K	stress intensity correction factor
$C_{i,j,k,l}$	Elastic Compliance Tensor
e	2.71828
E	Young's modulus of elasticity
G	shear modulus
K	stress intensity factor
m	creep exponent
n	regression constant (exponent) in fracture equation
N	number of cycles in fracture
P	load
q	nodal displacements
R	statistical correlation coefficient
r	radius
S	Surface for integration
T	Traction vector normal to integration path
u	boundary displacement
x,y	x_1, y_1 directions respectively
ϵ	error in stress intensity predictions, strain
ζ	local coordinate axis
θ	angle
ν	Poisson's ratio
σ	stress
σ_3	principal stress
τ	shear stress
Σ	summation
$\sqrt{\quad}$	square root
\ln	natural (Naperian) logarithm (base e)
$\dot{\mathcal{E}}$	Strain energy release rate
ξ	coordinate distance from crack tip
σ_e	stress applied to crack tip

APPENDIX B: DESCRIPTION OF PROGRAM UNITS

Finite Element Fracture Program

A flow diagram of subroutines is shown in Figure B-1. The following is a brief description of each subroutine and the input data format follows.

DATAIN:

Reads and echo prints all data pertaining to the conventional elements. It also performs checks on this data.

ASEMBL;

Initializes and assembles the global stiffness matrix (including the crack element) and the global load vector. Modifies the above to reflect the geometric boundary conditions.

QUAD:

Computes the stress-strain matrix, stiffness matrix, body force vector, and strain-displacement matrix of either a 4-CST quadrilateral or a triangular element.

CST:

Computes the strain-displacement matrix, stiffness matrix, and body force vector of a CST element.

GEOMBC:

Applies prescribed displacement boundary conditions at a single, specified node.

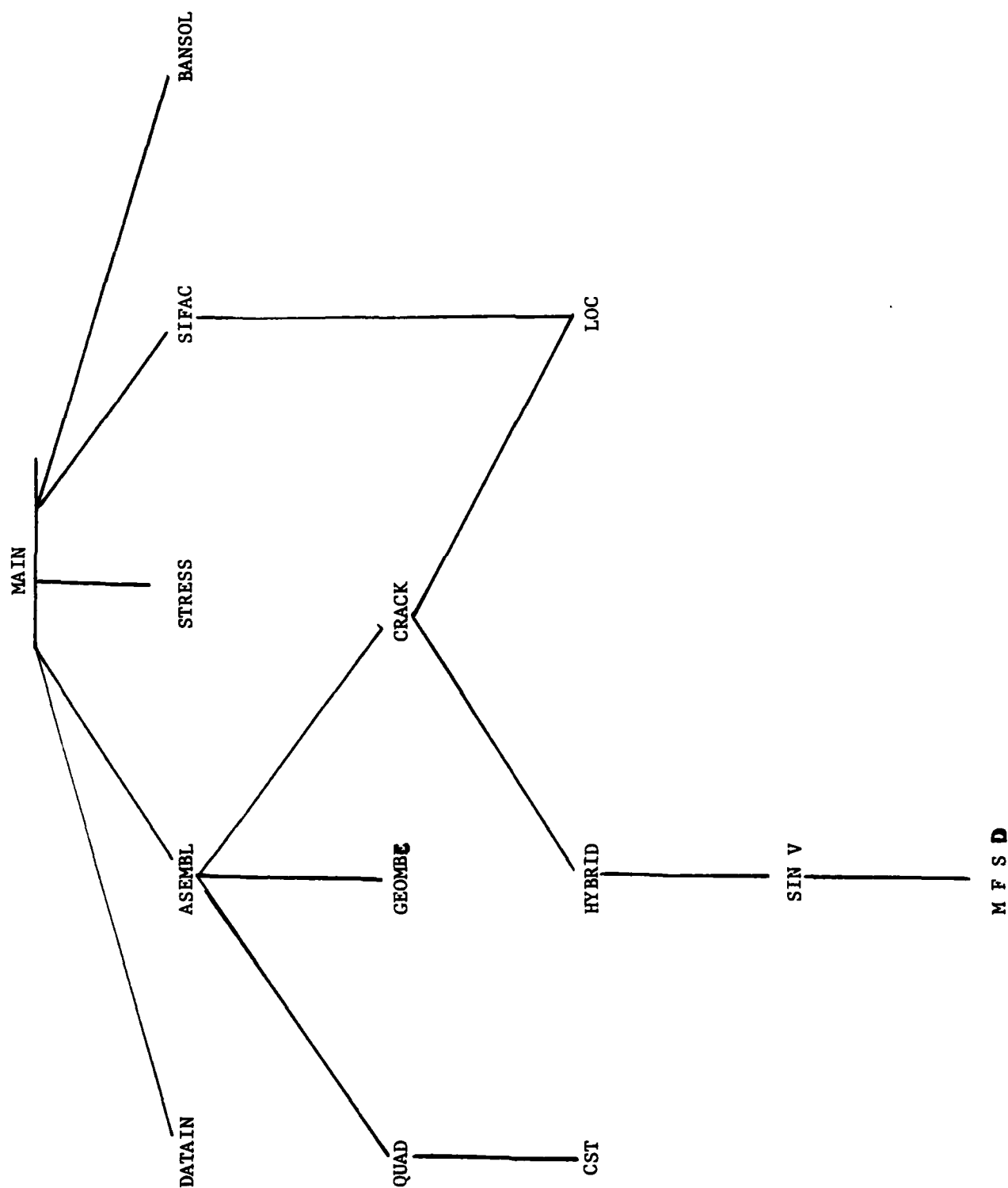


Figure B-1. Flow Diagram of Program for Finite Element Fracture Calculations

CRACK:

Reads the input data pertaining to the crack element. Computes the constants and of Section II. Incorporates the crack element stiffness matrix into the global stiffness matrix which had previously contained stiffnesses from only conventional elements.

HYBRID:

Computes the matrices $[G]$ and $[H]$ of Section II. Computes the crack element stiffness matrix. Computes the $(B_I)^T$ and $(B_{II})^T$ row vectors of Section II.

LOC:

The crack element stiffness matrix is not derived in two-dimensional array form but rather in vector form. Thus, in CRACK and SIFAC where this stiffness matrix is used in calculations and manipulations, some operator must be used to translate from two- to one-dimensional form. This operator is LOC. This is probably done to save memory.

MFSD:

Uses numerical methods to perform the integration implied by equations (16) and (17) of Section II.

SINV:

Performs the inversion of $[H]$.

BANSOL:

Triangularizes the global banded stiffness matrix by symmetric Gauss-Doolittle decomposition and/or solves for the global displacement vector corresponding to a given load vector

STRESS:

Computes strains, stresses, and principal stresses for conventional elements. Computes the strain energy due to all the conventional elements which can be used to calculate stress intensity factors by energy methods, as mentioned in Section I. Prints stresses and principal stresses at element centroids, and the strain energy.

SIFAC:

Computes the stress intensity factors by equations (23) in Section II. Computes the crack element strain energy and the global strain energy, which can be used to compute stress intensity factors by energy methods, as in Section I. Prints the crack element strain energy.

Input Guide- Descriptions are included for each card.

IDENTIFICATION CARD:

One card per problem: Format (I5,3x,9A8)

cc 1-5 NPROB: The problem number. If NPROB = 0, execution of the program is halted.

cc 9-80 TITLE(I): Title of the problem. The vector has 9 elements, each of which contains 9 characters of the title.

Multiple problems can be handled, but as soon as the next problem is read the previous problem is lost, so a problem cannot be recalled.

BASIC PARAMETERS:

One card per problem: Format 615

cc 1-5 NNP: Number of nodal points

cc 6-10 NEL: Number of conventional elements

cc 11-15 NMAT: Number of different materials

cc 16-20 NLSC: Number of surface tractions

cc 21-25 NOPT: Option for stress state. 1 = plane strain, 2 = plane stress.

cc 26-30 NBODY: Option for body force. 0 = no weight, 1 = weight in the negative y direction.

cc 31-35 NCKEL: Number of crack elements.

MATERIAL PROPERTIES

NMAT cards per problem: Format 4F10.0

cc 1-10 E: Modulus of elasticity

cc 11-20 PR: Poisson's ratio

cc 21-30 RO: Density of the material

cc 31-40 TH: Thickness of the material

NODAL POINT DATA:

Up to NNP cards per problem: Format (215,4F10.0)

cc 1-5 M: Nodal point number

cc 6-10 KODE(I): Index of displacement and concentrated load conditions at node I. The values Kode can assume and the conditions assigned to each are given in Table IV-1.

cc 11-20 X: Horizontal coordinate of node I.

cc 31-30 Y: Vertical coordinate of node I.

cc 31-40 ULX: Concentrated load or displacement in X direction at node
I.

cc 41-50 ULY: Concentrated load or displacement in Y direction at node
I.

Usually one card is needed for each node. However, if some nodes fall on a straight line and are equidistant, data for only the first and last points of this group are needed. Intermediate nodal point data are automatically generated by linear interpolation. The nodal data must be entered in order from smallest to largest, leaving out those nodes which are to be interpolated. The nodes which are interpolated are assigned values of $KODE = 0$, $ULX = 0$, and $ULY = 0$. The signs of prescribed nodal displacements or forces follow the signs of the coordinate directions assigned by the user.

ELEMENT DATA:

Up to NEL cards per problem: Format 615.

cc 1-5 M: The element number

cc 6-10 IE(M,1): The index of the first node of a CST or 4-CST
quadrilateral element.

cc 11-15 IE(M,2): The index of the second node of a CST or 4-CST
quadrilateral element.

cc 16-20 IE(M,3): The index of the third node in a CST or 4-CST
quadrilateral element.

cc 21-25 IE(M,4): The index of the fourth node in a CST or 4-CST
quadrilateral element. If it is a CST element, the
index of fourth node equals that of the third node;
that is, $IE(M,3) = IE(M,4)$.

cc 26-30 IE(M,5): Material type number corresponding to element M.

Usually, one card is needed for each element. However, if some elements are on a line in such a way that their corner node indexes each increase by one compared to the previous element, only the data for the first element on the line need be input. As the elements on the line are generated by adding one to each node of the preceeding element, starting with the first element on the line, the last element on a line needn't be input. However, data for the last element in the assemblage must be input whether it could be generated or not. Also, please note that triangular elements cannot be generated from quadrilateral elements because the third and fourth indexes of a triangular element are equal. The same material type as the first element on a line is assigned to all elements generated on that line.

For a right-handed coordinate system, the nodal indices for an element must be input counter-clockwise around the element.

SURFACE TRACTIONS:

As many cards as needed: Format (2I5,4F10.0)

cc 1-5	ISC:	I, the first node upon which tractions act.
cc 6-10	JSC:	J, the second node upon which tractions act.
cc 11-20	SURX1:	The intensity of the traction in the X direction acting at node I.
cc 21-30	SURX2:	The intensity of the traction in the X direction acting at node J.
cc 31-40	SURY1:	The intensity of the traction in the Y direction acting at node I.
cc 41-50	SURY2:	The intensity of the traction in the Y direction acting at node J.

Surface tractions must be specified between two adjacent nodes only. The tractions in both the X and Y directions are assumed to vary linearly between the two nodes. Intensities are expressed in units of force/length so that pressures must be multiplied by the thickness before being input into the computer. The signs of the tractions follow the directions of the coordinate axes assigned by the user.

CRACK ELEMENT DATA:

There are NCKEL cards per problem for each of the two types of card which comprise the crack element data.

Card one: Format (215,2F10.0,15)

cc 1-5 KEY: The type of crack element. 1 = five node case, 2 =
nine node case.

cc 6-10 MATYP: The type of material where the crack element is placed.

cc 11-20 XC: The horizontal coordinate of the crack tip.

cc 21-30 YC: The vertical coordinate of the crack tip.

```
cc 31-35  NCOT:      Flag which determines which direction nodal indexes
                    are to be counted around the crack element.  For the
                    five node case 1 = clockwise, 0 = counter-clockwise.
```

Card two: Format 1015

```
cc (5I-4)-5I  KCRK(I):  Nodal incidence for the Ith node of the crack
                    element.
```

cc (5K+1)-5(K+1) MAXDIF: Maximum difference between the nodal incidences
of the crack element. Used to compute the
bandwidth after the addition of the crack
element.

For the five node case there are five incidences input for the crack element ($K = 5$) and for the nine node case there are nine ($K = 9$). In either case the input for MAXDIF immediately follows the last incidence input. All crack element data is omitted if NCKEL = 0.

The listing of the finite element computer code is given on the following pages.

```

1 C      PROGRAM FEQD(INPUT,OUTPUT,TAPE3=INPUT,TAPE6=OUTPUT,
2 C      +TEMP1,TEMP2,TEMP3,TEMP4,TEMP5,
3 C      +TAPE1=TEMP1,TAPE2=TEMP2,TAPE3=TEMP3,TAPE4=TEMP4,TAPE5=TEMP5)
4 C      IMPLICIT REAL*8(A-H,O-Z)
5 C      PROGRAM FEQD
6 C      DIMENSION TITLE(8)
7 C      COMMON/CONS/NNP,NEL,NMAT,NSLC,NOPT,NBODY,MTYP,NCKEL
8 C      COMMON/ONE/QK(10,10),Q(10),B(3,10),C(3,3),BT(3,6),XQ(5),YQ(5)
9 C      COMMON/TWO/IBAND,NEQ,R(1400),AK(1400,100)
10 C      COMMON/TH/IE(700,5)
11 C      DATA MAXEL,MAXNP,MAXMAT,MAXBW/700,700,10,100/
12 C      DATA MAXDOF/2*MAXNP/
13 C      DATA MAXDOF/2*MAXNP/
14 C      DATA MAXDOF/2*MAXNP/
15 C      DATA MAXDOF/2*MAXNP/
16 C      DATA MAXDOF/2*MAXNP/
17 C      DATA MAXDOF/2*MAXNP/
18 C      DATA MAXDOF/2*MAXNP/
19 C      DATA MAXDOF/2*MAXNP/
20 C      DATA MAXDOF/2*MAXNP/
21 C      DATA MAXDOF/2*MAXNP/
22 C      DATA MAXDOF/2*MAXNP/
23 C      DATA MAXDOF/2*MAXNP/
24 C      DATA MAXDOF/2*MAXNP/
25 C      DATA MAXDOF/2*MAXNP/
26 C      DATA MAXDOF/2*MAXNP/
27 C      DATA MAXDOF/2*MAXNP/
28 C      DATA MAXDOF/2*MAXNP/
29 C      DATA MAXDOF/2*MAXNP/
30 C      DATA MAXDOF/2*MAXNP/
31 C      DATA MAXDOF/2*MAXNP/
32 C      DATA MAXDOF/2*MAXNP/
33 C      DATA MAXDOF/2*MAXNP/
34 C      DATA MAXDOF/2*MAXNP/
35 C      DATA MAXDOF/2*MAXNP/
36 C      DATA MAXDOF/2*MAXNP/
37 C      DATA MAXDOF/2*MAXNP/
38 C      DATA MAXDOF/2*MAXNP/
39 C      DATA MAXDOF/2*MAXNP/
40 C      DATA MAXDOF/2*MAXNP/
41 C      DATA MAXDOF/2*MAXNP/
42 C      DATA MAXDOF/2*MAXNP/
43 C      DATA MAXDOF/2*MAXNP/
44 C      DATA MAXDOF/2*MAXNP/
45 C      DATA MAXDOF/2*MAXNP/
46 C      DATA MAXDOF/2*MAXNP/
47 C      DATA MAXDOF/2*MAXNP/
48 C      DATA MAXDOF/2*MAXNP/
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51 C      DATA MAXDOF/2*MAXNP/
52 C      DATA MAXDOF/2*MAXNP/
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54 C      DATA MAXDOF/2*MAXNP/
55 C      DATA MAXDOF/2*MAXNP/
56 C      DATA MAXDOF/2*MAXNP/
57 C      DATA MAXDOF/2*MAXNP/
58 C      DATA MAXDOF/2*MAXNP/
59 C      DATA MAXDOF/2*MAXNP/
60 C      DATA MAXDOF/2*MAXNP/

```

```

61      GOTO 9999
62      WRITE(16,901) IBAND,MAXBW
63      GOTO 9999
64      FORMAT(15,3X,9A8)
65      200 FORMAT(/"PROBLEM",15,"",9A8/)
66      300 FORMAT(/"OUTPUT TABLE 1.. NODAL DISPLACEMENTS"//13X,"NODE",9X,
67      + "U = X-DISP",9X,"V = Y-DISP",/15X,112,2E20.8))
68      + "U = X-DISP",9X,"V = Y-DISP",/15X,112,2E20.8))
69      901 FORMAT(/"BANDWIDTH =",14," EXCEEDS MAX, ALLOWABLE =",14//
70      + " GO ON TO NEXT PROBLEM")
71      STOP
72      END
73 C      SUBROUTINE DATIN(MAXEL,MAXNP,MAXMAT,MAXSLC,ISTOP)
74 C      IMPLICIT REAL*8(A-H,O-Z)
75 C      COMMON/CONS/NNP,NEL,NMAT,NSLC,NOPT,NBODY,MTYP,NCKEL
76 C      COMMON/ONE/QK(10,10),Q(10),B(3,10),C(3,3),BT(3,6),XQ(5),YQ(5)
77 C      COMMON/TWO/IBAND,NEQ,R(1400),AK(1400,100)
78 C      COMMON/TH/IE(700,5)
79 C      DATA MAXEL,MAXNP,MAXMAT,MAXBW/700,700,10,100/
80 C      DATA MAXDOF/2*MAXNP/
81 C      DATA MAXDOF/2*MAXNP/
82 C      DATA MAXDOF/2*MAXNP/
83 C      DATA MAXDOF/2*MAXNP/
84 C      DATA MAXDOF/2*MAXNP/
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113 C      DATA MAXDOF/2*MAXNP/
114 C      DATA MAXDOF/2*MAXNP/
115 C      DATA MAXDOF/2*MAXNP/
116 C      DATA MAXDOF/2*MAXNP/
117 C      DATA MAXDOF/2*MAXNP/
118 C      DATA MAXDOF/2*MAXNP/
119 C      DATA MAXDOF/2*MAXNP/
120 C      DATA MAXDOF/2*MAXNP/

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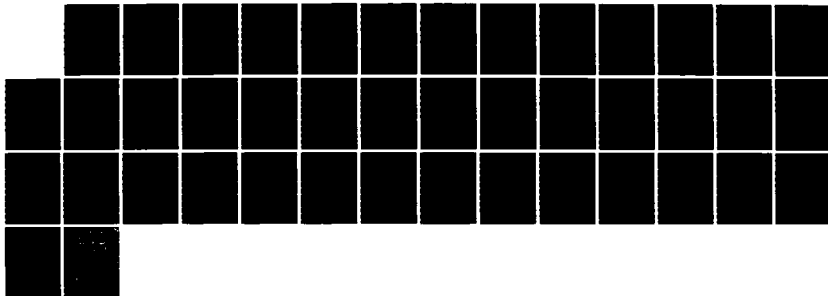
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TRANSPORTATION INST COLLEGE STATION D N LITTLE ET AL.
31 DEC 85 AFOSR-TR-86-0242-VOL-2 F49620-82-K-0027

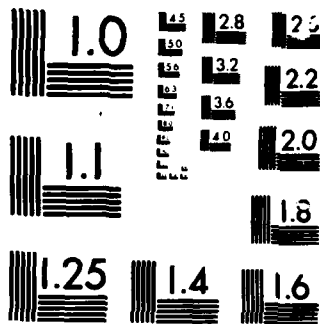
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NL





MICROCOPY

CHART

```

121 Y(N)=Y(N-1)+RY
122 ULX=0.0
123 VLY=0.0
124 GOTO 6
125
126 442 ULX=ULX
127 VLY=VLY
128 KODE=KCODE
129 WRITE(16,52)N,KODE,X(N),Y(N),ULX,VLY
130 WRITE(14)N,KODE,ULX,VLY
131 N=N+1
132 IF(N-N9,442,6)
133 IF(N-LE.NNP)GOTO5
134 9 IF(N-LE.NNP)GOTO5
135 C
136 READ AND PRINT ELEMENT PROPERTIES, TABLE 6-4
137 WRITE(16,106)
138 L=0
139 14 READ(15,15)M,(IE(M,1),I=1,5)
140 L=L+1
141 IF(N-L)117,17,18
142 117 WRITE(16,118)M
143 WRITE(16,53)M,(IE(M,1),I=1,5)
144 ISTOP=ISTOP+1
145 GOTO 14
146 18 IE(L,1)=IE(L-1,1)+1
147 IE(L,2)=IE(L-1,2)+1
148 IE(L,3)=IE(L-1,3)+1
149 IE(L,4)=IE(L-1,4)+1
150 IE(L,5)=IE(L-1,5)
151 IF(N-L,20,20,16)
152 IF(N-LE,21,21,14)
153 21 CONTINUE
154 C
155 READ AND PRINT SURFACE LOADING (TRACTION) CARDS
156 IF(NSLC.EQ.0)GOTO31
157 DO 40L=1,NSLC
158 WRITE(16,108)
159 READ(15,41)JSC,JSC,SURX1,SURX2,SURY1,SURY2
160 WRITE(16,42)JSC,JSC,SURX1,SURX2,SURY1,SURY2
161 40 IF(ISTOP.EQ.0)GOTO999
162 WRITE(16,900)ISTOP
163 C
164 1 FORMAT(7I5)
165 100 FORMAT('INPUT TABLE 1.. BASIC PARAMETERS',15X,
166 '* NUMBER OF NODAL POINTS',15X,
167 '* NUMBER OF ELEMENTS',15X,
168 '* NUMBER OF DIFFERENT MATERIALS',15X,
169 '* NUMBER OF SURFACE LOAD CARDS',15X,
170 '* 1 = PLANE STRAIN, 2 = PLANE STRESS',15X,
171 '* BODY FORCES(1 = IN - Y DIRECTION, 0 = NONE)',15X,
172 200 FORMAT('0',6X,'NUMBER OF CRACK ELEMENTS',15X)
173 251 FORMAT('0',6X,'TOO MANY NODAL POINTS, MAXIMUM =',15)
174 252 FORMAT('0',6X,'TOO MANY ELEMENTS, MAXIMUM =',15)
175 253 FORMAT('0',6X,'TOO MANY MATERIALS, MAXIMUM =',15)
176 255 FORMAT('0',6X,'EXECUTION HALTED BECAUSE OF',15,' FATAL ERRORS')
177 2 FORMAT(4F10.0)
178 101 FORMAT('INPUT TABLE 2.. MATERIAL PROPERTIES',15X,
179 '* MODULUS OF',6X,'POISSON',6X,'RATIO',6X,'DENSITY',6X,
180 '* NUMBER',6X,'ELASTICITY',6X,'RATIO',6X,'DENSITY',6X,
181 '* THICKNESS')
182 51 FORMAT(110,4E15.4)
183 103 FORMAT('INPUT TABLE 3.. NODAL POINT DATA',15X,'NODAL',48X,
184 '* X-DISP.',6X,'Y-DISP.',15X,'POINT',6X,'TYPE',14X,'X',14X,'Y',6X,
185 '* OR LOAD',6X,'OR LOAD')
186 3 FORMAT(2I5,4F10.0)
187 105 FORMAT(5X,'ERROR IN CARD NO.',15X)
188 52 FORMAT(2I10,4E15.4)
189 106 FORMAT('INPUT TABLE 4.. ELEMENT DATA',15X,
190 '* GLOBAL INDICES OF ELEMENT NODES',3X,'ELEMENT',7X,'1',7X,'2',7X,
191 '* 3',7X,'4',2X,'MATERIAL')
192 118 FORMAT(5X,'ERROR IN ELEMENT CARD NO.',15X)
193 15 FORMAT(6I5)
194 53 FORMAT(110,4I19,110)
195 108 FORMAT('INPUT TABLE 5.. SURFACE LOADING DATA',15X,
196 '* SURFACE LOAD INTENSITIES AT NODES',74X,'NODE 1',4X,'NODE J',10X,
197 '* X1',10X,'XJ',10X,'Y1',10X,'YJ')
198 41 FORMAT(2I5,4F10.0)
199 42 FORMAT(2I10,4E12.4)
200 900 FORMAT('')
201 999 RETURN
202
203 SUBROUTINE ASEMBL(ISTOP)
204 IMPLICIT REAL*8(A-H,O-Z)
205 DIMENSION LP(8)
206
207 C
208 COMMON/CONS/NNP,NEL,NMAT,NSLC,NOPT,NBDDY,MTYP,NCKEL
209 COMMON/CRACK/PR(10),RG(10),TH(10),X(700),Y(700)
210 COMMON/ONE/OK(10),Q(10),B(3,10),C(3,3),BT(3,6),XG(5),YG(5)
211 COMMON/TWO/IBAND,NEG,R(1400),AK(1400,100)
212 COMMON/THREE/IE(700,5)
213 REVIND 11
214 REVIND 12
215 REVIND 13
216 REVIND 14
217 REVIND 15
218 C
219 INITIALIZE
220 ISTOP=0
221 C
222 INITIALIZE PARTS OF MATRICES C AND BT
223 BT(1,4)=0.0
224 BT(1,5)=0.0
225 BT(2,1)=0.0
226 BT(2,2)=0.0
227 BT(2,3)=0.0
228 C(1,3)=0.0
229 C(2,3)=0.0
230 C(3,1)=0.0
231 C(3,2)=0.0
232 C
233 INITIALIZE OVERALL STIFFNESS MATRIX AK AND OVERALL LOAD VECTOR R
234 DO 21=1,NEG
235 R(1)=0.0
236 DO 2J=1,IBAND
237 2 AK(1,J)=0.0
238 C
239 C COMPUTE ELEMENT STIFFNESSES AND LOADS ONE BY ONE
240 C

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241 DO 10M=1,NEL
242 IF(IE(M,5).GT.0)GOTO11
243 ISTOP=ISTOP+1
244 GOTO 10
245 11 CALL QUAD(M,AREA)
246 IF(AREA.GT.0)GOTO16
247 ISTOP=ISTOP+1
248 WRITE(16,20)M
249 C
250 C
251 C STORE ELEMENT STIFFNESS MATRIX TO COMPUTE STORED ENERGY
252 16 IF(IE(M,3).EQ.1E(M,4))GOTO201
253 LIM=10
254 GOTO 205
255 201 LIM=6
256 205 WRITE(13)LIM,((OK(I,J),J=1,LIM),I=1,LIM)
257 C
258 C CONDENSE ELEMENT STIFF. FROM 10X10 TO 9X9, EQ. (5-64), AND ELEMENT
259 C LOADS FROM 10X1 TO 9X1, EQ. (5-64D). (REF. 2)
260 IF(IE(M,3).EQ.1E(M,4))GOTO26
261 DO 31J=1,2
262 IJ=10-J
263 IK=1-J+1
264 PIVOT=OK(IK,IK)
265 DO 32K=1,IJ
266 P=OK(IK,K)/PIVOT
267 OK(IK,K)=P
268 DO 33I=K,IJ
269 OK(I,K)=OK(I,K)-P*OK(I,IK)
270 33 OK(K,I)=OK(I,K)
271 32 OK(K)=OK(K)-OK(IK,K)*OK(IK)
272 31 Q(IK)=Q(IK)/PIVOT
273 C
274 C STORE MULTIPLIERS,PIVOTS,CONDENSED LOADS, STRAIN-DISP. AND STRESS-STR
275 C MATRICES ON SCRATCH TAPE NO. 1 (TO BE USED LATER TO COMPUTE STRAINS A
276 C STRESSES )
277 26 WRITE(11)((OK(I,J),J=1,IJ),I=9,10),Q(I),((B(I,J),J=1,IJ),I=
278 9,1),((C(I,J),J=1,3),I=1,3),XQ(5),YQ(5)
279 C
280 C ASSEMBLE STIFF. AND LOADS , DIRECT STIFF, METHOD, SEC, 6-5.
281 C
282 LIM=6
283 IF(IE(M,3).EQ.1E(M,4))LIM=6
284 DO 40I=2,LIM,2
285 IJ=I/2
286 LP(I-1)=2*IE(M,IJ)-1
287 LP(I)=2*IE(M,IJ)
288 DO 50LL=1,LIM
289 I=LP(LL)
290 R(I)=R(I)+Q(LL)
291 DO 50MM=1,LIM
292 J=LP(MM)-1+1
293 J=LP(MM)+1+1
294 IF(J.LE.0)GOTO50
295 AK(I,J)=AK(I,J)+OK(LL,MM)
296 50 CONTINUE
297 10 CONTINUE
298 IF(NCKEL.EQ.0)GOTO35
299 DO 14I=1,NCKEL
300 14 CALL CRACK
301 C

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301 C
302 C ADD EXTERNALLY APPL., CONC, MODAL LOADS TO R
303 DO 55N=1,NMP
304 READ(14)M,KODE,ULX,VLY
305 IF(KODE.EQ.3)GOTO55
306 K=2*N
307 IF(KODE.EQ.1)GOTO57
308 R(K-1)=R(K-1)+ULX
309 IF(KODE.NE.0)GOTO55
310 57 R(K)=R(K)+VLY
311 55 CONTINUE
312 C
313 C CONVERT LINEARLY VARYING SURFACE TRACTION TO STATIC EQUIVALENTS,
314 C AND ADD TO OVERALL LOAD VECTOR R, EQ. (5-81A).
315 IF(NSLC.EQ.0)GOTO60
316 DO 61L=1,NSLC
317 READ(18)ISC,JSC,SURX1,SURX2,SURY1,SURY2
318 J=JSC
319 11=2+1
320 11=2+J
321 J=2+J
322 DX=X(J)-X(1)
323 DY=Y(J)-Y(1)
324 EL=SQRT(DX*DX+DY*DY**2)
325 PXI=SURX1*EL
326 PYI=SURX2*EL
327 PYJ=SURY1*EL
328 PYJ=SURY2*EL
329 R(11-1)=R(11-1)+PXI/3.0+PYJ/6.0
330 R(11-1)=R(11-1)+PXI/6.0+PYJ/3.0
331 R(11)=R(11)+PYI/3.0+PYJ/6.0
332 R(JJ)=R(JJ)+PYI/6.0+PYJ/3.0
333 61 CONTINUE
334 C
335 C INTRODUCE KINEMATIC CONSTRAINTS (GEOMETRIC BOUNDARY CONDITIONS),
336 C EQ. (6-16),
337 C REF. 1.
338 60 CONTINUE
339 REWIND 14
340 DO 70M=1,NMP
341 READ(14)N,KODE,ULX,VLY
342 IF(KODE.GE.0.AND.KODE.LE.3)GOTO72
343 ISTOP=ISTOP+1
344 GOTO 70
345 72 IF(KODE.EQ.0)GOTO70
346 IF(KODE.EQ.2)GOTO71
347 CALL GEOMBC(ULX,2*M-1)
348 IF(KODE.EQ.1)GOTO70
349 71 CALL GEOMBC(VLY,2*M)
350 70 CONTINUE
351 ENDFILE11
352 ENDFILE12
353 ENDFILE13
354 ENDFILE14
355 ENDFILE16
356 IF(ISTOP.EQ.0)GOTO61
357 WRITE(16,100)ISTOP
358 20 FORMAT(//5X," AREA OF ELEMENT ",IS," IS NEGATIVE "//)
359 100 FORMAT(//5X," SOLUTION WILL NOT BE PERFORMED BECAUSE OF ",IS,
360 4," DATA ERRORS "//)

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361 61 RETURN
362 ENO
363 SUBROUTINE QUAD(M,TOTALA)
364 C IMPLICIT REAL*8(A-H,O-Z)
365 COMMON/CONS/NNP,NEL,NMAT,NSLC,NOPT,NBDDY,MTYP,NCKEL
366 COMMON/ONE/OK(10,10),G(10),B(3,3),BT(3,6),XQ(S),YQ(S)
367 COMMON/TWO/IBAND,NEQ,R(1400),AK(1400,100)
368 COMMON/TH/IE(700,5)
369 C
370 C I=IE(M,1)
371 C J=IE(M,2)
372 C K=IE(M,3)
373 C L=IE(M,4)
374 C MTYP=IE(M,5)
375 C TOTALA=0.0
376 C
377 C
378 C CONSTRUCT STRESS-STRAIN MATRIX C.EQ.(3-16C). FOR PLANE STRAIN
379 C NOPT=1 AND FOR PLANE STRESS NOPT=2, PRESENT CODE IS FOR
380 C ISOTROPIC MATERIALS
381 C IF(NMAT.EQ.1.AND.M.ST.1)GOTO5
382 C IF(NOPT.EQ.2)GOTO2
383 C CF=E(MTYP)/(1.0-PR(MTYP))=(1.0-2.0*PR(MTYP))
384 C G(1,1)=CF*(1.0-PR(MTYP))
385 C G(1,2)=CF*PR(MTYP)
386 C G(2,1)=G(1,2)
387 C G(2,2)=G(1,1)
388 C G(3,3)=CF*(1.0-2.0*PR(MTYP))/2.0
389 C G(3,4)=CF*(1.0-2.0*PR(MTYP))/2.0
390 C GOTO 5
391 C 2 CF=E(MTYP)/(1.0-PR(MTYP))=PR(MTYP)
392 C G(1,1)=CF
393 C G(1,2)=PR(MTYP)=CF
394 C G(2,1)=G(1,2)
395 C G(2,2)=CF
396 C G(3,3)=CF*(1.0-PR(MTYP))/2.0
397 C LHM=4
398 C IF(K.EQ.L)LHM=3
399 C XQ(S)=0.0
400 C YQ(S)=0.0
401 C DO 10N=1,LHM
402 C NN=IE(M,N)
403 C XQ(N)=X(NN)
404 C YQ(N)=Y(NN)
405 C XQ(S)=XQ(S)+X(NN)/FLOAT(LHM)
406 C YQ(S)=YQ(S)+Y(NN)/FLOAT(LHM)
407 C INITIALIZE QUAD. STIFFNESS,LOAD VECTOR AND STRAIN-DISPLACEMENT VECTOR
408 C DO 1311=1,10
409 C Q(11)=0.0
410 C DO 12JJ=1,10
411 C OK(11,JJ)=0.0
412 C DO 13JJ=1,3
413 C B(JJ,11)=0.0
414 C B(JJ,11)=0.0
415 C IF(K.NE.L)GOTO15
416 C CALL CST(1,2,3,TOTALA)
417 C GOTO 999
418 C 15 CALL CST(1,2,5,AREA)
419 C TOTALA=TOTALA+AREA
420 C CALL CST(2,3,5,AREA)

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421 TOTALA=TOTALA+AREA
422 CALL CST(3,4,5,AREA)
423 TOTALA=TOTALA+AREA
424 CALL CST(4,1,5,AREA)
425 TOTALA=TOTALA+AREA
426 RETURN
427 ENO
428 C SUBROUTINE CST(I,J,K,AREA)
429 C IMPLICIT REAL*8(A-H,O-Z)
430 C IMPLICIT REAL(A-H,O-Z)
431 C DIMENSION CB(3,6),LC(6),LT(3),TK(6,6)
432 C
433 C COMMON/CONS/NNP,NEL,NMAT,NSLC,NOPT,NBDDY,MTYP,NCKEL
434 C COMMON/ONE/OK(10,10),G(10),TH(10),X(700),Y(700)
435 C COMMON/ONE/OK(10,10),G(10),B(3,3),BT(3,6),XQ(S),YQ(S)
436 C COMMON/TWO/IBAND,NEQ,R(1400),AK(1400,100)
437 C LT(1)=1
438 C LT(2)=J
439 C LT(3)=K
440 C COMPUTE STRAIN-DISPLACEMENT MATRIX B FOR TRIANGLE, EQ. (5-35A)
441 C BT(1,1)=YQ(J)-YQ(K)
442 C BT(1,2)=YQ(K)-YQ(I)
443 C BT(1,3)=YQ(I)-YQ(J)
444 C BT(2,1)=XQ(K)-XQ(J)
445 C BT(2,2)=XQ(I)-XQ(K)
446 C BT(2,3)=XQ(J)-XQ(I)
447 C BT(3,1)=BT(2,4)
448 C BT(3,2)=BT(2,5)
449 C BT(3,3)=BT(2,6)
450 C BT(3,4)=BT(1,1)
451 C BT(3,5)=BT(1,2)
452 C BT(3,6)=BT(1,3)
453 C AREA=(BT(2,4)*BT(1,3)-BT(2,6)*BT(1,1))/2.0
454 C
455 C COMPUTE C=B
456 C DO 1011=1,3
457 C DO 10JJ=1,6
458 C CB(11,JJ)=0.0
459 C DO 10KK=1,3
460 C 10 CB(11,JJ)=CB(11,JJ)+C(I,K)*BT(KK,JJ)
461 C
462 C COMPUTE (B*B)=C*B, EQ.(5-45A)
463 C DO 1211=1,6
464 C DO 12JJ=1,6
465 C TK(11,JJ)=0.0
466 C DO 12KK=1,3
467 C 12 TK(11,JJ)=TK(11,JJ)+BT(KK,11)*CB(KK,JJ)
468 C
469 C ADD TRIANGLE STIFFNESS TO QUADRILATERAL STIFFNESS, EX.(6-2),
470 C ADD TRIANGLE STRAIN-DISPLACEMENT MATRIX TO QUADRILATERAL STRAIN-
471 C DISPLACEMENT MATRIX
472 C DO 1511=1,3
473 C LC(11)=2*LT(11)-1
474 C 15 LC(11+3)=2*LT(11)
475 C DO 3011=1,6
476 C LL=LC(11)
477 C FK=1.0/(4.0*AREA)
478 C FB=2.0*FK
479 C DO 20JJ=1,6
480 C MM=LC(JJ)

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481 20 OK(LL,MM)=OK(LL,MM)+TK(11,JJ)+TH(MTYP)*FK
482 DO 30JJ=1,3
483 30 B(JJ,LL)=B(JJ,LL)+ST(JJ,11)*FB
484 C
485 C DEVELOP BODY FORCE VECTOR, EQ.(5-61B)
486 IF(NBODY.EQ.0)GOTO988
487 TBODYF=AREA*RG(MTYP)*TH(MTYP)
488 BODYF=TBODYF/3.0
489 DO 3511=1,3
490 JJ=2*LT(11)
491 35 Q(JJ)=Q(JJ)+BODYF
492 998 RETURN
493
494 C SUBROUTINE CRACK
495 C IMPLICIT REAL*8 (A-H,O-Z)
496 COMMON/CONS/NMP,NEL,NMAT,NSLC,NOPT,NBODY,MTYP,NCKEL
497 COMMON/THO/IBAND,NEG,RO(10),TH(10),X(700),Y(700)
498 COMMON/TWO/IBAND,NEG,R(1400),AK(1400,100)
499 COMMON/T3/BCR(2,18),EK(171),XXY(18),KCRK(9),LP(18)
500 READ(15,12)KEY,MATYP,XC,YC,NCONT
501 12 FORMAT(2I5,2F10.0,1B)
502 IF(KEY.EQ.1)GOTO10
503
504 C NOE=9
505 NOE=18
506
507 10 NOE=5
508
509 20 READ(15,2)(KCRK(11),I=1,NODE),MAXDIF
510 2 FORMAT(10I5)
511 K=2*(MAXDIF+1)
512 IF(K.LE.IBAND)GOTO91
513 L=IBAND+1
514 DO 100J=L,K
515 DO 100J=L,K
516 AK(1,J)=0.
517 IBAND=MAX0(IBAND,K)
518
519 91 CONTINUE
520 DO 30I=1,NODE
521 XXY(2+I-1)=X(KCRK(11))
522 XXY(2+I)=Y(KCRK(11))
523 DO 31I=1,NOE
524 WRITE(16,32)I,XXY(1)
525 32 FORMAT('0',5X,'-XXY(',13,')=',020.6)
526 SHU=E(MATYP)/(2.*(1.+PR(MATYP)))
527 IF(NOPT.EQ.1)GOTO40
528 ETA=(3.-PR(MATYP))/(1.+PR(MATYP))
529 GOTO 50
530
531 40 ETA=3.-4.*PR(MATYP)
532
533 50 CONTINUE
534 WRITE(16,35)SHU,ETA
535 35 FORMAT('0',5X,'-SHU,ETA =',2G20.6)
536 CALL HYBRID(KEY,SHU,ETA,XC,YC,NCONT)
537 DO 60I=1,NODE
538 LP(2+I-1)=2*KCRK(11)-1
539 LP(2+I)=2*KCRK(11)
540 DO 36I=1,NOE
541 WRITE(16,33)I,LP(1)
542 33 FORMAT('0',5X,'-LP(',13,')=',15)
543 DO 70LL=1,NOE
544
545 C
546 70 CONTINUE
547
548 84 FORMAT('0',5X,'-EK(',14,')=',020.6,'-AK(',13,')=',020.6)
549
550 C
551 34 FORMAT('0',5X,'-NODE NOE MATYP XC,YC =',3I4,20I5.3)
552 KK=(NOE+1)*NOE/2
553 WRITE(12)KK,NODE,KEY,LP(11),I=1,NOE),EK(11),I=1,NOE),
554 (J),I=1,KEY),J=1,NOE),XC,YC,MATYP,(KCRK(11),I=1,NOE)
555 RETURN
556
557 C SUBROUTINE LOC(I,J,IR,N,M,MS)
558 I=1
559 JX=J
560 IF(MS-1)10,20,30
561 10 IR=N*(JX-1)+IX
562 GOTO 36
563 20 IF(IX-JX)22,24,24
564 22 IR=IX*(JX-JX-IX)/2
565 GOTO 36
566 24 IR=JX*(IX-IX-IX)/2
567 GOTO 36
568 30 IR=0
569 IF(IX-JX)36,32,36
570 32 IR=IX
571 36 IR=IRX
572 RETURN
573
574 C SUBROUTINE HYBRID(KEY,SHU,ETA,XC,YC,NCONT)
575 C IMPLICIT REAL*8(A,B,D-H,O-Y),COMPLEX*16(Z,C)
576 IMPLICIT REAL(A,B,D-H,O-Y),COMPLEX(Z,C)
577 COMPLEX SINT,SJNT
578 COMPLEX=16 ZET,Z,DCMPLX,DCONJ0,F1,F2,F3,FF2(32),FF3(32),XY
579 COMPLEX ZET,Z,DCMPLX,DCONJ0,F1,F2,F3,FF2(32),FF3(32),XY
580 COMPLEX=16 ZETK,CZK,ZETT,CZZ,ZET4,ZE,ZD,ZC,ZA,ZB,C1,CDSORT
581 COMPLEX ZETK,CZK,ZETT,CZZ,ZET4,ZE,ZD,ZC,ZA,ZB,C1,CDSORT
582 DIMENSION VG(32,34),X(8),Y(8),G(8),BK(32,34),VA(136),VI(32,34),
583 +VB(136),W(8)
584 COMMON/T3/BCR(2,18),EK(171),XXY(18),KCRK(9),LP(18)
585 EQUIVALENCE(Z,ZETT),(UX,UXK)
586 EQUIVALENCE(VI(1,1),VG(1,1))
587 DATA Y/-3061799,-3364693,0.,5364693,0.,5364693,0.,5364693,
588 DATA Y/-3061799,-3364693,0.,5364693,0.,5364693,0.,5364693,
589 DATA NMT/9/
590 C1=CMPLX(0.E0,1.E0)
591 IF(KEY.EQ.1)NDE=10
592 IF(KEY.EQ.2)NDE=18
593 ELO=1.0
594 NT=KEY*NMT
595 NNPE=NDE/2
596 IF(NCONT.EQ.0)GOTO5
597 DO 61I=1,NNPE
598 12=1+2
599 61 CONTINUE
600 5 CONTINUE

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601 NNN=(NNPE-1)/2
602 NTT=(NNT+NNT+NNT)/2
603 DO 11=1,NTT
604 VA(1)=0.
605 VB(1)=0.
606 DO 41=1,NT
607 DO 4J=1,NDOPE
608 VG(1,J)=0.
609 C
610 INTEGRATION COEFFICIENTS
611 DO 111=1,5
612 X(11)=1./Y(11)/2.
613 11 Q(11)=W(11)/2.
614 13IDE=NNPE/KEY-1/KEY
615 ES=SQRT((XXY(11)-XC)*2+(XXY(2)-YC)*2)
616 UXX=(XXY(2)-YC)/ES
617 UY=(XXY(1)-XC)/ES
618 DO 4401=1,1SIDE
619 11=1
620 XXY(11-1)=(XXY(11-1)-XC)/ES
621 XX(11)=(XXY(11)-YC)/ES
622 XT=XXY(11-1)*UY+XXY(11)*UX
623 XX(11)=XXY(11-1)*UX+XXY(11)*UY
624 XXY(11-1)=XT
625 13IDE=NNPE/KEY-1/KEY
626 DO 411S1=1,1SIDE
627 IV=IS1/IS1
628 UXX=XXY(11+2)-XXY(11)
629 UY=XXY(11+1)-XXY(11+1)
630 DO 4111=1,5
631 XX=XXY(11+1)=1.-X(111)*XXY(11+1)+X(11)
632 YY=XXY(11+1)=1.-X(111)*XXY(11+2)+X(11)
633 Z=CMPLX(XA,YY)
634 ZET=CSORT(ZET)
635 ZET4=ZET*ZET
636 C2Z=CONJG(ZET)
637 ZETK=1./ZET4
638 KK=1
639 IF(KEY.EQ.2)GOTO1011
640 DO 1010K=1,NNT
641 FF2(K)=0.
642 ZETK=ZETK*ZET
643 CZK=CONJG(ZETK)
644 KK=-KK
645 IF(UY.EQ.0.)GOTO1009
646 ZD=C2K=C2Z-KK*ZETK=ZET
647 ZC=ZETK*(C2Z-ZET)
648 FF2(K)=0.5*K*(K-2)*ZC+K*ZD
649 FF2(K)=FF2(K)+UY*.5
650 ZE=C2K=C2Z*(C2Z-ZET)
651 FF3(K)=ETA*ZETK=ZET4*KK+C2K=CONJG(ZET4)*.5*K*ZE
652 FF3(K)=FF3(K)+.25
653 IF(UX.EQ.0.)GOTO1010
654 IB=K*2+KK+KK
655 ZA=(K-2)*ZETK-2.*CZK
656 FF2(K)=K*(C2Z-ZA-IB*ZETK*ZET)=CI*.25*UX+FF2(K)
657 1010 CONTINUE
658 GOTO 2000
659 DO 1012K=1,NNT
660 FF2(K)=0.
FF2(K+NNT)=0.

NNN=(NNPE-1)/2
NTT=(NNT+NNT+NNT)/2
DO 11=1,NTT
VA(1)=0.
VB(1)=0.
DO 41=1,NT
DO 4J=1,NDOPE
VG(1,J)=0.
C
INTEGRATION COEFFICIENTS
DO 111=1,5
X(11)=1./Y(11)/2.
11 Q(11)=W(11)/2.
13IDE=NNPE/KEY-1/KEY
ES=SQRT((XXY(11)-XC)*2+(XXY(2)-YC)*2)
UXX=(XXY(2)-YC)/ES
UY=(XXY(1)-XC)/ES
DO 4401=1,1SIDE
11=1
XXY(11-1)=(XXY(11-1)-XC)/ES
XX(11)=(XXY(11)-YC)/ES
XT=XXY(11-1)*UY+XXY(11)*UX
XX(11)=XXY(11-1)*UX+XXY(11)*UY
XXY(11-1)=XT
13IDE=NNPE/KEY-1/KEY
DO 411S1=1,1SIDE
IV=IS1/IS1
UXX=XXY(11+2)-XXY(11)
UY=XXY(11+1)-XXY(11+1)
DO 4111=1,5
XX=XXY(11+1)=1.-X(111)*XXY(11+1)+X(11)
YY=XXY(11+1)=1.-X(111)*XXY(11+2)+X(11)
Z=CMPLX(XA,YY)
ZET=CSORT(ZET)
ZET4=ZET*ZET
C2Z=CONJG(ZET)
ZETK=1./ZET4
KK=1
IF(KEY.EQ.2)GOTO1011
DO 1010K=1,NNT
FF2(K)=0.
ZETK=ZETK*ZET
CZK=CONJG(ZETK)
KK=-KK
IF(UY.EQ.0.)GOTO1009
ZD=C2K=C2Z-KK*ZETK=ZET
ZC=ZETK*(C2Z-ZET)
FF2(K)=0.5*K*(K-2)*ZC+K*ZD
FF2(K)=FF2(K)+UY*.5
ZE=C2K=C2Z*(C2Z-ZET)
FF3(K)=ETA*ZETK=ZET4*KK+C2K=CONJG(ZET4)*.5*K*ZE
FF3(K)=FF3(K)+.25
IF(UX.EQ.0.)GOTO1010
IB=K*2+KK+KK
ZA=(K-2)*ZETK-2.*CZK
FF2(K)=K*(C2Z-ZA-IB*ZETK*ZET)=CI*.25*UX+FF2(K)
1010 CONTINUE
GOTO 2000
DO 1012K=1,NNT
FF2(K)=0.
FF2(K+NNT)=0.

ZETK=ZETK*ZET
CZK=CONJG(ZETK)
KK=-KK
IF(UY.EQ.0.)GOTO1014
ZD=C2K=C2Z-KK*ZETK*ZET
ZC=ZETK*(C2Z-ZET)
FF2(K)=0.5*K*(K-2)*ZC+K*ZD
FF2(K)=FF2(K)+UY*.5
FF2(K+NNT)=(FF2(K)-K*ZD+UY)*CI
ZE=C2K=C2Z*(C2Z-ZET)
FF3(K)=ETA*ZETK=ZET4*KK+C2K=CONJG(ZET4)*.5*K*ZE
FF3(K)=FF3(K)+.25
FF3(K+NNT)=(FF3(K)-.25*K*ZE)*CI
IF(UX.EQ.0.)GOTO1012
IB=K*2+KK+KK
ZA=(K-2)*ZETK-2.*CZK
FF2(K)=K*(C2Z-ZA-IB*ZETK*ZET)=CI*.25*UX+FF2(K)
1012 CONTINUE
2000 KJ=0
DO 41K=1,NNT
L=2*IS1-1
DO 40J=1,KEY
I=J+NNT-NNT*K
SINT=FF2(11)*X(11)
SINT=FF2(11)*SINT
VG(1,L)=VG(1,L)+Q(11)*AIMAG(SINT)
VG(1,L+1)=VG(1,L+1)+Q(11)*REAL(SINT)
VG(1,L+2)=VG(1,L+2)+Q(11)*REAL(SINT)
40 VG(1,L+3)=VG(1,L+3)+Q(11)*REAL(SINT)
DO 41J=1,K
SINT=FF2(K)*FF3(J)*FF2(J)*FF3(K)
KJ=KJ+1
VA(KJ)=VA(KJ)+Q(11)*AIMAG(SINT)/SHU
IF(KEY.EQ.1)GOTO41
L=J+NNT
SINT=FF2(11)*FF3(L)*FF2(L)*FF3(11)
VB(KJ)=VB(KJ)+Q(11)*AIMAG(SINT)/SHU
CONTINUE
702 41
IF(KEY.EQ.1)GOTO64
DO 50011=1,NNT
VG(111,2)=0.
VG(111,1)=2.*VG(111,1)
11=111+NNT
VG(111,2)=2.*VG(111,2)
VG(111,1)=0.
DO 456J=1,NNN
JU=J+1
JL=NNPE+1-J
VG(111,2+JL)=VG(111,2+JU-1)
VG(111,2+JL)=VG(111,2+JU)
VG(111,2+JL-1)=VG(111,2+JU-1)
455 VG(111,2+JL)=VG(11,2+JU)
712 JU=J+1
JL=NNPE+1-J
VG(111,2+JL)=VG(111,2+JU-1)
VG(111,2+JL)=VG(111,2+JU)
VG(111,2+JL-1)=VG(111,2+JU-1)
716 455
717 500 CONTINUE
718 DO 5011=1,NDOPE
719 501 VG(NNT+2,1)=0.
720 DO 631=1,NTT

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721 83 VA(1)=VA(1)+2.
722 VB(1)=VB(1)+2.
723 VB(2)=0.
724 VB(3)=1.
725 DO 621=3,NNT
726 VB((1+1)-1/2+2)=0.
727 64 CONTINUE
728 CALL SINV(VA,NNT,1E-05,IER)
729 IF(IEQ.EQ.2)CALLSINV(VB,NNT,1E-05,IEE)
730 DO 111J=1,NNOE
731 DO 1101=1,NNT
732 11=(1+1)-1/2
733 BK(1,J)=0.
734 DO 110K=1,NNT
735 IK=1+K
736 IF(K.GT.1)IK=(K+K)/2+1
737 BK(1,J)=BK(1,J)+VA(1+K)*V(1+K,J)
738 IF(IEQ.EQ.1)GOTO111
739 J=NNT+1
740 DO 1211=1,NNT
741 1J=1-NNT
742 1J=(1J+1J)/2
743 BK(1,J)=0.
744 DO 121K=1,NNT
745 IK=1+K
746 IF(K.GT.1)IK=(K+K)/2+1-NNT
747 BK(1,J)=BK(1,J)+VB(1+K)*V(1+K+NNT,J)
748 111 CONTINUE
749 1J=0
750 DO 1121=1,NNOE
751 DO 112J=1,1
752 1J=1J+1
753 EK(1J)=0.
754 DO 112K=1,NNT
755 EK(1J)=EK(1J)+BK(1,J)*V(1+K,1)
756 ELO=SQRT(2./ES)
757 DO 113J=1,NNOE
758 BCR(2,J)=0.
759 IF(IEQ.EQ.2)BCR(2,J)=ELO*BK(10,J)
760 113 BCR(1,J)=ELO*BK(1,J)
761 RETURN
762 END
763 SUBROUTINE SIFAC(M,TOTAL)
764 C IMPLICIT REAL*8 (A-H,O-Z)
765 IMPLICIT REAL*8 (A-H,O-Z)
766 REAL*8 K1,K2
767 REAL K1,K2
768 COMMON/TWO/IBAND,NEG,R(1400),AK(1400,100)
769 COMMON/T3/BCR(2,18),EK(171),XXY(18),KCRK(9),LP(18)
770 WRITE(16,35)M
771 35 FORMAT('O',SX,'CKEL =',IS,/)
772 READ(12)KK,NODE,NOE,KEY,(LP(1),I=1,NNOE),(EK(1),I=1,KK),(BCR(1,J)
773 *,I=1,KEY),J=1,NNOE),XC,YC,MATYP,(KCRK(1),I=1,NODE)
774 TOTAL=0.
775 K1=0.
776 K2=0.
777 DO 201=1,NNOE
778 K1=K1+BCR(1,1)*R(LP(1))
779 IF(IEQ.EQ.1)GOTO20
780 K2=K2+BCR(2,1)*R(LP(1))

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781 20 CONTINUE
782 P1=3.1415926
783 K1=K1+SQRT(P1)
784 K2=K2+SQRT(P1)
785 WRITE(16,1)M,K1,K2
786 1 FORMAT('O',SX,'... STRESS INTENSIVITY FACTOR ...',12,/, 'O',SX,
787 + 'OPENING MODE K1 =',G20.6,/, 'O',SX, 'SHEARING MODE K2 =',G20.
788 +6)
789 WRITE(16,100)XC,YC,MATYP
790 100 FORMAT('O',SX,'CRACK TIP XC =',G16.6, ' YC =',G16.6,
791 + ' AT MATERIAL',14)
792 WRITE(16,200)NODE,(KCRK(1),I=1,NODE)
793 200 FORMAT('O',SX,THE,14,'NODES =',1014)
794 N=0.
795 DO 301=1,NNOE
796 XXY(1)=0.
797 DO 30J=1,NNOE
798 CALL LOC(1,J,1J,NNOE,NOE,1)
799 30 XXY(1)=XXY(1)+R(LP(J))*EK(1J)*.8
800 DO 401=1,NNOE
801 40 N=N+XXY(1)*R(LP(1))
802 WRITE(16,2)N
803 2 FORMAT('O',SX,'STRAIN ENERGY =',G16.6)
804 TOTAL=TOTAL+N
805 WRITE(16,300)
806 300 FORMAT('O',10X,' =',G20.6,/, 'O',SX, 'SHEARING MODE K2 =',G20.
807 +6)
808 RETURN
809 END
810 SUBROUTINE GEOMBC(U,N)
811 C IMPLICIT REAL*8 (A-H,O-Z)
812 COMMON/TWO/IBAND,NEG,R(1400),AK(1400,100)
813 C THIS SUBROUTINE MODIFIES THE ASSEMBLAGE STIFFNESS AND LOADS FOR THE
814 C PRESCRIBED DISPLACEMENT U AT DEGREE OF FREEDOM N, EQ.(6-188). (REF.1)
815 DO 100M=2,IBAND
816 K=N-M+1
817 IF(K.LE.0)GOTO50
818 R(K)=R(K)-AK(K,M)*U
819 AK(K,M)=0.0
820 K=N-M-1
821 IF(K.GT.NEG)GOTO100
822 R(K)=R(K)-AK(N,M)*U
823 AK(N,M)=0.0
824 100 CONTINUE
825 AK(N,1)=1.0
826 R(N)=U
827 RETURN
828 END
829 SUBROUTINE STRESS(WORK)
830 C IMPLICIT REAL*8 (A-H,O-Z)
831 IMPLICIT REAL*8 (A-H,O-Z)
832 DIMENSION SIG(6)
833 C COMMON/CONS/NNP,NEL,NMAT,NSLC,NOPT,NBODY,MTYP,NCKEL
834 COMMON/ONE/OK(10,10),Q(10),B(3,10),C(3,3),BT(3,6),XQ(5),VQ(5)
835 COMMON/TWO/IBAND,NEG,R(1400),AK(1400,100)
836 COMMON/T1/IE(700,5)
837 REVIND 11
838 REVIND 13
839 REVIND 13

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841 WRITE(16,300)
842 WORK=0.0
843 NOLINE=47
844 C
845 C RETRIEVE MULTIPLIERS, PIVOTS, MATRICES B AND C, AND CENTROIDAL COORD.
846 C FOR ELEMENT
847 DO 3M=1,NEL
848 READ(11)((OK(I,J),J=1,10),I=1,2),Q(9),Q(10),((B(I,J),J=1,10),I=1,
849 >3),((C(I,J),J=1,3),I=1,3),XC,YC
850 C
851 C SELECT NODAL DISPLACEMENTS FOR THE ELEMENT
852 LIM=4
853 IF(IE(M,3).EQ.IE(M,4))LIM=3
854 DO 10I=1,LIM
855 I1=2*I
856 JJ=2+IE(M,1)
857 Q(I1-1)=R(JJ-1)
858 Q(I1)=R(JJ)
859 C
860 C RECOVER CONDENSED DISPLACEMENTS FOR THE QUADRILATERAL, EQ. (5-64B)
861 IF(LIM.EQ.3)GOTO16
862 DO 15K=1,2
863 JK=K*8
864 IK=JK-1
865 DO 15L=1,1K
866 Q(JK)=Q(JK)-Q(K,L)*Q(L)
867 C
868 C COMPUTE ELEMENT STRAINS, EQ. (5-35A)
869 LIM=10
870 FAC=0.25
871 GOTO 17
872 16 LIM=6
873 FAC=1.0
874 DO 20I=1,3
875 E(I)=0.0
876 DO 20J=1,LIM
877 E(I)=E(I)+B(I,J)*Q(J)*FAC
878 C
879 C COMPUTE STRAIN ENERGY IN EACH ELEMENT
880 READ(13)KK,((OK(I,J),J=1,KK),I=1,KK)
881 DO 40I=1,KK
882 RO(I)=0.0
883 DO 40J=1,KK
884 RO(I)=RO(I)+0.5*Q(J)*OK(I,J)
885 W=0.0
886 DO 50I=1,KK
887 W=W+RO(I)*Q(I)
888 WORK=WORK+W
889 C
890 C COMPUTE ELEMENT STRESSES, EQ. (5-35B)
891 DO 30I=1,3
892 SIG(I)=0.0
893 DO 30J=1,3
894 SIG(I)=SIG(I)+C(I,J)*E(J)
895 C
896 C COMPUTE PRINCIPAL STRESSES AND THE ANGLE WITH THE POSITIVE X AXIS
897 SP=(SIG(1)+SIG(2))/2.0
898 SH=(SIG(1)-SIG(2))/2.0
899 DS=SQRT(SH*SH+SIG(3)*SIG(3))
900 SIG(4)=SP+DS
901 SIG(5)=SP-DS
902
901 SIG(6)=0.0
902 IF(SIG(3).NE.0.0.AND.SH.NE.0.0)SIG(6)=28.648*ATAN2(SIG(3),SH)
903 C PRINT STRESSES, 50 LINES PER PAGE
904 IF(NOLINE.GT.0)GOTO54
905 WRITE(16,1000)
906 NOLINE=49
907 54 NOLINE=NOLINE-1
908 5 WRITE(16,1010)M,XC,YC,(SIG(I),I=1,6)
909 21 FORMAT(16,21)WORK
910 21 FORMAT(16,21)WORK
911 ENDOFILE11
912 300 FORMAT(16,21)OUTPUT TABLE 2.. STRESSES AT ELEMENT CENTROIDS//IX,
913 "ELEMENT",9X,"X",9X,"Y",4X,"SIGMA(X)",4X,"SIGMA(Y)",4X,
914 "TAUX,Y)",4X,"SIGMA(1)",4X,"SIGMA(2)",7X,"ANGLE"
915 1000 FORMAT(16,21)ELEMENT,9X,"X",9X,"Y",4X,"SIGMA(X)",4X,
916 "SIGMA(Y)",4X,"TAUX,Y)",4X,"SIGMA(1)",4X,"SIGMA(2)",7X,
917 "ANGLE"
918 1010 FORMAT(16,21)10.2,1P6E12.4)
919 RETURN
920 END
921 SUBROUTINE BANSOL(A,B,NEQ,NBAND,MAXDOF,MAXBW)
922 C IMPLICIT REAL*8 (A-H,O-Z)
923 IMPLICIT REAL*4 (A-H,O-Z)
924 DIMENSION A(MAXDOF,MAXBW),B(1)
925 DO 10N=1,NEQ
926 I=N
927 DO 20L=2,NBAND
928 I=I+1
929 IF(A(N,L).EQ.0.100T020
930 C=A(N,L)/A(N,1)
931 J=0
932 DO 30K=L,NBAND
933 J=J+1
934 IF(A(N,K).EQ.0.100T030
935 A(I,J)=A(I,J)-C*A(N,K)
936 CONTINUE
937 A(N,L)=C
938 C REDUCED LOAD VECTOR
939 B(I)=B(I)-C*B(N)
940 CONTINUE
941 10 B(N)=B(N)/A(N,1)
942 C BACK SUBSTITUTION
943 N=NEQ
944 N=N-1
945 IF(N.LE.0)GOTO43
946 L=N
947 DO 40K=2,NBAND
948 L=L+1
949 IF(A(N,K).EQ.0.100T040
950 B(N)=B(N)-A(N,K)*B(L)
951 CONTINUE
952 GOTO 35
953 43 RETURN
954 END
955 SUBROUTINE SINVA(N,EPS,IER)
956 C REAL*8 A(1) APS,DIN,WORK
957 REAL A(1),APS,DIN,WORK
958 CALL HPSO(A,N,EPS,IER)
959 IF(ITER).1,1
960 1 IPIV=N*(N+1)/2

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1021      7 IF(IER)8,9
1022      8 IER=K-1
1023      9 DPIV=SQRT(DSUM)
1024      A(KPIV)=DPIV
1025      DPIV=1.0/DPIV
1026      GOTO 11
1027      10 A(LIND)=DSUM=DPIV
1028      11 IND=IND+1
1029      12 RETURN
1030      12 IER=-1
1031      RETURN
1032      END

```

```

961      IND=PIV
962      DO 61=1,N
963      DIN=1.0/DBLE(A(PIV))
964      A(PIV)=DIN
965      MIN=N
966      KEND=1-1
967      LANF=N-KEND
968      IF(KEND)5,5,2
969      2 J=IND
970      DO 4K=1,KEND
971      WORK=0.0
972      MIN=MIN-1
973      LHOR=PIV
974      LVER=J
975      DO 3L=LANF,MIN
976      LVER=LVER+1
977      LHOR=LHOR+L
978      3 WORK=WORK+DBLE(A(LVER)*A(LHOR))
979      A(J)=--WORK=DIN
980      4 J=J-MIN
981      5 PIV=PIV-MIN
982      6 IND=IND-1
983      DO 61=1,N
984      PIV=PIV+1
985      J=PIV
986      DO 8K=1,N
987      WORK=0.0
988      LHOR=J
989      DO 7L=K,N
990      LVER=LHOR+K-1
991      7 WORK=WORK+DBLE(A(LHOR)*A(LVER))
992      LHOR=LHOR+L
993      A(J)=WORK
994      8 J=J+K
995      9 RETURN
996      END
997      SUBROUTINE MFSD(A,N,EPS,IER)
998      REAL*8 A(1),EPS,TOL
999      REAL A(1),EPS,TOL
1000      REAL*8 DPIV,DSUM
1001      IF(N-1)12,1,1
1002      1 IER=0
1003      KPIV=0
1004      DO 11K=1,N
1005      KPIV=KPIV+K
1006      IND=KPIV
1007      LEND=K-1
1008      TOL=ABS(EPS*A(KPIV))
1009      DO 111=K,N
1010      DSUM=0.0
1011      IF(LEND)2,4,2
1012      2 DO 3L=1,LEND
1013      LANF=KPIV-L
1014      LIND=IND-L
1015      3 DSUM=DSUM+DBLE(A(LANF)*A(LIND))
1016      4 DSUM=DSUM+DBLE(A(IND)*A(IND))
1017      IF(1-K)10,5,10
1018      5 IF(DSUM)-TOL)6,6,9
1019      6 IF(DSUM)12,12,7
1020

```

Sample Input for Finite Element Fracture Program- The following pages contain the input cards necessary to run the finite element program.

61	99	0 58.0	28.0	0.0	0.0
62	103	0 66.0	28.0	0.0	0.0
63	105	0 68.0	28.0	0.0	0.0
64	107	0 69.0	28.0	0.0	0.0
65	108	0 69.3	28.0	0.0	0.0
66	109	0 69.75	28.0	0.0	0.0
67	110	1 69.95	28.0	0.0	0.0
68	111	3 0.0	34.0	0.0	0.0
69	114	0 24.0	34.0	0.0	0.0
70	117	0 42.0	34.0	0.0	0.0
71	121	0 58.0	34.0	0.0	0.0
72	125	0 66.0	34.0	0.0	0.0
73	127	0 68.0	34.0	0.0	0.0
74	129	0 69.0	34.0	0.0	0.0
75	130	0 69.3	34.0	0.0	0.0
76	131	0 69.75	34.0	0.0	0.0
77	132	1 69.95	34.0	0.0	0.0
78	133	3 0.0	40.0	0.0	0.0
79	136	0 24.0	40.0	0.0	0.0
80	139	0 42.0	40.0	0.0	0.0
81	143	0 58.0	40.0	0.0	0.0
82	147	0 66.0	40.0	0.0	0.0
83	149	0 68.0	40.0	0.0	0.0
84	151	0 69.0	40.0	0.0	0.0
85	152	0 69.3	40.0	0.0	0.0
86	153	0 69.75	40.0	0.0	0.0
87	154	1 69.95	40.0	0.0	0.0
88	155	3 0.0	44.0	0.0	0.0
89	158	0 24.0	44.0	0.0	0.0
90	161	0 42.0	44.0	0.0	0.0
91	165	0 58.0	44.0	0.0	0.0
92	169	0 66.0	44.0	0.0	0.0
93	171	0 68.0	44.0	0.0	0.0
94	173	0 69.0	44.0	0.0	0.0
95	174	0 69.3	44.0	0.0	0.0
96	175	0 69.75	44.0	0.0	0.0
97	176	1 69.95	44.0	0.0	0.0
98	177	3 0.0	46.0	0.0	0.0
99	180	0 24.0	46.0	0.0	0.0
100	183	0 42.0	46.0	0.0	0.0
101	187	0 58.0	46.0	0.0	0.0
102	191	0 66.0	46.0	0.0	0.0
103	193	0 68.0	46.0	0.0	0.0
104	195	0 69.0	46.0	0.0	0.0
105	196	0 69.3	46.0	0.0	0.0
106	197	0 69.75	46.0	0.0	0.0
107	198	1 69.95	46.0	0.0	0.0
108	199	3 0.0	48.0	0.0	0.0
109	202	0 24.0	48.0	0.0	0.0
110	205	0 42.0	48.0	0.0	0.0
111	209	0 58.0	48.0	0.0	0.0
112	213	0 66.0	48.0	0.0	0.0
113	215	0 68.0	48.0	0.0	0.0
114	217	0 69.0	48.0	0.0	0.0
115	218	0 69.3	48.0	0.0	0.0
116	219	0 69.75	48.0	0.0	0.0
117	220	1 69.95	48.0	0.0	0.0
118	221	3 0.0	49.0	0.0	0.0
119	224	0 24.0	49.0	0.0	0.0
120	227	0 42.0	49.0	0.0	0.0

121	0 58.0	49.0	0.0	0.0
122	0 66.0	49.0	0.0	0.0
123	0 68.0	49.0	0.0	0.0
124	0 69.0	49.0	0.0	0.0
125	0 69.3	49.0	0.0	0.0
126	0 69.75	49.0	0.0	0.0
127	0 69.95	49.0	0.0	0.0
128	3 0.0	50.0	0.0	0.0
129	0 24.0	50.0	0.0	0.0
130	0 42.0	50.0	0.0	0.0
131	0 58.0	50.0	0.0	0.0
132	0 66.0	50.0	0.0	0.0
133	0 68.0	50.0	0.0	0.0
134	0 69.0	50.0	0.0	0.0
135	0 69.3	50.0	0.0	0.0
136	0 69.75	50.0	0.0	0.0
137	0 69.95	50.0	0.0	0.0
138	3 0.0	50.833	0.0	0.0
139	0 24.0	50.833	0.0	0.0
140	0 42.0	50.833	0.0	0.0
141	0 58.0	50.833	0.0	0.0
142	0 66.0	50.833	0.0	0.0
143	0 68.0	50.833	0.0	0.0
144	0 69.0	50.833	0.0	0.0
145	0 69.3	50.833	0.0	0.0
146	0 69.75	50.833	0.0	0.0
147	0 69.95	50.833	0.0	0.0
148	3 0.0	51.5	0.0	0.0
149	0 24.0	51.5	0.0	0.0
150	0 42.0	51.5	0.0	0.0
151	0 58.0	51.5	0.0	0.0
152	0 66.0	51.5	0.0	0.0
153	0 68.0	51.5	0.0	0.0
154	0 69.0	51.5	0.0	0.0
155	0 69.3	51.5	0.0	0.0
156	0 69.75	51.5	0.0	0.0
157	0 69.95	51.5	0.0	0.0
158	3 0.0	51.833	0.0	0.0
159	0 24.0	51.833	0.0	0.0
160	0 42.0	51.833	0.0	0.0
161	0 58.0	51.833	0.0	0.0
162	0 66.0	51.833	0.0	0.0
163	0 68.0	51.833	0.0	0.0
164	0 69.0	51.833	0.0	0.0
165	0 69.3	51.833	0.0	0.0
166	0 69.75	51.833	0.0	0.0
167	0 69.95	51.833	0.0	0.0
168	3 0.0	52.167	0.0	0.0
169	0 24.0	52.167	0.0	0.0
170	0 42.0	52.167	0.0	0.0
171	0 58.0	52.167	0.0	0.0
172	0 66.0	52.167	0.0	0.0
173	0 68.0	52.167	0.0	0.0
174	0 69.0	52.167	0.0	0.0
175	0 69.3	52.167	0.0	0.0
176	0 69.75	52.167	0.0	0.0
177	0 69.95	52.167	0.0	0.0
178	3 0.0	52.5	0.0	0.0
179	0 24.0	52.5	0.0	0.0
180	0 42.0	52.5	0.0	0.0
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181	363	0 58.0	52.5	0.0	0.0
182	367	0 66.0	52.5	0.0	0.0
183	369	0 68.0	52.5	0.0	0.0
184	371	0 69.0	52.5	0.0	0.0
185	372	0 69.3	52.5	0.0	0.0
186	373	0 69.75	52.5	0.0	0.0
187	374	0 69.95	52.5	0.0	0.0
188	375	3 0.0	52.833	0.0	0.0
189	378	0 24.0	52.833	0.0	0.0
190	381	0 42.0	52.833	0.0	0.0
191	385	0 58.0	52.833	0.0	0.0
192	389	0 66.0	52.833	0.0	0.0
193	391	0 68.0	52.833	0.0	0.0
194	393	0 69.0	52.833	0.0	0.0
195	394	0 69.3	52.833	0.0	0.0
196	395	0 69.75	52.833	0.0	0.0
197	396	0 69.95	52.833	0.0	0.0
198	397	3 0.0	53.25	0.0	0.0
199	400	0 24.0	53.25	0.0	0.0
200	403	0 42.0	53.25	0.0	0.0
201	407	0 58.0	53.25	0.0	0.0
202	411	0 66.0	53.25	0.0	0.0
203	413	0 68.0	53.25	0.0	0.0
204	415	0 69.0	53.25	0.0	0.0
205	416	0 69.3	53.25	0.0	0.0
206	417	0 69.75	53.25	0.0	0.0
207	418	0 69.95	53.25	0.0	0.0
208	419	3 0.0	53.5	0.0	0.0
209	422	0 24.0	53.5	0.0	0.0
210	425	0 42.0	53.5	0.0	0.0
211	429	0 58.0	53.5	0.0	0.0
212	433	0 66.0	53.5	0.0	0.0
213	435	0 68.0	53.5	0.0	0.0
214	437	0 69.0	53.5	0.0	0.0
215	438	0 69.3	53.5	0.0	0.0
216	439	0 69.75	53.5	0.0	0.0
217	440	3 0.0	53.75	0.0	0.0
218	443	0 24.0	53.75	0.0	0.0
219	446	0 42.0	53.75	0.0	0.0
220	450	0 58.0	53.75	0.0	0.0
221	454	0 66.0	53.75	0.0	0.0
222	456	0 68.0	53.75	0.0	0.0
223	458	0 69.0	53.75	0.0	0.0
224	459	0 69.3	53.75	0.0	0.0
225	460	0 69.75	53.75	0.0	0.0
226	461	1 70.0	53.75	0.0	0.0
227	462	3 0.0	54.1	0.0	0.0
228	465	0 24.0	54.1	0.0	0.0
229	468	0 42.0	54.1	0.0	0.0
230	472	0 58.0	54.1	0.0	0.0
231	476	0 66.0	54.1	0.0	0.0
232	478	0 68.0	54.1	0.0	0.0
233	480	0 69.0	54.1	0.0	0.0
234	481	0 69.3	54.1	0.0	0.0
235	482	0 69.75	54.1	0.0	0.0
236	483	1 70.0	54.1	0.0	0.0
237	484	3 0.0	54.5	0.0	0.0
238	487	0 24.0	54.5	0.0	0.0
239	490	0 42.0	54.5	0.0	0.0
240	494	0 58.0	54.5	0.0	0.0

241	498	0 66.0	54.5	0.0	0.0
242	500	0 68.0	54.5	0.0	0.0
243	502	0 69.0	54.5	0.0	0.0
244	503	0 69.3	54.5	0.0	0.0
245	504	0 69.75	54.5	0.0	0.0
246	505	1 70.0	54.5	0.0	0.0
247	506	3 0.0	55.5	0.0	0.0
248	509	0 24.0	55.5	0.0	0.0
249	512	0 42.0	55.5	0.0	0.0
250	516	0 58.0	55.5	0.0	0.0
251	520	0 66.0	55.5	0.0	0.0
252	522	0 68.0	55.5	0.0	0.0
253	524	0 69.0	55.5	0.0	0.0
254	525	0 69.3	55.5	0.0	0.0
255	526	0 69.75	55.5	0.0	0.0
256	527	1 70.0	55.5	0.0	0.0
257	528	3 0.0	56.5	0.0	0.0
258	531	0 24.0	56.5	0.0	0.0
259	534	0 42.0	56.5	0.0	0.0
260	538	0 58.0	56.5	0.0	0.0
261	542	0 66.0	56.5	0.0	0.0
262	544	0 68.0	56.5	0.0	0.0
263	546	0 69.0	56.5	0.0	0.0
264	547	0 69.3	56.5	0.0	0.0
265	548	0 69.75	56.5	0.0	0.0
266	549	1 70.0	56.5	0.0	0.0
267	550	3 0.0	58.0	0.0	0.0
268	553	0 24.0	58.0	0.0	0.0
269	556	0 42.0	58.0	0.0	0.0
270	560	0 58.0	58.0	0.0	0.0
271	564	0 66.0	58.0	0.0	0.0
272	566	0 68.0	58.0	0.0	0.0
273	568	0 69.0	58.0	0.0	0.0
274	569	0 69.3	58.0	0.0	0.0
275	570	0 69.75	58.0	0.0	0.0
276	571	1 70.0	58.0	0.0	0.0
277	572	3 0.0	59.5	0.0	0.0
278	575	0 24.0	59.5	0.0	0.0
279	578	0 42.0	59.5	0.0	0.0
280	582	0 58.0	59.5	0.0	0.0
281	586	0 66.0	59.5	0.0	0.0
282	588	0 68.0	59.5	0.0	0.0
283	590	0 69.0	59.5	0.0	0.0
284	591	0 69.3	59.5	0.0	0.0
285	592	0 69.75	59.5	0.0	0.0
286	593	1 70.0	59.5	0.0	0.0
287	594	3 0.0	61.0	0.0	0.0
288	597	0 24.0	61.0	0.0	0.0
289	600	0 42.0	61.0	0.0	0.0
290	604	0 58.0	61.0	0.0	0.0
291	608	0 66.0	61.0	0.0	0.0
292	610	0 68.0	61.0	0.0	0.0
293	612	0 69.0	61.0	0.0	0.0
294	613	0 69.3	61.0	0.0	0.0
295	614	0 69.75	61.0	0.0	0.0
296	615	1 70.0	61.0	0.0	0.0
297	1	1 2	24 23	2	
298	21	21 22	44 43	2	
299	22	23 24	46 45	2	
300	42	43 44	66 65	2	

Sample Outupt From Finite Element Fracture Program - The following pages contain the output from the finite element program run using the sample data.

PROBLEM 1... PHASE MESH, P= 53.

INPUT TABLE 1... BASIC PARAMETERS

NUMBER OF NODAL POINTS. 615
 NUMBER OF ELEMENTS. 565
 NUMBER OF DIFFERENT MATERIALS 3
 NUMBER OF SURFACE LOAD CARDS. 7
 1 = PLANE STRAIN, 2 = PLANE STRESS. 1
 BODY FORCES(1 = IN -Y DIREC., 0 = NONE) 0
 NUMBER OF CRACK ELEMENTS. 1

INPUT TABLE 2... MATERIAL PROPERTIES

MATERIAL NUMBER	MODULUS OF ELASTICITY	POISSON'S RATIO	MATERIAL DENSITY	MATERIAL THICKNESS
1	0.8000E+06	0.3500E+00	0.0000E+01	0.1000E+01
2	0.9000E+04	0.3500E+00	0.0000E+01	0.1000E+01
3	0.3500E+06	0.3500E+00	0.0000E+01	0.1000E+01

INPUT TABLE 3. NODAL POINT DATA

NODAL POINT	TYPE	X	Y	X-DISP. GR LOAD	Y-DISP. GR LOAD
1	3	0.000E+01	0.000E+01	0.000E+01	0.000E+01
2	3	0.800E+01	0.000E+01	0.000E+01	0.000E+01
3	3	0.160E+02	0.000E+01	0.000E+01	0.000E+01
4	3	0.240E+02	0.000E+01	0.000E+01	0.000E+01
5	3	0.300E+02	0.000E+01	0.000E+01	0.000E+01
6	3	0.360E+02	0.000E+01	0.000E+01	0.000E+01
7	3	0.420E+02	0.000E+01	0.000E+01	0.000E+01
8	3	0.460E+02	0.000E+01	0.000E+01	0.000E+01
9	3	0.500E+02	0.000E+01	0.000E+01	0.000E+01
10	3	0.540E+02	0.000E+01	0.000E+01	0.000E+01
11	3	0.580E+02	0.000E+01	0.000E+01	0.000E+01
12	3	0.600E+02	0.000E+01	0.000E+01	0.000E+01
13	3	0.620E+02	0.000E+01	0.000E+01	0.000E+01
14	3	0.640E+02	0.000E+01	0.000E+01	0.000E+01
15	3	0.660E+02	0.000E+01	0.000E+01	0.000E+01
16	3	0.670E+02	0.000E+01	0.000E+01	0.000E+01
17	3	0.680E+02	0.000E+01	0.000E+01	0.000E+01
18	3	0.6850E+02	0.000E+01	0.000E+01	0.000E+01
19	3	0.6900E+02	0.000E+01	0.000E+01	0.000E+01
20	3	0.6930E+02	0.000E+01	0.000E+01	0.000E+01
21	3	0.6975E+02	0.000E+01	0.000E+01	0.000E+01
22	3	0.6995E+02	0.000E+01	0.000E+01	0.000E+01
23	3	0.000E+01	0.800E+01	0.000E+01	0.000E+01
24	0	0.800E+01	0.800E+01	0.000E+01	0.000E+01
25	0	0.160E+02	0.800E+01	0.000E+01	0.000E+01
26	0	0.240E+02	0.800E+01	0.000E+01	0.000E+01
27	0	0.300E+02	0.800E+01	0.000E+01	0.000E+01
28	0	0.360E+02	0.800E+01	0.000E+01	0.000E+01
29	0	0.420E+02	0.800E+01	0.000E+01	0.000E+01
30	0	0.460E+02	0.800E+01	0.000E+01	0.000E+01
31	0	0.500E+02	0.800E+01	0.000E+01	0.000E+01
32	0	0.540E+02	0.800E+01	0.000E+01	0.000E+01
33	0	0.580E+02	0.800E+01	0.000E+01	0.000E+01
34	0	0.600E+02	0.800E+01	0.000E+01	0.000E+01
35	0	0.620E+02	0.800E+01	0.000E+01	0.000E+01
36	0	0.640E+02	0.800E+01	0.000E+01	0.000E+01
37	0	0.660E+02	0.800E+01	0.000E+01	0.000E+01
38	0	0.670E+02	0.800E+01	0.000E+01	0.000E+01
39	0	0.680E+02	0.800E+01	0.000E+01	0.000E+01
40	0	0.6850E+02	0.800E+01	0.000E+01	0.000E+01
41	0	0.6900E+02	0.800E+01	0.000E+01	0.000E+01
42	0	0.6930E+02	0.800E+01	0.000E+01	0.000E+01
43	0	0.6975E+02	0.800E+01	0.000E+01	0.000E+01
44	1	0.6995E+02	0.800E+01	0.000E+01	0.000E+01
45	3	0.000E+01	0.160E+02	0.000E+01	0.000E+01
46	0	0.800E+01	0.160E+02	0.000E+01	0.000E+01
47	0	0.160E+02	0.160E+02	0.000E+01	0.000E+01
48	0	0.240E+02	0.160E+02	0.000E+01	0.000E+01
49	0	0.300E+02	0.160E+02	0.000E+01	0.000E+01
50	0	0.360E+02	0.160E+02	0.000E+01	0.000E+01
51	0	0.420E+02	0.160E+02	0.000E+01	0.000E+01
52	0	0.460E+02	0.160E+02	0.000E+01	0.000E+01
53	0	0.500E+02	0.160E+02	0.000E+01	0.000E+01
54	0	0.540E+02	0.160E+02	0.000E+01	0.000E+01
55	0	0.580E+02	0.160E+02	0.000E+01	0.000E+01
56	0	0.600E+02	0.160E+02	0.000E+01	0.000E+01

597	0	0.2400E+02	0.6100E+02	0.0000E+01	0.0000E+01
598	0	0.3000E+02	0.6100E+02	0.0000E+01	0.0000E+01
599	0	0.3600E+02	0.6100E+02	0.0000E+01	0.0000E+01
600	0	0.4200E+02	0.6100E+02	0.0000E+01	0.0000E+01
601	0	0.4600E+02	0.6100E+02	0.0000E+01	0.0000E+01
602	0	0.5000E+02	0.6100E+02	0.0000E+01	0.0000E+01
603	0	0.5400E+02	0.6100E+02	0.0000E+01	0.0000E+01
604	0	0.5800E+02	0.6100E+02	0.0000E+01	0.0000E+01
605	0	0.6000E+02	0.6100E+02	0.0000E+01	0.0000E+01
606	0	0.6200E+02	0.6100E+02	0.0000E+01	0.0000E+01
607	0	0.6400E+02	0.6100E+02	0.0000E+01	0.0000E+01
608	0	0.6600E+02	0.6100E+02	0.0000E+01	0.0000E+01
609	0	0.6700E+02	0.6100E+02	0.0000E+01	0.0000E+01
610	0	0.6800E+02	0.6100E+02	0.0000E+01	0.0000E+01
611	0	0.6850E+02	0.6100E+02	0.0000E+01	0.0000E+01
612	0	0.6900E+02	0.6100E+02	0.0000E+01	0.0000E+01
613	0	0.6930E+02	0.6100E+02	0.0000E+01	0.0000E+01
614	0	0.6975E+02	0.6100E+02	0.0000E+01	0.0000E+01
615	1	0.7000E+02	0.6100E+02	0.0000E+01	0.0000E+01

INPUT TABLE 4... ELEMENT DATA

ELEMENT	GLOBAL INDICES OF ELEMENT NODES				MATERIAL
	1	2	3	4	
1	1	2	24	23	2
2	2	3	25	24	2
3	3	4	26	25	2
4	4	5	27	26	2
5	5	6	28	27	2
6	6	7	29	28	2
7	7	8	30	29	2
8	8	9	31	30	2
9	9	10	32	31	2
10	10	11	33	32	2
11	11	12	34	33	2
12	12	13	35	34	2
13	13	14	36	35	2
14	14	15	37	36	2
15	15	16	38	37	2
16	16	17	39	38	2
17	17	18	40	39	2
18	18	19	41	40	2
19	19	20	42	41	2
20	20	21	43	42	2
21	21	22	44	43	2
22	22	23	45	44	2
23	23	24	46	45	2
24	24	25	47	46	2
25	25	26	48	47	2
26	26	27	49	48	2
27	27	28	50	49	2
28	28	29	51	50	2
29	29	30	52	51	2
30	30	31	53	52	2
31	31	32	54	53	2
32	32	33	55	54	2
33	33	34	56	55	2
34	34	35	57	56	2
35	35	36	58	57	2
36	36	37	59	58	2
37	37	38	60	59	2
38	38	39	61	60	2
39	39	40	62	61	2
40	40	41	63	62	2
41	41	42	64	63	2
42	42	43	65	64	2
43	43	44	66	65	2
44	44	45	67	66	2
45	45	46	68	67	2
46	46	47	69	68	2
47	47	48	70	69	2
48	48	49	71	70	2
49	49	50	72	71	2
50	50	51	73	72	2
51	51	52	74	73	2
52	52	53	75	74	2
53	53	54	76	75	2
54	54	55	77	76	2
55	55	56	78	77	2
56	56	57	79	78	2
57	57	58	80	79	2
58	58	59	81	80	2

↓ x6

537	563	564	586	585	3
538	564	565	587	586	3
539	565	566	588	587	3
540	566	567	589	588	3
541	567	568	590	589	3
542	568	569	591	590	3
543	569	570	592	591	3
544	570	571	593	592	3
545	572	573	595	594	3
546	573	574	596	595	3
547	574	575	597	596	3
548	575	576	598	597	3
549	576	577	599	598	3
550	577	578	600	599	3
551	578	579	601	600	3
552	579	580	602	601	3
553	580	581	603	602	3
554	581	582	604	603	3
555	582	583	605	604	3
556	583	584	606	605	3
557	584	585	607	606	3
558	585	586	608	607	3
559	586	587	609	608	3
560	587	588	610	609	3
561	588	589	611	610	3
562	589	590	612	611	3
563	590	591	613	612	3
564	591	592	614	613	3
565	592	593	615	614	3

INPUT TABLE 5. SURFACE LOADING DATA

NODE I	NODE J	SURFACE LOAD INTENSITIES AT NODES			
		XI	XJ	YI	YJ
264	286	0.1129E+04	0.9277E+03	0.0000E+01	0.0000E+01
286	308	0.9277E+03	0.7659E+03	0.0000E+01	0.0000E+01
308	330	0.7659E+03	0.6851E+03	0.0000E+01	0.0000E+01
330	352	0.6851E+03	0.6043E+03	0.0000E+01	0.0000E+01
352	374	0.6043E+03	0.5235E+03	0.0000E+01	0.0000E+01
374	396	0.5235E+03	0.4428E+03	0.0000E+01	0.0000E+01
396	418	0.4428E+03	0.3418E+03	0.0000E+01	0.0000E+01
XXY(1) =		69.7500			
XXY(2) =		53.2500			
XXY(3) =		69.9500			
XXY(4) =		53.2500			
XXY(5) =		70.0000			
XXY(6) =		53.7500			
XXY(7) =		69.7500			
XXY(8) =		53.7500			
XXY(9) =		69.7500			
XXY(10) =		53.5000			
SMU ETA =		296296.	1.60000		
LP (1) =	833				
LP (2) =	834				
LP (3) =	835				
LP (4) =	836				
LP (5) =	921				
LP (6) =	922				
LP (7) =	919				
LP (8) =	920				
LP (9) =	877				
LP (10) =	878				
EK(1) =		417700.	AK(833, 1) =	0.228690E+07	
EK(2) =		184875.	AK(833, 2) =	-62038.5	
EK(4) =		-588078.	AK(833, 3) =	-0.147536E+07	

EK(7)=	684789.	AK(833, 4)=	586023.
EK(11)=	-837088.	AK(833, 89)=	-837088.
EK(16)=	299415.	AK(833, 90)=	299415.
EK(22)=	-59503.0	AK(833, 87)=	-59503.0
EK(29)=	-212898.	AK(833, 88)=	-212898.
EK(37)=	45405.0	AK(833, 45)=	-16180.3
EK(46)=	-174474.	AK(833, 46)=	-75708.6
EK(3)=	957870.	AK(834, 1)=	0.274957E+07
EK(5)=	-215972.	AK(834, 2)=	-117207.
EK(8)=	877590.	AK(834, 3)=	766150.
EK(12)=	-358996.	AK(834, 88)=	-358996.
EK(17)=	36150.5	AK(834, 89)=	36150.5
EK(23)=	5510.71	AK(834, 86)=	5510.71
EK(30)=	3410.25	AK(834, 87)=	3410.25
EK(38)=	-44951.4	AK(834, 44)=	-143717.
EK(47)=	-52300.7	AK(834, 45)=	-800582.
EK(6)=	968673.	AK(835, 1)=	0.192701E+07
EK(9)=	-0.123052E+07	AK(835, 2)=	-983604.
EK(13)=	0.150160E+07	AK(835, 87)=	0.150160E+07
EK(18)=	-672270.	AK(835, 88)=	-672270.
EK(24)=	62758.4	AK(835, 85)=	62758.4
EK(31)=	382421.	AK(835, 86)=	382421.
EK(39)=	-203353.	AK(835, 43)=	-203353.
EK(48)=	338999.	AK(835, 44)=	338999.
EK(10)=	0.296001E+07	AK(836, 1)=	0.337935E+07
EK(14)=	-0.227816E+07	AK(836, 86)=	-0.227816E+07
EK(19)=	0.128738E+07	AK(836, 87)=	0.128738E+07
EK(25)=	-123993.	AK(836, 84)=	-123993.
EK(32)=	-835522.	AK(836, 85)=	-835522.
EK(40)=	409289.	AK(836, 42)=	409289.

EK(49)=	-678742.	AK(836, 43)=	-678742.
EK(15)=	0.304262E+07AK(921, 1)=	0.370452E+07	
EK(20)=	-0.136613E+07AK(921, 2)=	-0.161304E+07	
EK(21)=	994082.	AK(922, 1)=	0.143286E+07
EK(26)=	17294.2	AK(919, 3)=	-538778.
EK(27)=	92972.3	AK(919, 4)=	191738.
EK(28)=	220480.	AK(919, 1)=	0.175314E+07
EK(35)=	-4516.77	AK(919, 2)=	242397.
EK(33)=	807875.	AK(920, 2)=	709109.
EK(34)=	-649759.	AK(920, 3)=	-629988.
EK(36)=	633420.	AK(920, 1)=	0.251987E+07
EK(41)=	-694315.	AK(877, 45)=	-694315.
EK(42)=	312595.	AK(877, 46)=	312595.
EK(43)=	-6956.74	AK(877, 43)=	-68542.1
EK(44)=	-186293.	AK(877, 44)=	-87527.8
EK(45)=	380178.	AK(877, 1)=	0.121665E+07
EK(54)=	-131997.	AK(877, 2)=	-131997.
EK(50)=	565610.	AK(878, 44)=	565610.
EK(51)=	-385751.	AK(878, 45)=	-385751.
EK(52)=	-70180.5	AK(878, 42)=	-168946.
EK(53)=	209317.	AK(878, 43)=	-538965.
EK(55)=	316467.	AK(878, 1)=	0.197764E+07
NODE NOE MATYP XC,YC = 5 10 1 70.000			53.500

OUTPUT TABLE 1... NODAL DISPLACEMENTS

NODE	U = X-DISP.	V = Y-DISP.
1	0.0000000E+01	0.0000000E+01
2	0.0000000E+01	0.0000000E+01
3	0.0000000E+01	0.0000000E+01
4	0.0000000E+01	0.0000000E+01
5	0.0000000E+01	0.0000000E+01
6	0.0000000E+01	0.0000000E+01
7	0.0000000E+01	0.0000000E+01
8	0.0000000E+01	0.0000000E+01
9	0.0000000E+01	0.0000000E+01
10	0.0000000E+01	0.0000000E+01
11	0.0000000E+01	0.0000000E+01
12	0.0000000E+01	0.0000000E+01
13	0.0000000E+01	0.0000000E+01
14	0.0000000E+01	0.0000000E+01
15	0.0000000E+01	0.0000000E+01
16	0.0000000E+01	0.0000000E+01
17	0.0000000E+01	0.0000000E+01
18	0.0000000E+01	0.0000000E+01
19	0.0000000E+01	0.0000000E+01
20	0.0000000E+01	0.0000000E+01
21	0.0000000E+01	0.0000000E+01
22	0.0000000E+01	0.0000000E+01
23	0.0000000E+01	0.0000000E+01
24	-0.2945927E-03	-0.2819338E-03
25	-0.4487865E-03	-0.4317782E-03
26	-0.5145746E-03	-0.5836628E-03
27	-0.5117065E-03	-0.6891591E-03
28	-0.4706058E-03	-0.7853508E-03
29	-0.4006168E-03	-0.8649527E-03
30	-0.3437290E-03	-0.9064071E-03
31	-0.2834363E-03	-0.9390745E-03
32	-0.2226507E-03	-0.9636558E-03
33	-0.1632620E-03	-0.9813158E-03
34	-0.1347513E-03	-0.9880497E-03
35	-0.1067910E-03	-0.9933996E-03
36	-0.7932068E-04	-0.9974851E-03
37	-0.5224253E-04	-0.1000395E-02
38	-0.3893725E-04	-0.1001425E-02
39	-0.2568292E-04	-0.1002157E-02
40	-0.1908827E-04	-0.1002412E-02
41	-0.1250112E-04	-0.1002592E-02
42	-0.8553206E-05	-0.1002664E-02
43	-0.2630293E-05	-0.1002721E-02
44	0.0000000E+01	-0.1002728E-02
45	0.0000000E+01	0.0000000E+01
46	-0.3815781E-03	-0.4253890E-03
47	-0.6529215E-03	-0.7894529E-03
48	-0.7745461E-03	-0.1148539E-02
49	-0.7718705E-03	-0.1403588E-02
50	-0.6996053E-03	-0.1630774E-02
51	-0.5777434E-03	-0.1810342E-02
52	-0.4816237E-03	-0.1897754E-02
53	-0.3834340E-03	-0.1960841E-02
54	-0.2898458E-03	-0.2003027E-02
55	-0.2048258E-03	-0.2029336E-02
56	-0.1661895E-03	-0.2038573E-02
57	-0.1297512E-03	-0.2045627E-02

\vec{e}_x
 \vec{e}_y

598	0.25433114E-02	-0.54546608E-02
599	0.25694593E-02	-0.68603169E-02
600	0.23651390E-02	-0.79864856E-02
601	0.20386126E-02	-0.84418006E-02
602	0.15607427E-02	-0.85685298E-02
603	0.91011823E-03	-0.82453191E-02
604	0.97091072E-04	-0.73450069E-02
605	-0.38923418E-03	-0.65925266E-02
606	-0.83608933E-03	-0.56247722E-02
607	-0.11483184E-02	-0.44416935E-02
608	-0.11743366E-02	-0.31315278E-02
609	-0.10270291E-02	-0.25424788E-02
610	-0.76199918E-03	-0.20472161E-02
611	-0.59469481E-03	-0.18627626E-02
612	-0.40683914E-03	-0.17251853E-02
613	-0.28889666E-03	-0.16680948E-02
614	-0.10311623E-03	-0.16194794E-02
615	0.00000000E+01	-0.16124230E-02

OUTPUT TABLE 2... STRESSES AT ELEMENT CENTROIDS

ELEMENT	X	Y	SIGMA(X)	SIGMA(Y)	TAU(X,Y)	SIGMA(1)	SIGMA(2)	ANGLE
1	4.00	4.00	-4.0300E-01	-3.9773E-01	-1.2011E-01	-2.8023E-01	-5.2050E-01	-4.5629E+01
2	12.00	4.00	-4.8615E-01	-7.1928E-01	-1.8609E-01	-3.8313E-01	-8.2230E-01	-2.8969E+01
3	20.00	4.00	-5.5301E-01	-9.4870E-01	-2.3234E-01	-4.4569E-01	-1.0560E+00	-2.4793E+01
4	27.00	4.00	-6.1528E-01	-1.1472E+00	-2.4311E-01	-5.2091E-01	-1.2416E+00	-2.1215E+01
5	33.00	4.00	-6.6730E-01	-1.3045E+00	-2.3137E-01	-5.9216E-01	-1.3797E+00	-1.7993E+01
6	39.00	4.00	-7.1798E-01	-1.4445E+00	-2.0362E-01	-6.6481E-01	-1.4977E+00	-1.4636E+01
7	44.00	4.00	-7.5836E-01	-1.5438E+00	-1.7234E-01	-7.2221E-01	-1.5800E+00	-1.1847E+01
8	48.00	4.00	-7.8825E-01	-1.6074E+00	-1.4427E-01	-7.6358E-01	-1.6321E+00	-9.7018E+00
9	52.00	4.00	-8.1519E-01	-1.6586E+00	-1.1568E-01	-7.9961E-01	-1.6742E+00	-7.6693E+00
10	56.00	4.00	-8.3824E-01	-1.6981E+00	-8.7757E-02	-8.2938E-01	-1.7070E+00	-5.7681E+00
11	59.00	4.00	-8.5437E-01	-1.7225E+00	-6.7698E-02	-8.4913E-01	-1.7277E+00	-4.4325E+00
12	61.00	4.00	-8.6224E-01	-1.7344E+00	-5.4780E-02	-8.5881E-01	-1.7379E+00	-3.5798E+00
13	63.00	4.00	-8.6859E-01	-1.7439E+00	-4.2178E-02	-8.6657E-01	-1.7459E+00	-2.7523E+00
14	65.00	4.00	-8.7341E-01	-1.7510E+00	-2.9834E-02	-8.7240E-01	-1.7520E+00	-1.9448E+00
15	66.50	4.00	-8.7701E-01	-1.7555E+00	-2.0713E-02	-8.7653E-01	-1.7559E+00	-1.3500E+00
16	67.50	4.00	-8.7824E-01	-1.7572E+00	-1.4682E-02	-8.7799E-01	-1.7575E+00	-9.5667E-01
17	68.25	4.00	-8.7920E-01	-1.7584E+00	-1.0178E-02	-8.7908E-01	-1.7585E+00	-6.6318E-01
18	68.75	4.00	-8.7950E-01	-1.7588E+00	-7.1817E-03	-8.7944E-01	-1.7589E+00	-4.6790E-01
19	69.15	4.00	-8.7974E-01	-1.7591E+00	-4.7864E-03	-8.7971E-01	-1.7592E+00	-3.1184E-01
20	69.53	4.00	-8.7978E-01	-1.7592E+00	-2.5421E-03	-8.7977E-01	-1.7592E+00	-1.6561E-01
21	69.85	4.00	-8.7989E-01	-1.7593E+00	-5.9856E-04	-8.7989E-01	-1.7593E+00	-3.8997E-02
22	4.00	12.00	-6.8017E-01	-4.5820E-01	-1.6548E-01	-3.6993E-01	-7.6844E-01	-6.1924E+01
23	12.00	12.00	-6.2777E-01	-6.5927E-01	-1.6771E-01	-4.7507E-01	-8.1197E-01	-4.2318E+01
24	20.00	12.00	-6.1765E-01	-9.2396E-01	-2.0314E-01	-5.1640E-01	-1.0252E+00	-2.6493E+01
25	27.00	12.00	-6.1521E-01	-1.1513E+00	-2.0851E-01	-5.4366E-01	-1.2229E+00	-1.8939E+01
26	33.00	12.00	-6.2180E-01	-1.3347E+00	-1.9174E-01	-5.7351E-01	-1.3830E+00	-1.4138E+01
27	39.00	12.00	-6.3960E-01	-1.4924E+00	-1.5660E-01	-6.1175E-01	-1.5202E+00	-1.0084E+01
28	44.00	12.00	-6.6521E-01	-1.5997E+00	-1.1932E-01	-6.5021E-01	-1.6147E+00	-7.1630E+00
29	48.00	12.00	-6.9245E-01	-1.6633E+00	-8.9458E-02	-6.8427E-01	-1.6715E+00	-5.2208E+00
30	52.00	12.00	-7.2321E-01	-1.7107E+00	-6.2651E-02	-7.1925E-01	-1.7146E+00	-3.6160E+00
31	56.00	12.00	-7.5397E-01	-1.7441E+00	-4.0979E-02	-7.5227E-01	-1.7457E+00	-2.3660E+00
32	59.00	12.00	-7.7765E-01	-1.7640E+00	-2.8518E-02	-7.7683E-01	-1.7648E+00	-1.6548E+00
33	61.00	12.00	-7.8962E-01	-1.7731E+00	-2.1669E-02	-7.8914E-01	-1.7736E+00	-1.2616E+00
34	63.00	12.00	-7.9959E-01	-1.7803E+00	-1.5829E-02	-7.9934E-01	-1.7805E+00	-9.2447E-01
35	65.00	12.00	-8.0726E-01	-1.7855E+00	-1.0785E-02	-8.0714E-01	-1.7857E+00	-6.3156E-01
36	66.50	12.00	-8.1332E-01	-1.7894E+00	-7.3716E-03	-8.1326E-01	-1.7894E+00	-4.3269E-01
37	67.50	12.00	-8.1527E-01	-1.7907E+00	-5.1834E-03	-8.1524E-01	-1.7907E+00	-3.0447E-01
38	68.25	12.00	-8.1695E-01	-1.7917E+00	-3.5814E-03	-8.1694E-01	-1.7917E+00	-2.1051E-01
39	68.75	12.00	-8.1743E-01	-1.7920E+00	-2.5200E-03	-8.1742E-01	-1.7920E+00	-1.4815E-01
40	69.15	12.00	-8.1785E-01	-1.7923E+00	-1.6792E-03	-8.1784E-01	-1.7923E+00	-9.8732E-02
41	69.53	12.00	-8.1790E-01	-1.7923E+00	-8.9318E-04	-8.1789E-01	-1.7923E+00	-5.2518E-02
42	69.85	12.00	-8.1810E-01	-1.7925E+00	-2.0823E-04	-8.1810E-01	-1.7925E+00	-1.2245E-02
43	4.00	19.00	-7.6347E-01	-4.7405E-01	-2.0083E-01	-3.7123E-01	-8.6629E-01	-6.2888E+01
44	12.00	19.00	-7.1854E-01	-6.4419E-01	-2.1090E-01	-4.6761E-01	-8.9512E-01	-5.0008E+01
45	20.00	19.00	-6.5778E-01	-9.0472E-01	-2.3254E-01	-5.1797E-01	-1.0445E+00	-3.1017E+01
46	27.00	19.00	-6.1373E-01	-1.1639E+00	-2.2811E-01	-5.3145E-01	-1.2462E+00	-1.9834E+01
47	33.00	19.00	-5.8915E-01	-1.3760E+00	-1.9533E-01	-5.4333E-01	-1.4218E+00	-1.3202E+01

ELEMENT	X	Y	SIGMA(X)	SIGMA(Y)	TAU(X,Y)	SIGMA(1)	SIGMA(2)	ANGLE
538	66.50	58.75	-1.2728E+01	4.9650E+00	7.0390E+01	6.7063E+01	-7.4826E+01	4.8582E+01
539	67.50	58.75	1.0436E+01	-1.3283E+01	6.8006E+01	6.7610E+01	-7.0456E+01	4.0054E+01
540	68.25	58.75	2.7765E+01	-2.9699E+01	5.7093E+01	6.2948E+01	-6.4882E+01	3.1643E+01
541	68.75	58.75	3.8928E+01	-3.8991E+01	4.4427E+01	5.9058E+01	-5.9121E+01	2.4376E+01
542	69.15	58.75	4.6020E+01	-4.5326E+01	3.1820E+01	5.6012E+01	-5.5317E+01	1.7433E+01
543	69.53	58.75	5.0431E+01	-4.8837E+01	1.8259E+01	5.3683E+01	-5.2089E+01	1.0099E+01
544	69.88	58.75	5.2578E+01	-5.0865E+01	4.8212E+00	5.2802E+01	-5.1089E+01	2.5627E+00
545	4.00	60.25	5.0753E+01	1.4270E+01	-4.6237E+00	5.1330E+01	1.3693E+01	-7.1116E+00
546	12.00	60.25	3.6270E+01	-1.8187E+00	4.7763E+01	3.6276E+01	-1.8247E+00	-7.1834E-01
547	20.00	60.25	2.6229E+01	3.2224E-02	-2.0418E+00	2.6387E+01	-1.2596E-01	-4.4301E+00
548	27.00	60.25	1.5221E+01	-1.5222E-01	1.9310E+00	1.5460E+01	-3.9106E-01	-7.0510E+00
549	33.00	60.25	3.4321E+00	2.0267E-01	-2.2143E+00	4.5579E+00	-9.2315E-01	-2.6950E+01
550	39.00	60.25	-1.0424E+01	-7.1816E-02	-2.4162E+00	4.6435E-01	-1.0960E+01	-7.7489E+01
551	44.00	60.25	-2.7063E+01	-8.9174E-01	-2.4098E+00	-6.7170E-01	-2.7283E+01	-8.4783E+01
552	48.00	60.25	-4.0146E+01	1.6554E-01	-3.2820E+00	4.3100E-01	-4.0412E+01	-8.5376E+01
553	52.00	60.25	-5.5168E+01	3.4740E-02	-3.5934E+00	2.6766E-01	-5.5401E+01	-8.6292E+01
554	56.00	60.25	-7.0142E+01	7.4205E-02	-2.6765E+00	1.7608E-01	-7.0244E+01	-8.7821E+01
555	59.00	60.25	-8.2451E+01	-8.2955E-01	9.9843E-01	-8.1734E-01	-8.2463E+01	8.9300E+01
556	61.00	60.25	-7.7362E+01	1.5862E+00	4.7301E+00	1.8685E+00	-7.7644E+01	8.6584E+01
557	63.00	60.25	-5.7162E+01	2.2114E+00	1.3690E+01	5.2162E+00	-6.0166E+01	7.7622E+01
558	65.00	60.25	-1.2926E+01	4.0496E+00	2.5963E+01	2.2877E+01	-3.1753E+01	5.4052E+01
559	66.50	60.25	3.5522E+01	2.9363E+00	3.0149E+01	5.3499E+01	-1.5040E+01	3.0806E+01
560	67.50	60.25	7.0942E+01	-2.2983E+00	2.9547E+01	8.1376E+01	-1.2732E+01	1.9449E+01
561	68.25	60.25	9.4457E+01	-5.8944E+00	2.4019E+01	9.9909E+01	-1.1347E+01	1.2790E+01
562	68.75	60.25	1.0719E+02	-8.0856E+00	1.8544E+01	1.1010E+02	-1.0995E+01	8.9175E+00
563	69.15	60.25	1.1428E+02	-9.7672E+00	1.3108E+01	1.1565E+02	-1.1137E+01	5.9663E+00
564	69.53	60.25	1.1937E+02	-1.0105E+01	7.5472E+00	1.1981E+02	-1.0543E+01	3.3249E+00
565	69.88	60.25	1.2090E+02	-1.0897E+01	2.0391E+00	1.2094E+02	-1.0929E+01	8.8613E-01

... STRAIN ENERGY W/O CKEL = 9.28372

NCKEL = 1

CKEL = 1

... STRESS INTENSITY FACTOR ... 1

OPENING MODE K1 = -1066.60

SHEARING MODE K2 = 0.00000

CRACK TIP XC = 70.0000 YC = 53.5000 AT MATERIAL 1

THE SNODES = 417 418 461 460 439

STRAIN ENERGY = 0.425538

... TOTAL STRAIN ENERGY = 9.7092569 ...

Crack Propagation Calculation Program

Figure B-2 shows the program flow diagram. The following is a brief description of the main program and each subroutine.

CRACK:

Main program. Sets values of necessary parameters and reads in necessary data. Iterates from problem to problem, reading in data specific to the problem and calling NVSC. Writes out modulus values, A, and n for each modulus condition considered.

NVSC: Calculates and applies crack increments. Computes N_f values for each increment. Computes N_f for each crack increment. Prints the

base and overlay thicknesses and the modulus condition of the pavement being analyzed.

TRAPRLE:

Given the left and right limits of the crack increment, numerically integrates the Paris equation using the trapezoidal rule, producing N_f . Iterates until the percent difference is below 10^{-3} or to a limit of 15 iterations.

SIMPRLE:

Performs exactly as does TRAPRLE except that Simpson's rule is used for the numerical integration.



Figure B-2. Flow Diagram of Program for Solution of Paris Equation

PARIS:

Determines the value of K_I for a given value of (c/b) according to equation (6), Section I. Computes the integrand of equation (4), Section I, including the base thickness term. Paris is a function which returns as its value the above integrand.

Input Guide- The input parameters and formats are explained here.

BASIC PARAMETERS:

One Card: Format 2I5

- cc 1-5 Nmodcon: The number of modulus conditions considered for the problem.
- cc 6-10 K: A value of $K = 0$ indicates that the trapezoidal rule will be used. Any other 5 digit integer, normally 1, indicates that Simpson's rule will be used.

Nmodcon cards: Format 4F10.0

- cc 1-10 Evalues(I,1): Modulus of elasticity of the base corresponding to modulus condition 1.
- cc 11-20 Evalues(I,2): Modulus of elasticity of the overlay corresponding to modulus condition 1
- cc 21-30 AA(I): Values of the parameter A in the Paris equation corresponding to modulus condition 1.
- cc 31-40 NN(I): Value of the parameter n in the Paris equation corresponding to modulus condition 1.

PROBLEM PARAMETERS:

One card per problem: Format 3I5

cc 1-5 Nprob: The problem number of the current problem. If the value is 0, execution of the program is halted.

cc 6-10 Base: Thickness of the base.

cc 11-15 Olay: Thickness of the overlay.

Nmodcon cards per problem: Format 6F10.0

cc 1-10 B_0 : Coefficient B_0 in equation (6), Section I.

cc 11-20 B_1 : Coefficient B_1 in equation (6), Section I.

cc 21-30 B_2 : Coefficient B_2 in equation (6), Section I.

cc 31-40 B_3 : Coefficient B_3 in equation (6), Section I.

cc 41-50 Left: Initial crack length for a pavement with given base and overlay thicknesses and modulus condition.

cc 51-60 Right: Final crack length for a pavement with given base and overlay thicknesses and modulus condition.

The computer code listing for the paris equation calculation scheme is given on the following pages.

```

1  PROGRAM CRACK(INPUT,OUTPUT,TAPES=INPUT,TAPES=OUTPUT)
2  IMPLICIT INTEGER (A-Z)
3  REAL LEFT,RIGHT,TEST,AA(20),NN(20),EVALUES(20,2),OBASE
4  REAL B0,B1,B2,B3
5  COMMON/INTCON/ BASE,OLAY,B0,B1,B2,B3,AA,NN
6  COMMON/PARAM/ IN,OUT,TEST,Z,EVALUES
7  DATA TEST/0.001/,Z/15/,IN/5/,OUT/6/,OBASE/3500000.0/
8  C
9  READ (IN,100) NMODCON,K
10 C
11 WRITE(OUT,500)
12 C
13 DO 10 I=1,NMODCON
14 READ (IN,200) EVALUES(I,1),EVALUES(I,2),AA(I),NN(I)
15 WRITE(OUT,600) I,EVALUES(I,1),EVALUES(I,2),OBASE,AA(I),NN(I)
16 10 CONTINUE
17 C
18 READ (IN,300) NPROB,BASE,OLAY
19 C
20 IF(NPROB.EQ.0) GO TO 40
21 C
22 DO 30 I=1,NMODCON
23 READ (IN,400) B0,B1,B2,B3,LEFT,RIGHT
24 CALL NVSC(LEFT,RIGHT,I,K)
25 30 CONTINUE
26 C
27 IF(NPROB.NE.0) GO TO 20
28 C
29 FORMAT(2I5)
30 FORMAT(4F10.0)
31 300 FORMAT(3I5)
32 400 FORMAT(6F10.0)
33 500 FORMAT(//////35X,'MODULUS',7X,'BASE',8X,'SUBORDE',6X,'OVERLAY'//
34 +34X,'CONDITION',5X,'MODULUS',2(6X,'MODULUS'),9X,'A',12X,'N'/30X,
35 +6(4X,'-----'))
36 600 FORMAT(34X,15,8X,F9.1,6X,F6.1,6X,F8.1,4X,E9.4,6X,F5.2)
37 C
38 40 STOP
39 END
40 C
41 C
42 C
43 SUBROUTINE TRAPRLE(CL,CR,MODCON,DNF,FLAG,DIFF)
44 IMPLICIT REAL (A-Z)
45 INTEGER YOMAMA,NDIV,I,J,MODCON,IN,OUT,Z
46 LOGICAL FLAG
47 COMMON/PARAM/ IN,OUT,TEST,Z,EVALUES
48 C
49 C
50 FLAG=.FALSE.
51 ODDSUM=0.0
52 YOMAMA=0
53 OLOVAL=0.0
54 NDIV=4
55 C
56 C
57 H=(CR-CL)/NDIV
58 INTSUM=PARIS(CL+2*H,MODCON)
59 ENDSUM=PARIS(CL,MODCON)+PARIS(CR,MODCON)
60 C

```



```

181 COMMON/PARAM/ IN,OUT,TEST,Z
182 C
183 C
184 FLAG=.FALSE.
185 YODADY=0
186 NDIV=4
187 C
188 H=(CR-CL)/NDIV
189 EVENS=PARIS(CL+2*H,MODCON)
190 ODDS=PARIS(CL+H,MODCON)+PARIS(CL+3*H,MODCON)
191 ENDSUM=PARIS(CL,MODCON)+PARIS(CR,MODCON)
192 OLDVAL=H/3.0*(ENDSUM+4*ODDS+2*EVENS)
193 C
194 DO 10 I=2,Z
195 NDIV=2*NDIV
196 H=(CR-CL)/NDIV
197 EVENS=EVENS+ODDS
198 ODDS=O.O
199 LIM=NDIV-1
200 DO 20 J=1,LIM,2
201 ODDS=ODDS+PARIS(CL+J*H,MODCON)
202 CONTINUE
203 NEWVAL=H/3.0*(ENDSUM+4*ODDS+2*EVENS)
204 DIFF=ABS(NEWVAL-OLDVAL)/NEWVAL
205 IF(DIFF.LT.TEST) THEN
206 YODADY=1
207 GO TO 30
208 ELSE
209 OLDVAL=NEWVAL
210 END IF
211 10 CONTINUE
212 C
213 30 IF(YODADY.EQ.O) THEN
214 FLAG=.TRUE.
215 DNF=NEWVAL
216 ELSE
217 DNF=NEWVAL
218 END IF
219 C
220 RETURN
221 END

```

Sample Input for Crack Propagation Program- The following pages contain a printout of the input data for the sample program.

Sample Output for Crack Propagation Program- The following pages contain the output for the sample problem.

MODULUS CONDITION	BASE MODULUS	SUBGRDE MODULUS	OVERLAY MODULUS	A	N
1	1200000.0	4000.0	350000.0	.1000E-09	2.00
2	1200000.0	9000.0	350000.0	.1000E-09	2.00
3	800000.0	4000.0	350000.0	.1000E-09	2.00
4	800000.0	9000.0	350000.0	.1000E-09	2.00
5	400000.0	4000.0	350000.0	.1000E-09	2.00
6	400000.0	9000.0	350000.0	.1000E-09	2.00

DEPTH OF BASE..... 16

DEPTH OF OVERLAY..... 0

MODULUS CONDITION.... 1

INITIAL C/B RATIO	CURRENT C/B RATIO	INCREMENTAL LOAD CYCLES	CURRENT LOAD CYCLES
0.0000	.0500	.88171E+04	.88171E+04
.0500	.1000	.55496E+04	.14367E+05
.1000	.1500	.43884E+04	.18755E+05
.1500	.2000	.39863E+04	.22741E+05
.2000	.2500	.40022E+04	.26744E+05
.2500	.3000	.43572E+04	.31101E+05
.3000	.3500	.50935E+04	.36194E+05
.3500	.4000	.63598E+04	.42554E+05
.4000	.4500	.84512E+04	.51005E+05
.4500	.5000	.11886E+05	.62891E+05
.5000	.5500	.17455E+05	.80346E+05
.5500	.6000	.25903E+05	.10625E+06
.6000	.6500	.36316E+05	.14257E+06
.6500	.7000	.43316E+05	.18588E+06
.7000	.7500	.39819E+05	.22570E+06
.7500	.8000	.27747E+05	.25345E+06
.8000	.8500	.15880E+05	.26933E+06
.8500	.9000	.83164E+04	.27764E+06
.9000	.9500	.43036E+04	.28195E+06
.9500	1.0000	.22864E+04	.28423E+06

etc.
↓

END
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